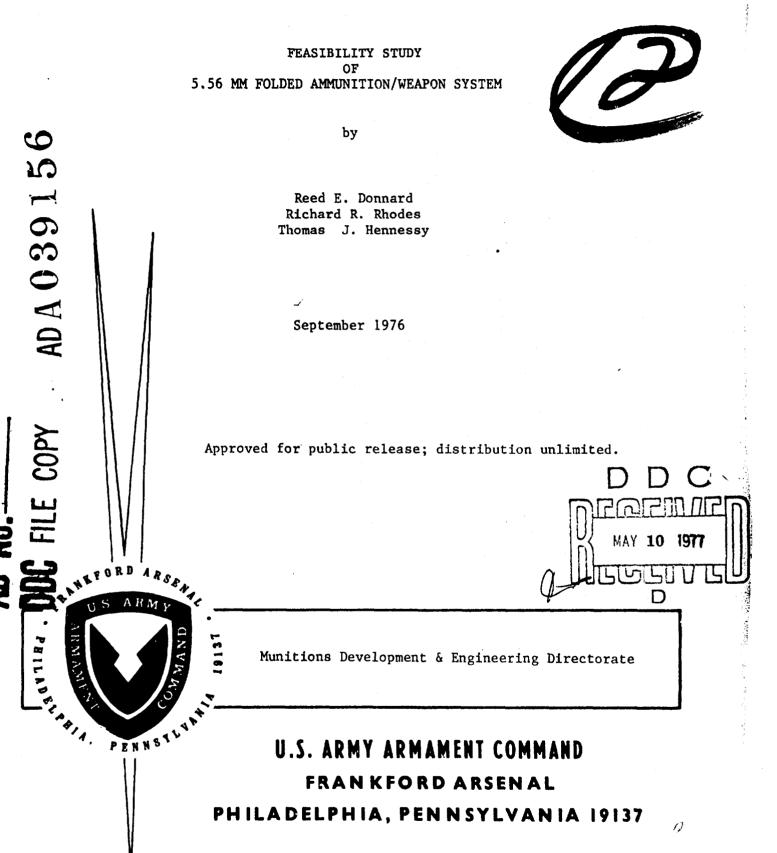


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18. SUPPLEMENTARY NOTES - Cont'd

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| Fabrication Processing | John F. Kloskey Samuel J. Marziano Albert Zalcmann |
| Firing Fixture Design | John A. Duffy Robert W. Markgraf |

20. ABSTRACT - Cont'd

of the cartridge using the Folded Ammunition approach makes possible now what had previously been unattainable in the way of weapon/ ammunition system optimization. This report describes the concept, outlines its advantages and presents the results of a short-term analytical and experimental program that successfully demonstrated the feasibility of Folded Ammunition.

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INTRODUCTION

Folded Ammunition is a unique concept in ammunition design that relocates the propellant charge from the convention position behind and coaxial with the projectile to one beside the projectile (Figure 1). US Patent 3857339, shown in Appendix A, documents this concept. For a given energy output, conventional axially symmetric ammunition cartridges do not provide the most efficient geometrical shape for a minimum system parametric profile (system length, weight and bulk). Reconfiguration of the cartridge using the Folded Ammunition approach makes possible now what had previously been unattainable in the way of weapon/ammunition system optimization.

Figure 2 depicts the Weapon/Ammunition systems relationships in diagramed cross-sectional view. The basis for weight saving in the Folded Cartridge can be seen as the result of complete support afforded by the weapon which permits use of a uniformly thin-walled cartridge case. In addition to the significant savings in cartridge weight, the shortened cartridge length results in another important system improvement, i.e., shortened minimum bolt stroke of the weapon. At least two weapon benefits arise from this attribute. These are: (1) a greater range in cyclic rate capability including increased cyclic rate if desired, and (2) reduced bolt velocity for a given cyclic rate. This latter feature would be a direct contributing factor to increased weapons parts life.

Perhaps the most far-reaching attribute of the Folded Ammunition concept is the significant reduction in "packing volume" occupied by the cartridge in comparison to conventional axisymmetrically shaped ammunition. Figure 3 reveals the geometrical basis for this statement of fact. The packing volume occupied by the cartridge is defined as the minimum dimension rectangular solid encasing the cartridge. The shaded area around the cartridge represents dead space which detracts from packing efficiency. In this illustration, a conventional 5.56 mm FABRL cartridge is shown in side and end views with appropriate dimensions. The packing volume of this cartridge is:

$$V_{\rm p} = 1 \times d^2 = 0.323 \, {\rm in}^3$$

where:

Į

 V_p = packing volume in inches³

1 = cartridge length in inches

d = cartridge case base diameter in inches

In contrast, the three views of a 5.56 mm Folded Cartridge represent the cartridge size in the folded configuration required to house the same propellant charge and fire the same projectile at the same velocity as

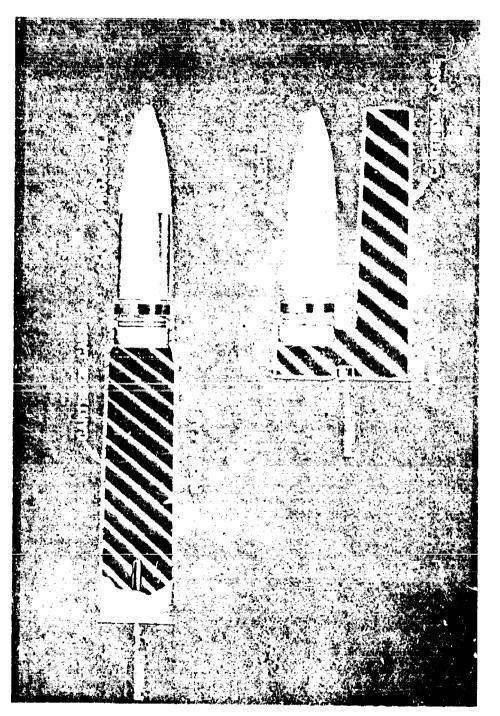
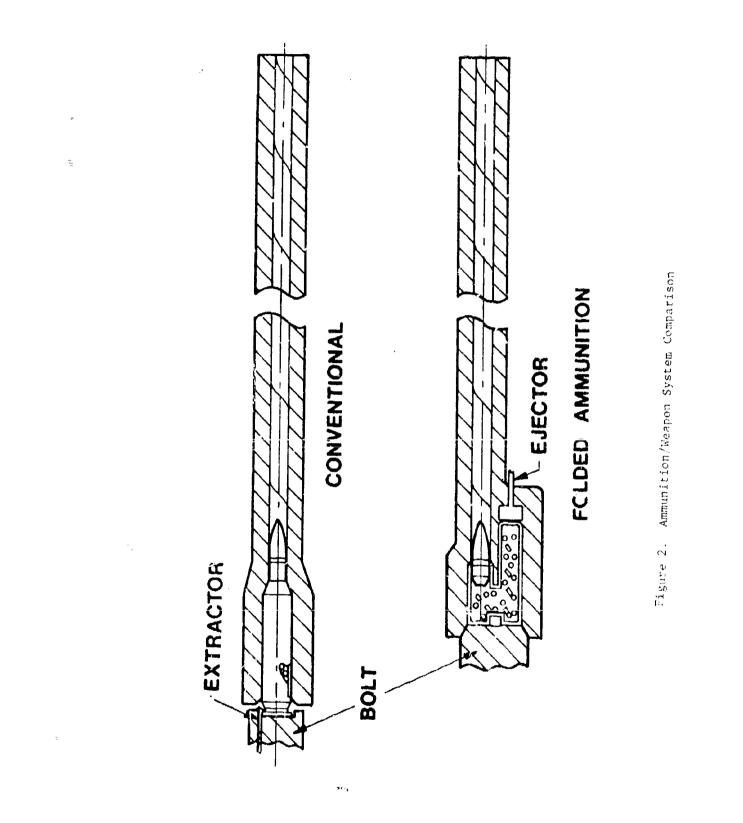
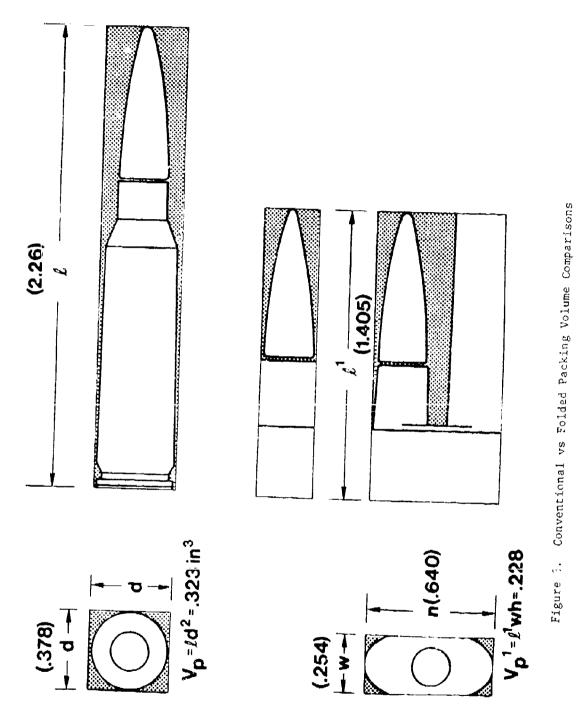


Figure 1. Ammunition Concept



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the conventional. Using the same packing volume analogy as that for the conventional cartridge, it can be seen that the packing volume (rectangular volume or envelope occupied by the folded cartridge) is significantly smaller than that of the conventional cartridge. This is verified by calculating the new cartridge volume. Thus:

$$V'_n = 1' w h = 0.228 in^3$$

where:

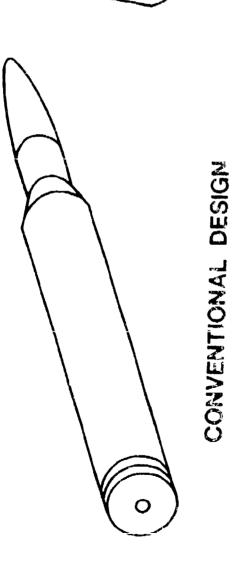
V'p = packing volume in inches³
l' = cartridge length in inches
w = cartridge width in inches
h = cartridge height in inches

This represents about a 29 percent volume decrease based on the packing volume of the conventional 5.56 mm FABRL cartridge. This is the dramatic space saving result of Folded Ammunition.

Using examples in all three caliber regimes (small, automatic cannon and large), the comparisons shown in Figure 4 have been projected to show the universality of cartridge geometric and weight savings that can accrue using the folded design. It is interesting to note that even though the 30 mm GAU-8 cartridge has an aluminum case, a weight reduction is still indicated in the folded version utilizing a thin wall steel case. An excellent illustration of the operational potential of this new idea is to compare the effect it would have on the infantry soldier using the MI6Al rifle and 5.56 mm ammunition as a reference. In the infantry squad, the rifleman can carry 450 rounds comprising a weight of about 16.4 pounds. With Folded Ammunition, this weight is reduced to 13.5 pounds. Or, keeping the ammunition load at 16.4 pounds, he would carry approximately 574 rounds, an increase of 124 cartridges (27 percent of the basic load). In addition to this, it is estimated that weapon length could be decreased (due to reduced length of bolt stroke and chamber) with a resultant weight saving of about 1/4 pound.

In connection with the shortened bolt stroke of the weapon, at least two advantages are pertinent. Of course, rate of fire can be extended more easily into the higher ranges if that is desired. But, for a given rate of fire, the shortened bolt stroke should enable weapon designers to reduce acceleration of the bolt and associated parts with a resultant benefit to parts life.

From the systems viewpoint, and aside from the logistic aspects, improvement in space utilization is of particular importance equally in vehicle and tank turret application. This is true not only for ammunition oriented space utilization, but also for weapon intrusion into otherwide cramped quarters. Shortened ram stroke requirements



0

FOLDED DESIGN

| VOLUME LENGTH WEIGHT REDUCTION REDUCTION REDUCTION | M153 38 44 25 | FABRL 30 38 22 | M80 27 44 18 | SAU-8 24 41 12 | 1CAAAC 42 41 22 |
|---|---------------|----------------|--------------|----------------|-----------------|
| | 5.56 MM M193 | 5.56 MM FABRL | 7.62 MM M80 | 30 MM GAU-8 | 60 MM MCAAAC |

Figure 4. Cartridge Comparisons

play a significant role here. Additionally, the advantages in system applications to aircraft where volume and/or weight savings can be critical should not be overlooked.

DISCUSSION

Based on the significant potential advantages offered by the concept of Folded Ammunition, a hardware-oreinted study was undertaken to investigate the basic feasibility of this new idea and determine the practicality of adapting it to a working system design. The feasibility investigation consisted of a series of interrelated activities combining analytical and hardware phases which served as the basis for the feasibility assessment. These included design analysis, ballistic studies, case fabrication processes and automatic firing fixture elevation.

Design Analysis

The experimental folded test weapon chamber and ammunition cartridge case were designed using the finite element stress analysis technique which took into account chamber and case deflections at peak pressure for satisfactory case performance. The analysis was made using a cartridge configuration designed around the 5.56 mm FABRL projectile. As illustrated in Figure 5, there is a significant difference in the weapon support given to the cartridge case in the folded configuration as opposed In order to provide for extractor engagement with conto conventional. ventional design, the case head can be only partially supported. Thus, it must be relatively massive to withstand the pressure developed by the burning propellant. On the other hand, Folded Ammunition can be fully supported since the cartridge case is not extracted from the rear but is pushed out from the front of the propellant capsule. This permits the design of a case which is thin wall in construction in the head as well as the body. Since approximately 40 percent of conventional case weight is in the head region, a considerable weight saving is achieved.

The analysis required to design a Folded Ammunition case is less than that required for the conventional round because the folded case is completely inclosed by the chamber. The strength requirements in the case are determined solely by the need for case recovery after firing.

The first part of the analysis is calculation of the radial expansion of the inside chamber wall as a function of chamber pressure. Conventional weapon chambers are bodies of revolution with simple generating curves. Usually, the chamber may be approximated as an assembly of uniform cylinders. The deflection of a uniform cylinder under internal pressure is:

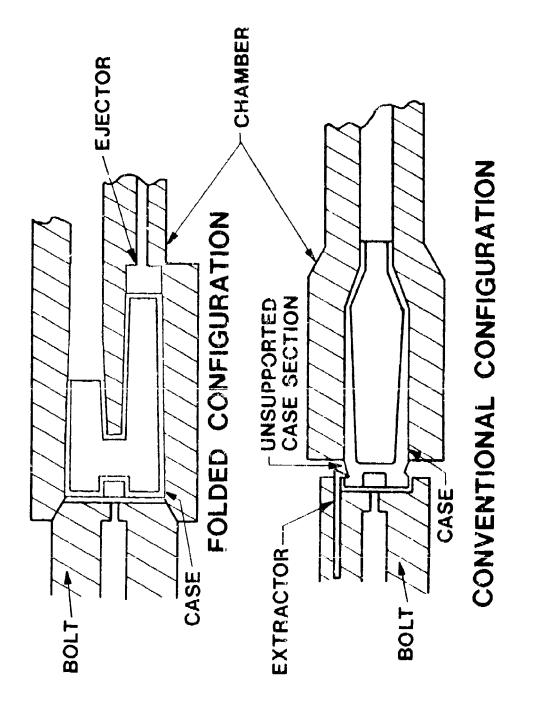


Figure 5. Design Analysis

$$\sigma = \frac{r_{i} P}{E} \left[\frac{1 + (r_{i}/r_{o})^{2}}{1 - (r_{i}/r_{o})^{2}} + \gamma \right]$$

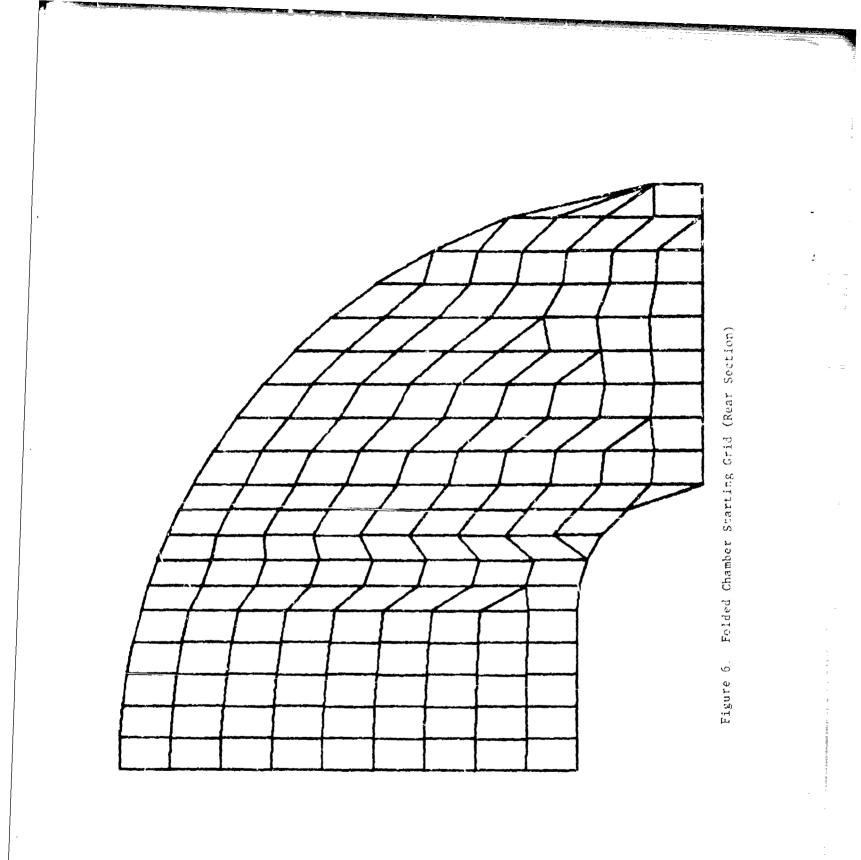
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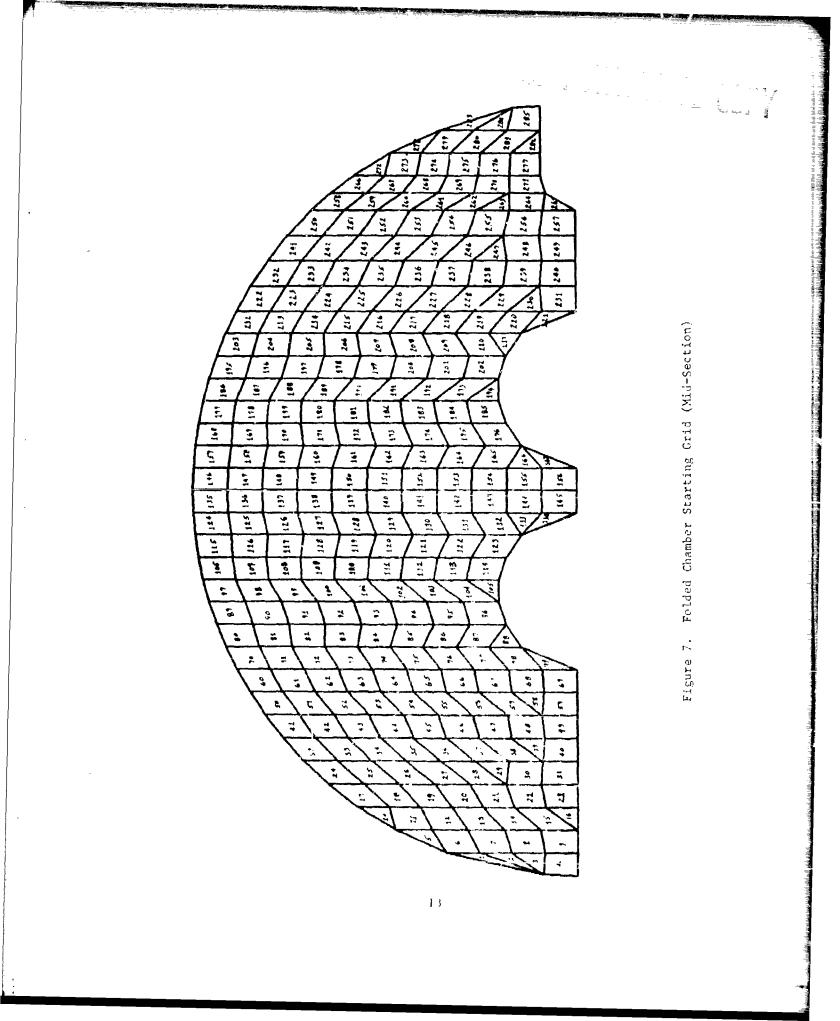
- E = Young's Modulus of Elasticity
- P = Chamber pressure in psi
- r₁ = Inside chamber radius in inches
- $r_0 =$ Outside chamber radius in inches
- σ = Deflection of inside chamber radius in inches
- γ = Poisson's ratio

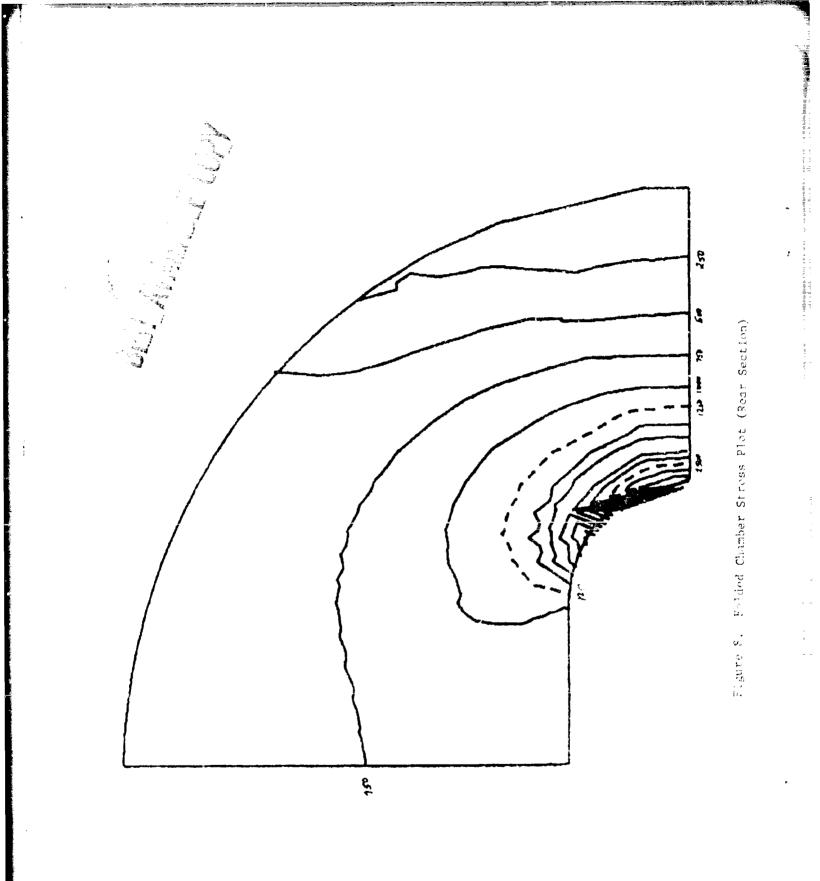
The Folded Ammunition case, however, is not a simple body of revolution. In fact, a complete stress analysis of the chamber or case would require solution of the three-dimensional Balltrami-Mitchell equations. However, an approximation of chamber deflection can be obtained by analyzing crosssectional slices of the chamber as two-dimensional problems. Using this approach, considered sufficient for this analysis, two sections, one at the rear and the other at the mid-section of the chamber, were chosen. A finite element grid was prepared for each section. The starting grids are shown in Figures 6 and 7, and the results of the analyses are shown in Figures 8 and 9. These plots show lines of constant stress at intervals of 250 psi throughout each cross section. Figures 10 and 11 show a plot of the deformed geometry (solid lines) superimposed on a plot of the original grid (dashed lines). Both sections are assumed to be sections of long regular bodies, i.e., in plane strain. This means that the axial variation of stress in the weapon chamber is completely neglected and axial stress is assumed to be zero. This assumption gives radial chamber deflections that we estimate to be 10 to 20 percent greater than actually occur in the weapon, thus making the analysis conservative.

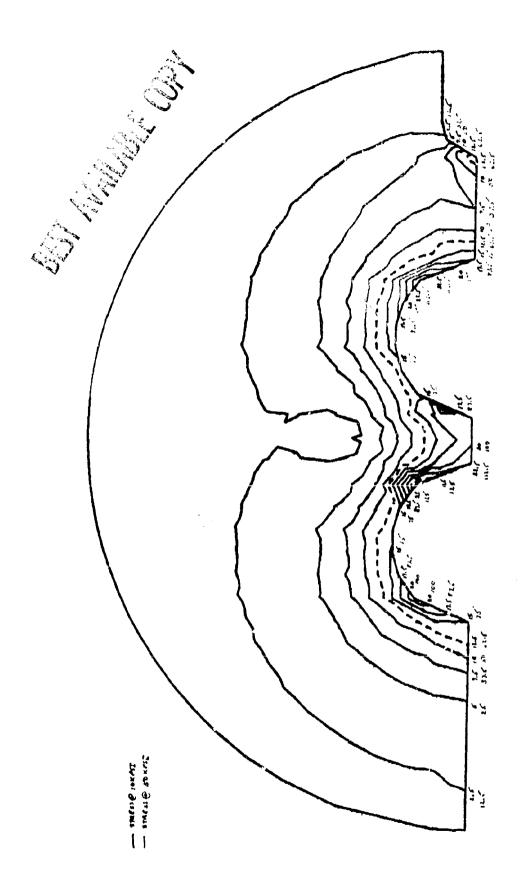
Table I presents the x and y components of the deformation vector of the nodal points on the interior radius of the chamber. The numbers may be combined vectorially to yield the radial deflection at each point. These, in turn, are averaged to determine the average radial deflection in the cross section.

Once the average radial deflections are known, case yield strength is determined so that the case will recover enough of its original shape after pressurization to insure extraction. Specifically, the requirement that case radial recovery be greater than chamber deflection implies that static extraction force is zero at zero chamber pressure.









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Figure 9. Folded Chamber Stress Plot (Mid-Section)

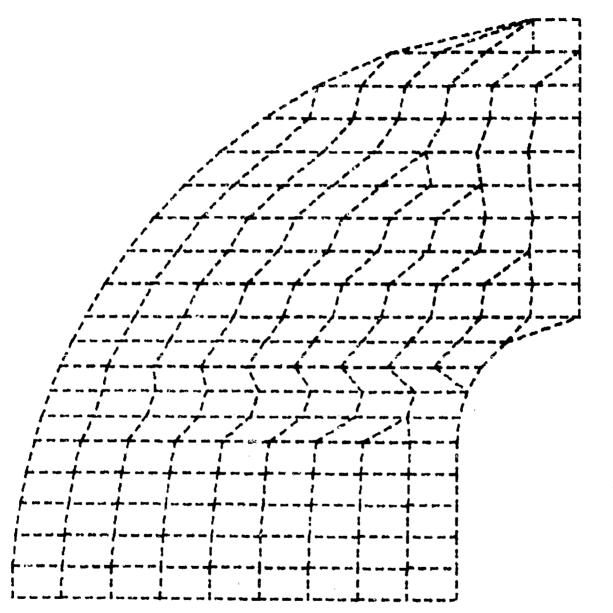
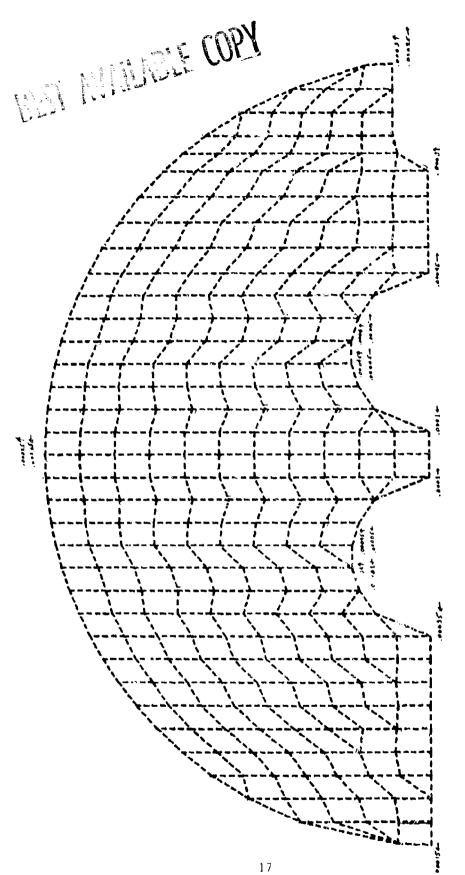


Figure 10. Folded Chamber Distortion Plot (Rear Section)

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Folded Chamber Distortion Plot (Mid-Section) Figure 11.

| X(x 10 ⁻⁶ in.) | Y(x 10-6 in.) | X(x 10 ⁻⁶ in.) | Y(x 10 ⁻⁶ in.) |
|---------------------------|-----------------|---------------------------|---------------------------|
| 26 | 0. | 4 | r |
| 26 | 0. | 6 | -5 -4 |
| 25 | υ. | 7 | -4 ~2 |
| 23 | 9. | 7 | - |
| 21 | 0. | 7 | -1 |
| 19 | 0. | 7 | -1 |
| 16 | 0. | 7 | 0 |
| 3 | 0. | / | 1 |
| 9 | ů. | / | 1 |
| 0 | 0. | 5 | 2 |
| 0 | 0. | 0 | 2 |
| Verage radial | do Classifica - | | |

| | | LE 1. | | | |
|---------|--------------|---------|---------|----|------|
| Chamber | Deflection a | t Nodes | Located | on | 1.D. |

6 inchar

| 19.7 x 10 ⁻⁶ | looo psi | | 6.3×10^{-6} | inches 1000 psi |
|-------------------------|----------|--|----------------------|--------------------|
|-------------------------|----------|--|----------------------|--------------------|

Application of thin shell theory to the cylindrical portion of the Folded Ammunition case permits this criteria to be expressed as:

$$\begin{bmatrix} \frac{2}{\sqrt{3}} - \frac{\gamma}{\sqrt{3}} \end{bmatrix} \xrightarrow{\sigma_{\text{Yield}}} \frac{s}{E} \xrightarrow{\sigma_{\text{Y}}} \frac{\sigma_{\text{Yield}}}{r}$$

where:

1

 \approx Yield strength of cartridge case material ^oYield

This approach in conjunction with the results from Table I gives a minimum yield strength of 102,000 psi for a chamber pressure of 65,000 psi. Similar calculations carried out for the rear section of the case indicate a minimum required yield strength of 164,000 psi.

The results of this analysis can be compared with the known solution for a thick-walled cylinder. The purpose of the comparison is to see how much effect increasing the barrel outside diameter has on reducing the minimum yield strength values required in the case. As the outside diameter of the thick cylinder increases, the internal deflection decreases and, in the limit, it approaches the value given by:

$$\sigma = \frac{\lim_{r_0 \to \infty} \left\{ \frac{r_i p}{E} \left[\frac{1 + (r_i/r_0)}{1 - (r_i/r_0)} + \gamma \right] \right\}$$
$$\sigma = \frac{r_i p}{E} \left[1 + \gamma \right]$$

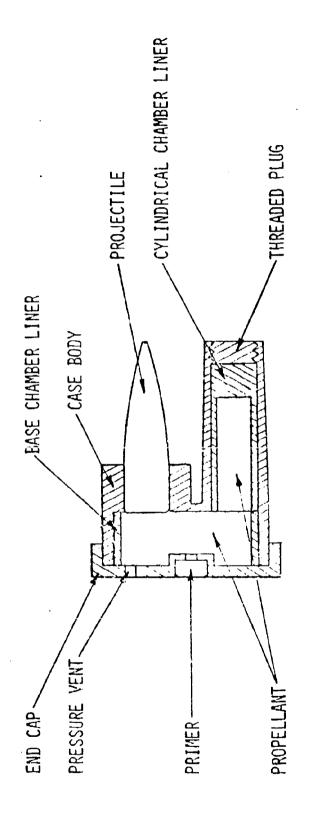
Comparison of both Folded Ammunition chamber sections with the cylinder equation shows that little can be gained by large increases in chamber outside diameter. The comparison is made by converting the Folded Ammunition to an area-equivalent cylinder and then calculating the limiting value of internal deflection of that thick cylinder. Results are summarized in Table II.

TABLE II. Comparison of Internal Diameter Deflections

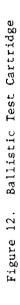
| | Finite Element Value | Limiting Value |
|-------------|---------------------------|-----------------------------|
| Section A-A | 128×10^{-5} in. | 104×10^{-5} in. |
| Section B-B | 40.9×10^{-5} in. | 34.4 x lŪ ⁼⁵ in. |

Ballistic Studies

Preliminary ballistic studies of the 5.56 mm Folded Ammunition were conducted in a heavy walled test weapon instrumented for measurement of chamber pressure. The weapon was designed to accept reuseable cartridge cases which were assembled from several components allowing variation of internal volume and propellant charge distribution. The components of a representative case are shown in Figure 12. Propellant granulation and ignition studies were performed with this test hardware to establish the desired performance level. In addition, a parametric study was conducted to evaluate the effect on interior ballistics of the 5.56 mm Folded Ammunition of varying the relative locations of the primer and propellant charge. Also studied was the effect of throttling gas flow from propellant bed to projectile using flow areas of 1/3, 2/3 and full bore area. A special breech for the test weapon and special case bases were fabricated to permit firing with the primer located on the axis of the cylindrical portion of the mase. Also fabricated were liners for the rear portion of the case which confine the propellant charge to the cylindrical portion and permit gas bleed-off through areas of 1/3 and 2/3 bore area. A representative liner is shown in Figure 13. The crosssectional area of the gas bleed-off control slot is shown elevated for clarity.



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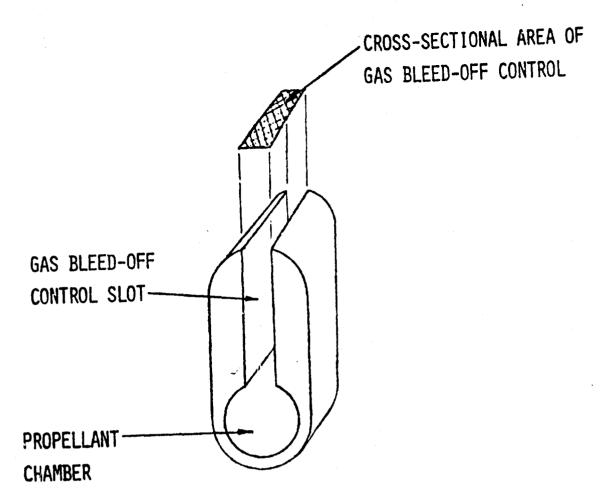


Figure 13. Gas Flow Control Liner

The modified cases were fired in an instrumented test weapon with the following results. The standard case configuration and loading with primer positioned on the axis of the cylindrical portion produced no detectable effect on either pressure or velocity. When the 2/3 bore area liner was inserted and the entire propellant charge confined to the cylinder, average peak pressure increased by approximately 8 percent and muzzle velocity increased by approximately 2 percent. Using the same configuration as above but loading propellant in both the cylinder and throat produced a 20 percent increase in average peak pressure. Average velocity was the same as before but uniformity was substantially degraded. Using the 1/3 bore area liner and the propellant charge again confined to cylindrical area of the case, average peak pressure rose 10 percent while velocity decreased 10 percent. Here again, ballistic uniformity was poor.

It was concluded from these firings that locating the entire propellant charge in the cylindrical portion of the case in combination with an axially positioned primer can provide a small increase in performance. However, the magnitude of this increase is such that other considerations such as case manufacturing procedures, may render its adoption impractical or cost-ineffective. It was also observed that restriction of gas flow from the propellant charge to the projectile tends to decrease performance and greatly degrades ballistic uniformity. However, this study was limited to the 5.56 mm caliber, and, specifically, the FABRL bullet, which has a comparatively low sectional density. More extensive studies will be required to determine whether specific ballistic advantages can be realized in other systems through variation of primer location, propellant position or gas bleed-off control.

Based on the preliminary ballistic studies in the test weapon and results of the design analysis, a prototype cartridge was designed and fabrication initiated. Concurrent with this, a test barrel was designed and fabricated to accept this ammunition and permit firing from a universal test fixture. This barrel, in addition to being instrumented for measurement of chamber pressure, also permitted port pressure measurement to obtain data required for automatic firing fixture operating mechanism design. In order to obtain a pressure gradient, port pressure taps were provided at the normal M16Al position (6 1/2" from muzzle) and at a position three inches closer to the bolt face.

Test firings were conducted with pressure gages at the chamber and both port positions. Results of these tests indicated a pressure at the normal port position of 9000 psi for rounds fired at the desired performance level. This translates into a bolt cavity pressure of approximately 750 psi. Since the MI6Al normally operates at a bolt cavity pressure of 1600 to 2400 psi, a relocation of the pressure port was indicated. Figure 14 is a graph of bolt cavity pressure versus port position calculated from the pressures measured in these test firings. From this curve, a new port position was selected for the automatic Folded Ammunition firing fixture. This position is four inches to the rear of the original in order to produce an estimated average bolt cavity pressure of 1800 psi.

Interior ballistic studies in both the above test weapon and eventually in the automatic test fixtures resulted in selection of a propellant charge for the 5.56 mm folded FABRL cartridge that yields velocity and pressure levels comparable to those obtained with the conventional design. The data shown in Figure 15 compares interior ballistics of the conventional and folded designs using the same 5.56 mm FABRL bullet, 18 grains of propellant WC 680 and the standard FA 41 primer. These curves are derived from a series of test firings and are representative of these firings. Performance levels are statistically similar as are calculated efficiencies. The Folded Ammunition time-pressure curves showed no evidence of unusual spikes, pressure waves or ignition anomalies. Representative firing data are shown in Appendix B.

The conclusion can be drawn that the folded design does not appear to introduce any negative effects on ignition or propellant combustion. There has been some indication in limited studies of a 30 mm system that in the heavier, larger caliber projectives improved ballistic efficiencies can be achieved with the Folded Ammunition design. Studies are being continued in the 30 mm experimental test fixtures to ascertain the validity of this preliminary evidence.

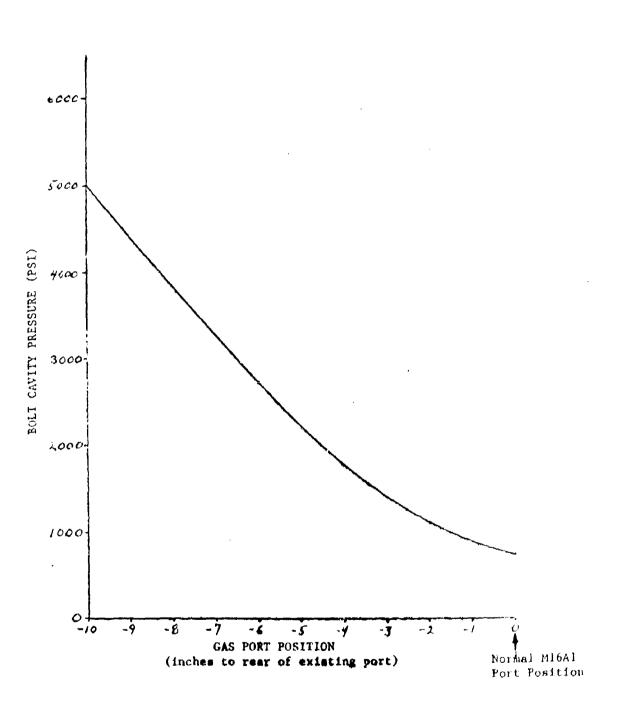


Figure 14. Bolt Cavity Pressure vs Gas Port Position

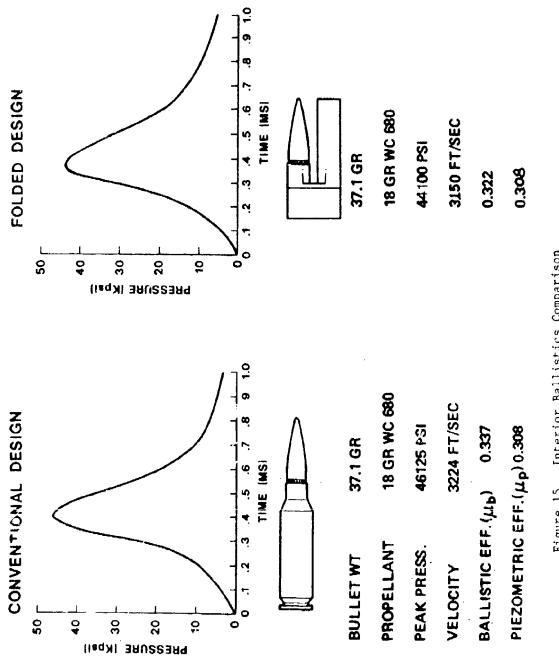


Figure 15. Interior Ballistics Comparison

Case Fabrication

Due to the unusual shape of a folded cartridge case, the ability to produce this case at a rate and cost comparable to those of a conventional case was a matter for serious concern in this feasibility study. Therefore, the establishment of a case fabrication process, that would indicate a high degree of case producibility, was considered an essential aspect in demonstrating the feasibility of Folded Ammunition.

Three decisions had to be made at the outset of this feasibility study with respect to the number of case components, the forming technique and the case material. A two-piece case was selected in lieu of either a one-piece or a three-piece case. The one-piece case was considered too difficult to mass produce while the three-piece, although easier to fabricate, was deemed too complex to meet the stringent requirements of a cartridge case. The forming technique chosen was an impact extrusion process (Figure 16) as the first approach in fabricating the unusually-shaped, two-piece folded case. For comparison, the standard blank cup and draw process used for fabricating the one-piece 5.56 mm bottleneck case, is shown in Figure 17. As can be seen, case processing steps have been reduced from 14 for the conventional to 8 for the folded. Of course, machine production rates must be taken into consideration as part of any comparative procedure, among other things, but the impact extrusion process used here enables a reduction in process steps when compared to conventional case fabrication. On the other hand, the folded case was fabricated in two parts and this then increased the number of fabrication steps for the total case. Even so, a decision was made and carried through which resulted in a practical experimental method for case fabrication. Steel was selected as the folded cartridge case material for its cost-savings potential since it is approximately 40 percent of the cost of brass, the material from which standard cases are made. Aluminum and plastics were considered as material candidates but were ruled out at this time although they could be considered as possible candidates in the future.

Having made these three crucial decisions, it remained to develop specific tooling and processing operations to fabricate the experimental Folded Ammunition cases. The tool drawings for the extrusion process for the 5.56 mm folded cartridge case are shown in Appendix B. Although many problems surfaced during this development, one of the major hurdles centered around extruding a steel with a high enough carbon content to achieve what were considered to be desired case mechanical properties (approximately 164 kpsi yield strength) through heat treatment. The lowest carbon steel which can be heat treated to the desired level is 1035, a steel which is not suitable for impact extruding at the degree of reduction required for the process. Thus, an investigation was conducted to develop a means for circumventing this problem. The approach taken utilized a carburizing technique which would be

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Figure 16. Initial Extrusion Process for Folded Cartridge Case

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Figure 17. Blank, Cup, and Draw Process for Standard Ammunition

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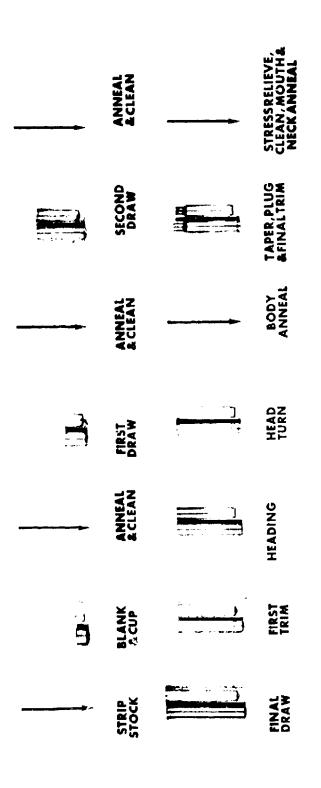
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5.56mm MI93 CARTRIDGE



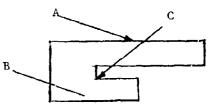
workable on a thin-walled structure such as the folded cartridge case. The case, therefore, was extruded from 1008 steel (the lowest carbon steel (commercially available and extrudable at the required degree of reduction), carburized to 1035 steel and then heat treated to the desired property levels. Figure 18 shows the uniformity or homogenity of such carburized microstructure and Table III lists the hardness readings obtained on the heat-treated carburized cases.

| | Reading | Hardnes | Hardness - Rc | | |
|----|----------|---------|---------------|--|--|
| | Position | Case 1 | Case 2 | | |
| | 1 | 29 | 32 | | |
| | 2 | 35 | 34 | | |
| | | 39 | 41 | | |
| | 4 | 32 | 33 | | |
| | 5 | 38 | 38 | | |
| TT | 6 | 36 | 36 | | |
| | 7 | 34 | 34 | | |

| | | 1 | CABLE III. | | |
|----------|----------|----|--------------|------------|-------|
| Hardness | Readings | of | Heat-Treated | Carburized | Cases |

It was found in subsequent testing that the carburized and heat treated cases were not sufficiently ductile to preclude splitting during test firings. In fact, it was found that uncarburized 1008 steel cases did possess sufficient ductility wherein the as-formed case could be successfully fired without splitting. This advantageous situation resulted in a significant simplification of the case process and it was found that satisfactory case ejection could be obtained in automatic test firing fixtures. The hardness pattern for this case is shown in Table IV. The final case process, then, is that shown in Figure 19.

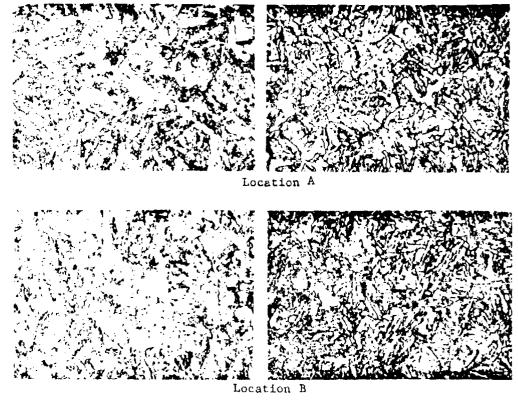
Loading and assembly of the experimental Folded Ammunition was accomplished in the manner depicted in Figure 20. The process consisted of inserting the projectile in the case to form a closed case body into which the propellant was loaded. The cap was primed separately and then joined to the loaded case body. Although this differs from the control cartridge assembly, these steps should be amenable to production processing. Shown in Figure 21 in both external and cutaway views is the assembled 5.56 mm folded cartridge used in this feasibility study. Details of cartridge and components are contained in Appendix C.

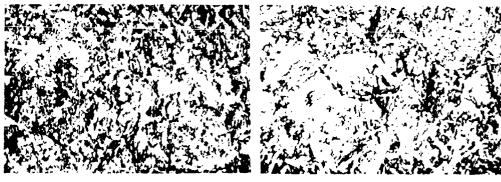




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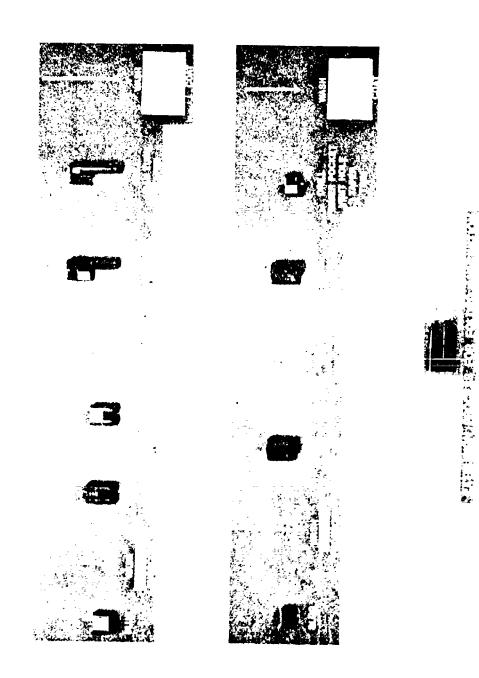






NOTE: Overall Carbon Content: 0.322





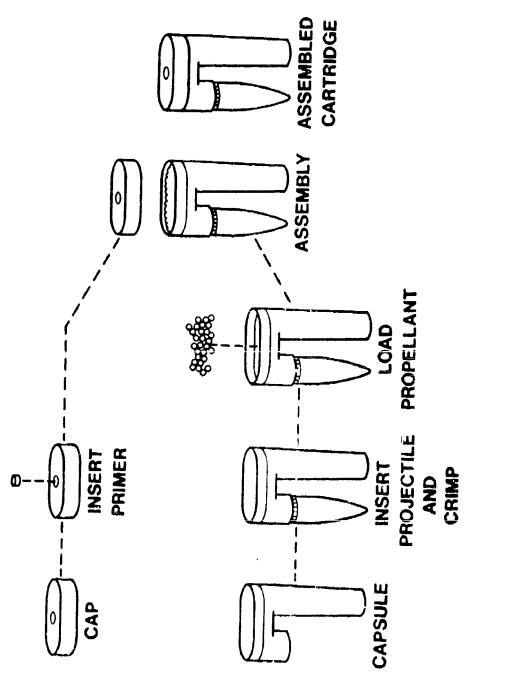
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Figure 19. Final Extrusion Process for Folded Cartridge Case

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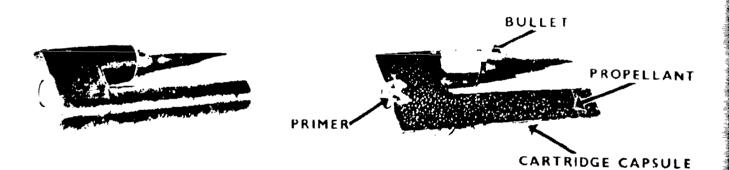
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| | Reading | Hardnes | ss - Rc |
|---|----------|-------------|---------|
| | Position | Case 1 | Case 2 |
| | | | |
| | 1 | B61 | В58 |
| | 2 | 368 | в74 |
| | 3 | 24 | B98 |
| | 4 | 22 | 22 |
| | - 5 | 22 | 24 |
| | 6 | 22 | 24 |
| \ | 7 | B 59 | B58 |

Hardness Readings of Heat-Treated Uncarburized Cases

B - Rockwell "B" readings given where too low to read on "C" scale



FOLDED AMMUNITION

Figure 21. Assembled 5.56 mm Folded Cartridge

Automatic Firing Test Fixtures

The ability to feed a two-pronged folded cartridge on an automatic basis posed a question that could only be answered by actual experimentation. In the interest of minimizing cost and taking maximum advantage of existing hardware, automatic test fixtures were evolved through modification of standard weapons. Two weapons, the M16A1 and a Belgian Light Automatic rifle were chosen to provide latitude to pursue different design approaches based upon two significantly different operating mechanisms. In each case, the weapon was fitted with a 5.56 mm barrel chambered for the folded 5.56 mm FABRL cartridge. Magazines were designed for feeding the Folded Ammunition and appropriate modifications were made to the bolt and bolt carrier to provide the proper feed and mating with the folded design chamber. In order to make use of the highly desirable front end case ejection advantages of the folded system approach, ejector systems were fitted to the weapons and gas operating systems modified to provide satisfactory cartridge case removal. The modified weapons are shown in Figure 22. Details of the new and modified parts for both weapons are contained in Appendixes D and E.

Firing tests were conducted in, both weapons to provide empirical data for refinement of operating mechanisms design resulting in evolution of fixtures which satisfactorily demonstrated that Folded Ammunition could be automatically fed, fired and ejected. Figure 23 shows a firing sequence representing a six-round burst from the modified Ml6Al test fixture at a cyclic rate of 706 s.p.m.

The firing cycle shown starts with the bolt in battery with a round in the chamber. Propellant gas from the first round fired pressurizes the bolt carrier. Then in proper sequence, the bolt unlocks and the expeller lever engages the expeller shaft pushing the fired case out of the chamber. As the bolt continues rearward, it permits the next round to enter from the magazine which is located horizontally on the other side of the weapon. This round pushes the fired case from the bolt face and into the ejection port as it moves into position for chambering. The return stroke of the bolt completes the ejection of the fired case and chambers the next round completing the cycle.



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Figure 23 a. Modified MI6Al Firing Sequence



Figure 2 (b) 2000 b) 2000 (b) 2000 b)

CONCLUSIONS AND RECOMMENDATIONS

The feasibility of the folded ammunition/weapon system has been demonstrated by automatic test firing of experimental prototype ammunition in automatic firing fixtures. Pertinent technology has been established in areas that are essential to the success of this new desIgn approach. These include: first, capability to match ballistic performance of conventional ammunition. Secondly, utilization of a two-piece case which performed satisfactorily in the firing test. Thirdly, the ammunition was fabricated using state-of-the-art impact extrusion machinery. Finally, it was shown that the folded shape ammunition could be automatically fed, fired and ejected with indications that this would not offer a stumbling block in the development of systems using this principle.

As a result of the success experienced in the testing of experimental folded system hardware and the real promise of future system optimization as indicated by the advantages outlined in the introduction to this report, it is recommended that the conceptual phase of exploratory development be completed to provide combat development agencies with hardware oriented data to determine operational capabilities, doctrine and specific material requirements that will provide Army, and very possibly Tri-Service forces with the improved capabilities that can accrue from the folded system design principle. We recommend that the following approach be taken to achieve this goal.

It is felt that the most cost effective program would be to develop required hardware oriented data using small caliber as the test vehicle. In the small caliber test vehicle the common cartridge, common fed rifle and light machine gun approach would not only supply basic data but would be very advantageous in its applicability to the future small arms program. The utility of one common ammunition package unit suitable for use in the rifle or light machine gun would be joined to the most advanced projectile design. In addition to the general information that would be gleaned from a small caliber program, specific parametric studies should also be conducted in both cannon and large caliber to provide the widest spectrum of information.

APPENDIX A

US Patent No. 3857339

United States Patent 1191

Grandy

[54] AMMUNITION AND WEAPON SYSTEMS

- [76] Inventor: Andrew J. Grandy, 2707 Grant Ave., North Hills, Pa. 19038
- [22] Filed: Mar. 30, 1972
- [21] Appl. No.: 239,595

- [58] Field of Search..... 102/38, 40, 43, 43 P, 44, 89/35, 35 A, 33

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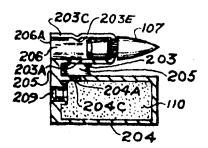
Primary Examiner-Benjamin A. Borchelt Assistant Examiner-H. J. Tudor

[57] ABSTRACT

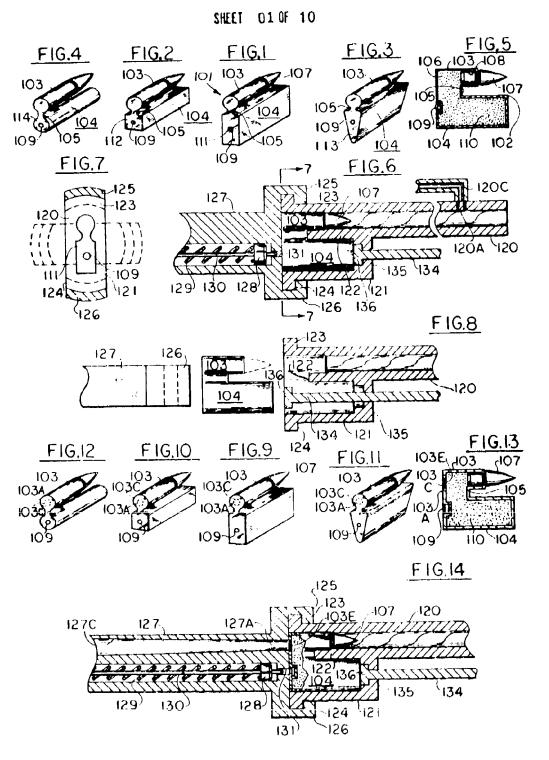
A weapon system employing encapsulated ammunition in which the pressure chamber, located axially rearward of the projectile, is longitudinally or axially offset from but in fluid communication with the propellant capsule chamber. This permits use of ammunition rounds having reduced length for given characteristics of prior rounds, resulting in lightest weight and improved bulk characteristics for the ammunition as well as associated weapon systems.

This ammunition concept is adaptable to recoilless, partially recoilless and closed breech ballistic systems in a variety of arrangements.

102 Claims, 67 Drawing Figures



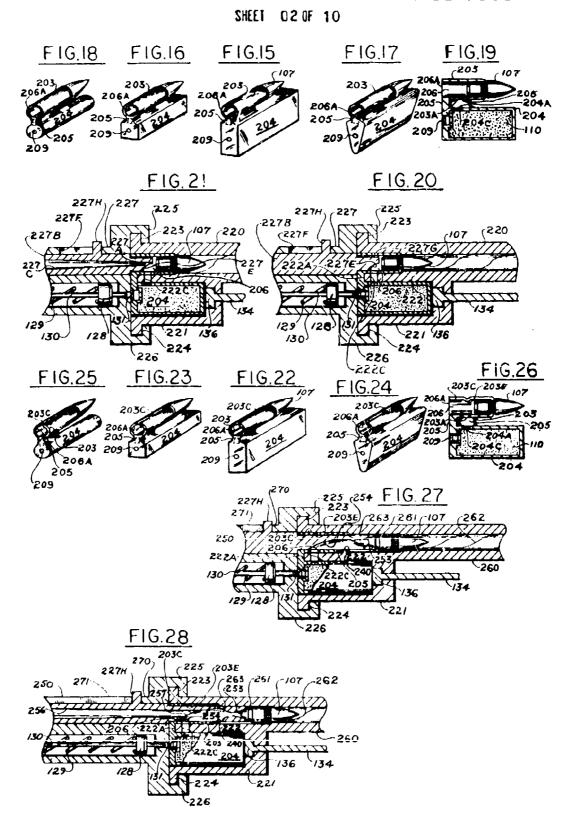
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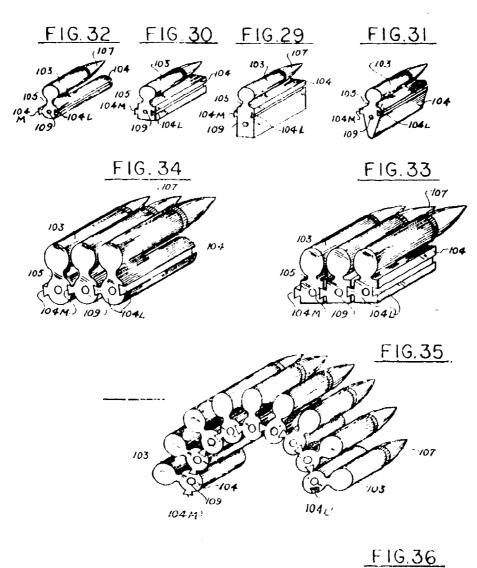
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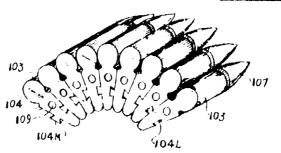
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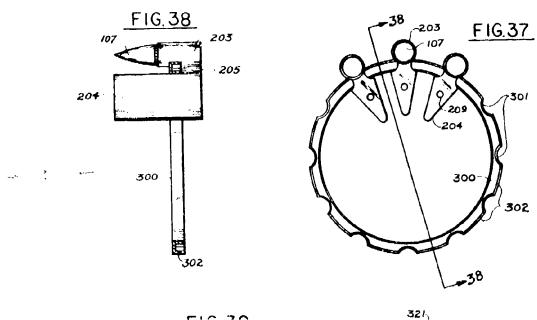
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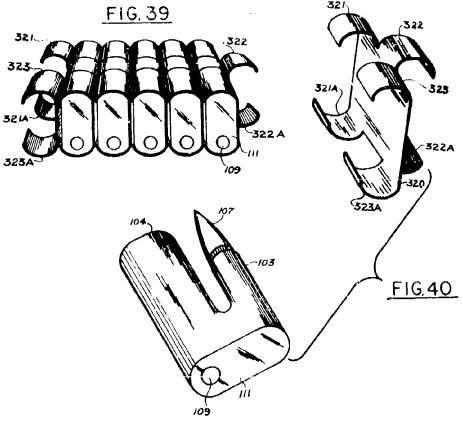
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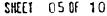
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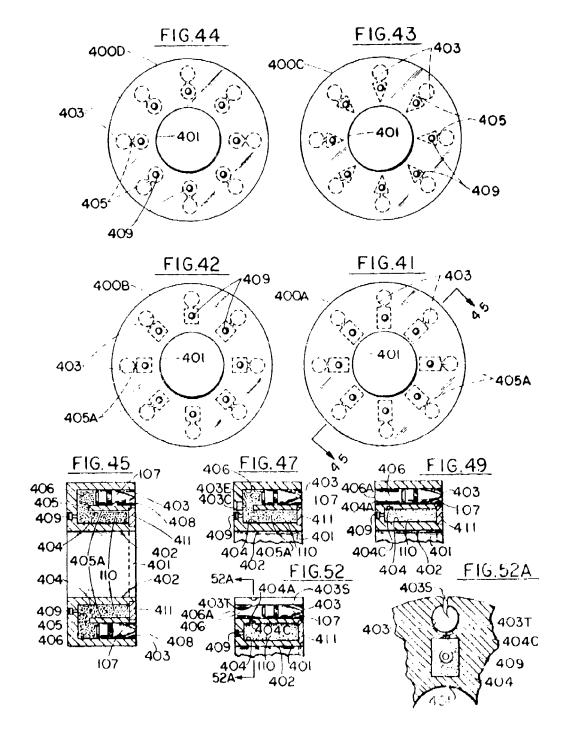
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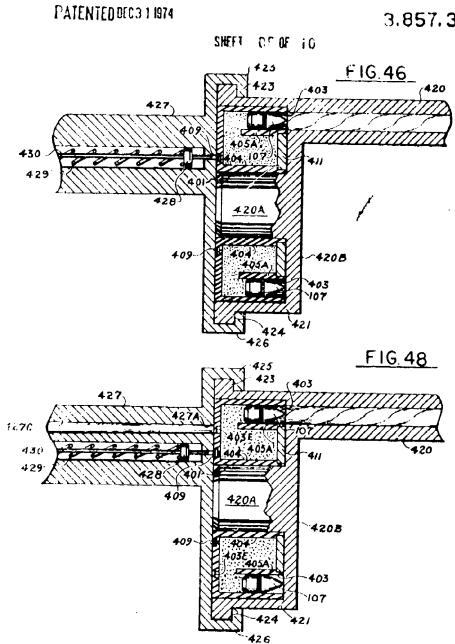




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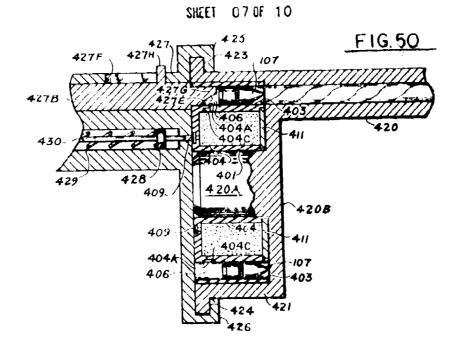


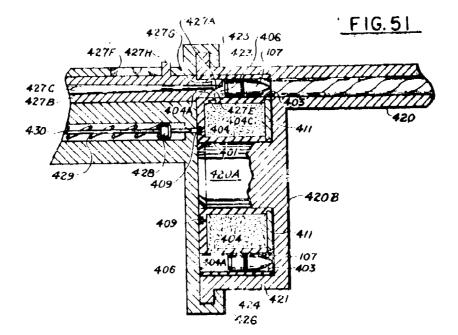


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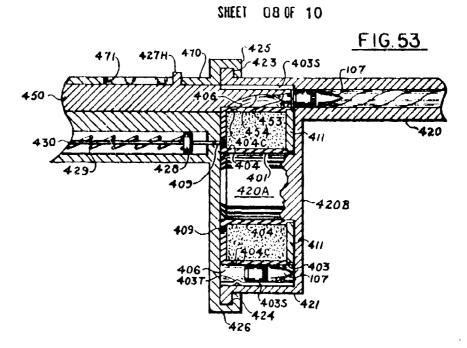
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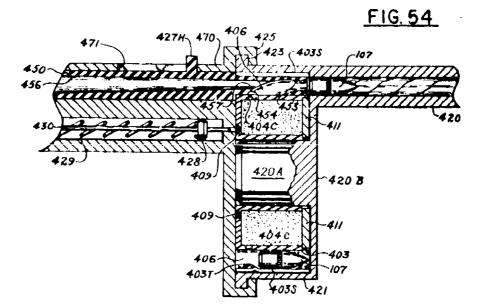




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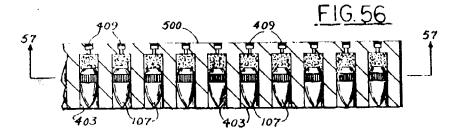
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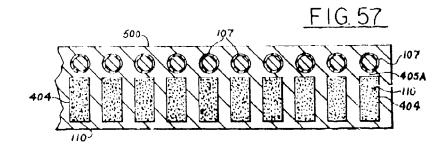
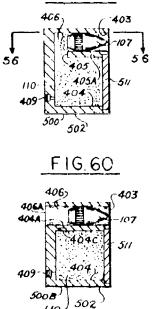
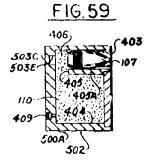


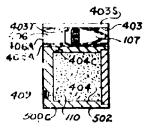
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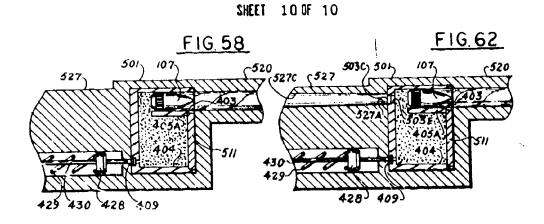








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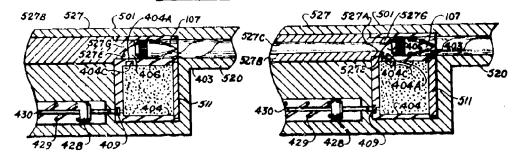
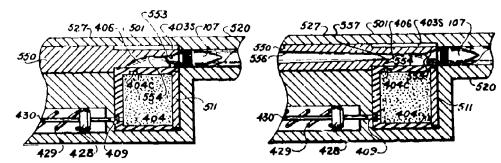


FIG.65

FIG.66



AMMUNITION AND WEAPON SYSTEMS

The invention described herein may be manufactured, used, and licensed by or for the Government for governmental purposes without the payment to me of 5 any royalty thereon.

This invention relates to ammunition and weapon systems therefor, and more particularly to a variety of such systems each having the capacity to utilize cartridge capsule ammunition.

Present day cylindrical ammunition does not represent the most efficient, over-all cartridge with respect to bulk and weight

It is an object of the invention to provide weapon arrangements and cartridge capsule animunition rounds for use therewith which are of lightness in weight and have improved bulk characteristics.

Another object of the invention is to provide such ammunition and weapon arrangements that can be advantageously used as partially recoilless, fully recoilless or closed breech ballistic systems. 20

A further object of the invention is to provide such arrangements that can be advantageously used in either fixed projectile or bolt rammed projectile systems.

A further object of the invention is to provide such arrangements that facilitate an improved manner of removing spent ammunition rounds from the firing chamber of the weapon.

A still further object of the invention is to provide means for attaching the cartridges in arrangements that can be advantageously used in single shot, semiautomatic and fully automatic weapons.

Another object of the invention is to provide specific capsule cartridge shapes resulting in efficient packag- 35ing arrangements which can be advantageously used in rotary fed and in-line fed single shot, semi-automatic and fully automatic weapons.

A further object of the invention is to provide both individual capsule cartridges and multi-cavity, unitized 40 cartridge arrangements which can be advantageously used in rotary fed and in-line fed, single shot, semiautomatic and fully automatic weapons.

These and other objects, features and advantages will become apparent from the following description and 45 accompanying drawings in which:

FIGS. 1-4 are perspective views of a variety of ammunition rounds embodying the principles of the invention.

FIG. 5 is a longitudinal sectional view of the FIG. 1 50 round.

FIG. 6 is a longitudinal sectional view of a portion of a closed breech weapon system for the FIG. 5 round.

FIG. 7 is a sectional view taken substantially along lines 7--7 of FIG. 6.

FIG. 8 is an exploded view of certain portions of the FIG. 6 arrangement subsequent to firing.

FIGS. 9-13 are views, similar to FIGS. 1-5, of a modified group of amnunition rounds.

FIG 14 is a longitudinal sectional view of a portion of a recoilless or partially recoilless weapon system for the FIGS 9-13 rounds.

FIOS. 15-19 are similar views of a modified group of ammunition rounds

TIGS 20 and 21 are longitudinal sectional views of ⁶⁵ modified portions of weapon systems associated with the rounds of FIGS 15-19.

FIGS. 22-26 are views, similar to FIGS. 15-19, of a modified group of animunition rounds.

FIGS. 27 and 28 are longitudinal sectional views of modified portions of weapon systems associated with the rounds of FIGS. 22-26.

FIGS. 29-32 are perspective views of a further modified group of rounds.

FIGS. 33 and 34 are perspective views of clusters of capsules of the FIG. 30 and FIG. 32 rounds, respectively.

FIGS. 35 and 36 are perspective views of clusters of capsules of the FIG. 32 and FIG. 31 rounds, respectively.

FIG. **37** is an end view of a modified form of cluster.

FIG. 38 is a sectional view taken along line 38---38 of FIG. 37.

FIG. **39** is a perspective view of a modified cluster arrangement.

 FIG. 40 is an exploded perspective view of a linking clip member and amnunition round used in the FIG. 39 arrangement.

FIGS. 41-44 are end views of integral multiple cavity containers of cartridge capsule portions arranged in cy-25 lindrical form.

FIG. 45 is a sectional view taken along line 45--45 of FIG. 41.

FIG. 46 is a longitudinal sectional view of a portion of a closed breech weapon system for the FIGS, 41-44 animunition.

FIG. 47 is a longitudinal sectional view, partially broken away, of multi-cavity container modifications for the FIGS. 41-44 cylinders.

 FIG. 48 is a longitudinal sectional view of a portion
 of a recoilless or partially recoilless weapon system for the FIG. 47 ammunition containers.

FIG. 49 is a view, similar to FIG. 47, of further multicavity container modifications for the FIGS. 41-44 ammunition cylinders.

FIGS. 50 and 51 are longitudinal section views of modified portions of weapon systems associated with the FIG. 49 ammunition containers.

FIG. 52 is a view, similar to FIG. 49, of further multicavity container modifications for the FIGS. 41-44 ammunition cylinders, and FIG. 52A is a partial sectional view taken along line 52A-52A of FIG. 52.

FIGS. 53 and 54 are longitudinal sectional views of modified portions of weapon systems associated with the FIG. 52 ammunition containers.

FIG. 55 is a longitudinal sectional view of a propellant capsule portion of a multi-cavity rectangular prism ammunition cluster.

FIG. 56 is a partial sectional view taken along line 55 56-56 of FIG. 55.

FIG. 57 is a sectional view taken along line 57-57 of FIG. 56.

FIG. 58 is a longitudinal sectional view of a portion of a closed breech weapon system for the FIG. 55 ammunition prisms.

FIGS. **59-61** are views similar to FIG. **55** of modified arrangements.

FIG. 62 is a longitudinal sectional view of a portion of a recoilless or partially recoilless weapon system for the FIG. 59 ammunition prisms.

FIGS. 63 and 64 are longitudinal sectional views of modified portions of weapons systems associated with the FIG. 60 ammunition prisms.

FIGS. 65 and 66 are longitudinal sectional views of modified portions of weapon systems associated with the FIG. 61 ammunition prisms.

The FIG. 1 encapsulated ammunition round shown generally at 101 meludes a thin capsule 102 (FIG. 5) 5 of ferrous, non-ferrous or synthetic material of predetermined contour. Preferably ferrous, the cartridge capsule has an elongated substantially cylindrical chamber 103 and a propellant chamber 104 integral mediate necked down metering orifice 105 interconnects or fluidly communicates the propellant chamber 104 with the pressure chamber portion 106 of the cylindrical chamber 103. A projectile 107 is slidably received in the forward barrel portion 108 of chamber 15 103 in longitudinal alignment with the pressure chamber portion 106. An appropriate percussion primer 109 is secured in a suitably recessed and apertured rear wall portion of the capsule 102, preferably aligned with the propellant chamber 104 which contains the desired 20 ber which is integrally connected with chamber 203 by granular, flake, sheet or solid grain propellant 110. The transverse sections and end walls 111 of the FIG. 1 round propellant chamber are of substantially rectangular configuration. The embodiments of FIGS. 2, 3, FIG. 1 round, but the transverse sections and corresponding end walls 112, 113, and 114 are of configurations which are substantially square, triangular, and cy-Indrical, respectively.

(FIGS. 6-8) includes a substantially cylindrical barrel 120 having an axially offset capsule chamber 121 integral therewith, both the barrel and capsule chamber having a somewhat shortened common wall portion 122 to accommodate the FIG. 1 round prior to firing 35 (FIG. 6) which function is accomplished after the barrel lug 123 and capsule chamber lug 124 are simultaneously engaged by the opposed hook or locking lugs 125, 126 of the rotatable and longitudinally translatable breech means or bolts 127. The closed breech type bolt 127 has a firing pin 128, biased by spring 129 sur rounding the firing pin rod or stem 130, with its receptive breech opening 131 in operative alignment with the primer of the round to be fired. The barrel 120 has a transverse opening 120A and an appropriate conduit 120C to direct gas energy for further weapon operation. An ejection rod 134 is slidably mounted in an apertured forward wall 135 of capsule chamber 121 to rearwardly eject a fired or undesired round when the bolt 127 has been rotated out of locking engagement and translated rearward (FIG. 8). The chamber wall 135 is internally recessed to normally seat the ejection rod head 136. The capsule chamber 121 has a crosssection of substantially rectangular configuration to receive the propellant capsule portion of the FIG. 1 round. For firing of the FIGS. 2, 3, or 4 rounds, this transverse section configuration is substantially square, triangular or cylindrical, respectively.

The ammunition rounds of FIGS. 9-13 are distinguished from the FIGS. 1-5 rounds in that the upper cylindrical portion 103A of the thin capsule rearward walls each have a press fitted blow out disc 103C or a pre-formed weakened section as defined by an internal recess 103E (FIG-13) in the pressure chamber rearward wall portions. The disc or weakened section 103C operatively aligns with the recoilless or partially recoilless weapon nozzle 127A (FIG-14) of the rearwardly

enlarging tapered recoil vent 127C provided in the breech or bolt 127. The forward internal surface of the nozzle 127A is formed with a substantially sharp annular edge to facilitate positive shearing action upon the operative portion of the weakened section 103C when sufficient propellant pressure is generated in the pressure chamber portion of cylindrical chamber 103 by the ignited propellant 110.

The ammunition rounds of FIGS. 15-19 contain sevtherewith but longitudinally offset therefrom. An inter 10 eral variations for their operative firings in the fixed projectile, inserted bolt weapon systems of FIGS. 20 and 21. The capsule cylindrical chamber 203 has an open rearward end 206A adjacent its pressure chamber portion 206, and contains a bottom gas vent 203A which is substantially in vertical alignment with the blow out disc or pre-formed weakened section 204A defined by internal recess 204C in the upper wall of capsule propellant chamber 204. A suitable primer 209 is located in the rearward wall of the propellant chamthe forwardly opening cavity side and rearward walls 205.

In both the FIG. 20 closed breech and FIG. 21 recoilless or partially recoilless weapon systems the common and 4 are of substantially the same construction as the 25 wall portion 222, for the cylindrical barrel 220 and axially offset capsule chamber 221, terminates with a rearwardly extending tongue 222A that fits into the connecting cavity and abuts the cavity rear wall 205 upon loading of the round. Fongue 222A has a vertical gas The portion of the closed breech weapon system 30 vent passage 222C that places capsule chambers 203 and 204 in fluid communication upon firing of the round as the developed pressure gas blows out a portion of the weakened section 204A at the sharp edged passage 222C and enters the pressure chamber 206 adjacent the concave recess 227E provided in the forward face of the reduced tip portion 227G on the bolt 227B. The bolt 227B is longitudinally translatable in the rotational and longitudinally translatable breech member 227. With bolt lug or pin 227H slidable in longitudinal 10 slot 227F of member 227, the bolt is withdrawn rearwardly prior to operative disengagement and engagement of the breech nooking or locking lugs 225, 226 with the barrel lug 223 and capsule chamber lug 224. A recoilless or partially recoilless weapon nozzle 227A 45 (FIG. 21) is provided in bolt 227B at the juncture of the rearwardly enlarging tapered recoil vent 227C and the concave recess 227E

> The ammunition rounds of FIGS. 22-26 are distinguished from the FIGS. 15-19 rounds in that the upper 50 portion of the capsule cylindrical chamber 203 has a lengthwise split or slotted wall 203C. A plurality of camming or diniple means 203E are provided on the internal surface of the pressure chamber portion 206 adjacent the slotted wall 203C and just rearward of projectile 107. This will facilitate the release and barrel chambering of the projectile as the longitudinally translatable bolt 250 (FIGS. 27 and 28) cams the dimple means 203E to spread the inwardly directed substantially annular flange 240 and pushes the projectile to its 60 final pre-firing position in barrel 260. Cylindrical barrel 260 differs from barrel 220 in that an additional intermediate internal recessed surface 261 is provided rearwardly adjacent the rifling 262 to accommodate the forwardmost portion of bolt 250 in its firing position. 65 while a further recessed surface 263 accommodates the spread portions of split chamber wall 203. The longitudinal slot 271 of the rotational an longitudinally trans

latable breech member 270 is substantially longer than slot 227F, such that the lug or pin 227H on bolt 250 can be accommodated for its intended full stroke. The forward end of bolt 250 has a forwardly opening nozzle surface 253 that intersects with a concave recess 254 provided on the underside of the bolt, so that the pressure chamber portion 206 fluidly communicates gas vent passage 222C with the rearward end of projectile 107 in both the EIG. 27 closed breech and EIG. 28 recoilless or partially recoilless weapon systems. The 10 rearwardly enlarging tapered recoil vent 256 (HG. 28) in bolt 250 intersects concave recess 254 at the effective recoilless nozzle throat 257.

The ammunition rounds of HGS. 29-32 are each provided with a longitudinally extending groove 1041, 15 loop 322A of the adjacent link. The substantially of dovetail transverse configuration throughout the entire length of a selected propellant capsule chamber wall portion 104, as well as a rib or tongue protuberance 104M of substantially similar dovetail configuration along the length of an opposed longitudinally ex- 20 tending wall portion. Where each substantially similar round has its corresponding tongue and groove suifaces 104M, 104L extending in parallel planes, successive similar cartridge capsules can be integrally linked or connected (FIGS: 33, 34) in a substantially straight 25 line cluster pattern. The corresponding dovetailed tongue and groove surfaces of similar rounds can also be formed or oriented in non-parallel planes such that successive similar capsules can be integrally linked or matingly joined to form an arcuate or circular cluster 30 pattern or array (FIGS, 35, 36). Each cluster may be held in an appropriate feeding and stripping mechanism (not shown) which will enable the corresponding breech member to longitudinally translate each stripped round into its weapon position prior to rota- 35 tional locking motion of the breech member. It is contemplated that each of the previously described rounds (FIGS. 5, 13, 19, and 26) may be so dovetailed for integral linking or connecting purposes, and each fired in 40 their respective weapons which can be slightly altered. to accommodate the protruding rib or tongue. The breech locking arrangement on each of the respective weapons also can be relocated to a position somewhat rearward to enable the forward portion of the bolt to 45 have a cross-sectional shape similar or identical to that of the particular cartridge to be chanibered.

A modified cluster arrangement (FIG. 37) includes an annular linking member 300 of substantially rearwardly opening U-shaped configuration (FIG. 38) have 50 ing a plurality of equally spaced arcuate or concave recesses 301 along its peripheral or outermost surface 302 to accommodate an arcuate undersurface portion of the capsule cylindrical chamber 203 on the FIG-17 ammunition rounds. The rounds are slid forward onto-55 the metal link ring 300 at each recessed surface 301 such that the ring cradles each round at the rearmost clearance between its propellant chamber 204 and projectile 107 or its supporting cylinder 203. It is conteniplated that the ring 300 be formed to cluster similar 60 rounds from any of the groups associated with FIGS. 5. 13, 19 or 26 and that the cluster can be suitably mounted and indexed on weapon system cylindrical stubs of the type to be later described with unitary and radially arranged ammunition containers or clusters. 65

The modified cluster arrangement (FIG: 39) employs a plurality of individual clip members 320 (FIG, 40) which are shown to be joined or linked by insertion of

respective FIG: 1 capsule cartridges 101 into selected substantially C-shaped wings or clip portions of adjacent clips 320. Each link member 320 is formed from a pre-slotted blank or metal member and contains a substantially straight integral or common portion and upper and lower groups of alternately disposed loop portions 321, 322, 323 and 321A, 322A, 323A. The upper loops are formed to enable simultaneous reception of the cartridge capsule cylindrical chamber 103 and projectile 107 by longitudinally spaced loops 323, 321 of one link and intermediate loop 322 of an adjacent link, while the substantially rectangular propellant capsule portion 104 is clipped or received by respective loop portions 323A, 321A of the one link and middle straight line type of cluster (FIG. 39) so formed can be used in the same manner as the unitary rectangular ammunition containers or clusters to be later described. Clip members 320 can also cluster separate groups of cartridges of FIGS. 9, 15, 22. The width of the straight common portion of the C shaped clip portions can be reduced such that similar clips can cluster separate groups of cartridges of FIGS. 2, 10, 16, 23 as well as FIGS. 4, 12, 18, 25. The clipping or elustering of senarate groups of cartridges of FIGS. 3, 11, 17, 24 can be accomplished where the pre-formed slits of the clip blank are of sufficient length that the lower loops can be bent or inclined to facilitate insertion of the rounds The integral or unitary multiple cavity containers 400A, 400B, 400C, 400D (FIGS. 41-44) are preferably made of ferrous, non-ferrous or synthetic material is evlindrical form to contain a cluster of cartridge capsule portions similar to the respective ammunition rounds of FIGS. 1-4. The annular container 400A has a central cylindrical opening 401 (FIGS, 41,45) and includes a plurality of circumferentially spaced cavities that are defined by integral cartridge capsule portions 402 each having a cylindrical chamber 403 and longitudually offset propellant chamber 404 which is interconnected by metering orifice 405 at the rearmost edge of separating wall 405A. Orifice 405 fluidly communicates the propellant chamber 404 with the pressure chamber portion 406 of chamber 403 whose forward barrel portion 408 slidably receives projectile 107. A suitable primer 409 is secured in an appropriately recessed and apertured rear wall portion of each cartridge capsule portion 402, preferably aligned with propellant chamber 404 that contains the desired propellant 110. An appropriate substantially annular ring shaped end wall closure 411 is suitably secured with cement or the like to seal the forward end of each propellant chamber 404 having a rectangular transverse configuration. Containers 400B, 400C, and 400D will require similar ring end wall closures that suitably seal their respective propellant chambers of substantially square, triangular and cylindrical configurations. The central opening 401 of each annular container

400A, 400B, 400C, 400D, is slidingly received on the cylindrical stub 420A (FIG. 46) that extends rearwardly from the weapon housing 420B and parallel to but substantially offset or below the barrel 420 of the closed breech weapon system for the FIGS, 41-44 cy-Indrical capsule containers or clusters. Preferably, stub 420A has a tapered rearward end and is centrally located within the rearwardly opening weapon housing annul it well 421 to facilitate the reception of the cylindrical or annular containers that can be delivered by

the longitudinally translatable breech means or bolt 427 prior to rotation of the breech to simultaneously secure the opposed hook or locking lugs 425, 426 in engagement with the weapon housing lugs 423, 424. After each successive cartridge capsule portion is suit ably indexed to a firing position by means not shown, actuation of spring 429 surrounding the rod or stem 430 of firing pin 428 will initiate the aligned primer 409 to fire the round. When all of the rounds in the cluster or container have been fired, bolt 427 is rotated out of (0 locking engagement and translated rearward to permit the spent container to be replaced by another multicapsule cluster or container.

The multi-cavity ammunition containers of FIG. 47 are distinguished from the FIG. 45 ammunition cylin-15 ders in that the rearward walls of each cartridge capsule portion 402 have a blow out disc or pre-formed weakened section 403C as defined by internal recess 403E adjacent the pressure chamber portion 406. Each recess 403E operatively aligns with the recoilless or 20 partially recoilless weapon sharp edged nozzle 427A (FIG. 48) of the rearwardly enlarging tapered recoil vent 427C provided in the breech or bolt 427.

The ammunition containers of FIG. 49, that are fired FIGS. 50 and 51, are distinguished from the FIG. 45 cylindrical clusters in that each cartridge cansule portion 402 has its cylindrical chamber 403 defined by a through bore to provide an open reasward end 406A tion of reduced tip portion 427G (FIGS, 50, 51) on the forward end of the bolt 427B that is longitudinally translatable in the rotational and longitudinally translatable breech member 427. Preferably, a rearward upper wall portion of each capsule propellant chamber 35 404 is internally recessed at 404C to provide a preformed weakened section or blow-out disc 404A in the full common wall that separates cavities 403 and 404. When each primer 409 is actuated or initiated, developed pressure gas blows out the weakened section or disc 404A and enters pressure chamber 406 adjacent the concave recess 427E provided in the forward face of bolt tip 427G. With bolt pin 427H slidable in longitudinal slot 427F of member 427, the bolt is withdrawn substantially simultaneous with indexing of the associated ammunition cylinder by means not shown, and when a replacement ammunition cylinder is required. bolt withdrawal is followed by operative disengagement and engagement of the breech or locking lugs 425, 426 50 with the weapon housing lugs 423, 424. A recoilless or partially recoilless weapon nozzle 427A (FIG. 51) is provided in bolt 427B at the juncture of the rearwardly enlarging tapered recoil vent 427C and the concave recess 427E

The multi-cavity ammunition containers of FIG. 52 55 are distinguished from the FIG. 47 ammunition cylinders in that each cartridge capsule portion 402 has the upper portion of its cylindrical chamber 403 provided with a lengthwise split or slotted wall 403S (FIGS. 52, 52A). A plurality of camming or dimple means 403T are provided on the internal surface of each pressure chamber portion 406 adjacent the slotted wall 403S just rearward of each projectile 107. This facilitates the release and barrel chambering of the projectile as the 65 longitudinally translatable bolt 450 (FIGS 53 and 54) cams the dimple means 40.31 to spread the slotted wall and pushes the projectile to its final pre-firing position

in barrel 420. The internal surface of the housing annular wall 421 is of appropriate dimensions that sufficient clearance is provided for lateral spreading of the split cylindrical chambers 403 during the projectile barrel chambering operations. The longitudinal slot 471 of the rotational and longitudinally translatable breech member 470 is substantially longer than slot 4271-(FIGS. 50, 51), such that the pin 427H on bolt 450 can be accommodated for the intended full stroke of the bolt. The forward end of bolt 450 has a forwardly opening nozzle surface 453 that intersects with a concave recess 454 provided on the underside of the bolt so that each pressure chamber portion 406 fluidly communicates developed pressure gas from the respective propellant chamber 404 with the rearward end of the corresponding projectile 107 during operation of both the FIG. 53 closed breech and FIG. 54 recoilless or partially recoilless weapon systems. The rearwardly enlarging tapered recoil vent 456 (FIG. 54) in bolt 450 intersects concave recess 454 at the effective recoilless nozzle throat 457.

The unitary or integral multiple cavity ammunition cluster 500 (FIGS, 55-57) is a rectangular prism which is made of materials similar to the aforementioned mulin the fixed projectile, inserted bolt weapon systems of 25 ti-cavity cylindrical ammunition containers and includes a plurality or cluster of cartridge capsule portions 502 that are integrally arranged in laterally spaced positions across the prism as it is successively moved or fed transversely through the aring chamber adjacent its pressure chamber portion 406 for recep- 30 501 (FIG. 58) of a closed breech weapon system having an integral arrangement for the closed breech or bolt and barrel portions 527 and 520, respectively. Each cartridge capsule portion 502 is very similar to the cartridge capsule portions 402 of the FIG. 45 ammunition cylinder. Preferably, a rectangular shaped or clongated cover 511 is cemented to an appropriate receptacle therefor extending across the forwardmost portions of the successively arranged propellant chambers 404. While each of the propellant cavities 404 in 40 the FIG. 55 elongated ammunition prism has been formed with a transverse section of substantially rectangular configuration, the propellant cavities 404 of

the rectangular prism may also be of substantially square, triangular and cylindrical configuration, and the end wall closure or scal 511 would not require moddication

The type of multi-cavity ammunition prism 500A (FIG. 59) are distinguished from the FIG. 55 ammunition prisms 500 in that the rearward walls of each cartridge capsule portion 502 for each of the prisms is provided with a blow out disc or a pre-formed weakened section 503C as defined by internal recess 503E adjacent the pressure chamber portion 406. Each recess 503E operatively aligns with the recoilless or partially

recoilless weapon sharp edged nozzle 527A (FIG. 62) of the rearwardly enlarging tapered recoil vent 527C provided in the bolt portion 527.

The type of ammunition prism 500B (FIG. 60), that are fired in the fixed projectile, inserted bolt weapon 60 systems of FIGS. 63 and 64, are distinguished from the FIG. 55 ammunition prisms 500 in that each cartridge capsule portion 502 has its cylindrical chamber 403 defined by a through bore to provide an open rearward end 406A adjacent its pressure chamber portion 406 for reception of reduced tip portion 527G (FIGS. 63, 64) on the forward end of the bolt 527B that is longitudinally translatable in breech portion 527. A rearward

upper wall portion of each capsule propellant chamber 404 is internally recessed at 404C to provide a preformed weakened section or blow out disc 404A in the full common wall that separates cavities 403 and 404. When each primer 409 is actuated or initiated, devel-5 oped pressure gas blows out the weakened section or disc 404A and enters pressure chamber 406 adjacent the concave recess 527E provided in the forward face of bolt tip 527G. An appropriate pin and slot connection (not shown) between bolt 527B and breech por- 10 tion 527 enables the bolt to be withdrawn substantially simultaneous with indexing or feeding of the associated ammunition prism through chamber 501 by means not shown. Bolt 527B is also withdrawn when a replacement ammunition prism is required. A recoilless or par-15 tially recoilless weapon nozzle 527A (FIG. 64) is provided in bolt 527B at the juncture of the rearwardly on larging tapered recoil vent 527C and the concave recess 527F

The type of animunition prism 500C (FIG. 61) are 20 distinguished from the FIG. 60 ammunition prisms in that each cartridge capsule portion 502 has the upper portion of its cylindrical chamber 403 provided with a lengthwise split or slotted wall 403S and a plurality of camming or dimple means 403T are provided on the 25 internal surface of each pressure chamber portion 406 adjacent the slotted wall 403S just learward of each projectile 107. This facilitates the release and barrel chambering of the projectile as the longitudinally translatable bolt 550 (FIGS, 65 and 66) cams the dimple 30 means 403T to spread the slotted wall and pushes the projectile to its final pre-firing position in barrel 520, The internal upper surface of the firing chamber 501 is suitably dimensioned that sufficient clearance is provided for spreading of the slotted walls. A suitable pin^{-35} and substantially long slot connection (not shown) between bolt 550 and breech portion 527 will enable full longitudinal motion of the bolt 550 throughout its intended full stroke. The forward end of bolt 550 has a forwardly opening nozzle surface 553 that intersects with a concave recess 554 provided on the underside of the bolt so that each pressure chamber portion 406 fluidly communicates developed pressure gas from the respective propellant chamber 404 with the rearward 45 end of the corresponding projectile 107 during operation of both the FIG. 65 closed breech and FIG. 66 recoilless or partially recoilless weapon systems. The rearwardly enlarging tapered recoil vent 556 (FIG. 66) in bolt 550 intersects concave recess 554 at the effec-50 tive recoilless nozzle throat 557.

Various modifications, changes and alterations may be resorted to without departing from the scope of the invention as defined by the appended claims. I claim

1. An ammunition capsule comprising,

- a unitary capsule body having an elongated longitudinally extending cylindrical chamber, said chamber having a forward barrel portion for slidably receiving a projectile and a pressure chamber portion 60 aligned with and rearward of said barrel portion.
- a one-piece imperforate propellant capsule chamber which remains imperforate and is integral with and laterally offset from said elongated chamber,
- said propellant capsule chamber having metering or 65 fice means for fluidly communicating the propelfant capsule chamber with said pressure chamber, and

a primer carried by an exterior surface of said ammunition capsule body for igniting a capsule propellant charge.

2. The structure in accordance with claim 1 wherein said propellant chamber has a transverse section of substantially rectangular configuration.

3. The structure in accordance with claim 2 wherein said transverse section is of substantially square configuration.

4. The structure in accordance with claim 1 wherein said propellant chamber has a transverse section of substantially trangular configuration.

5. The structure in accordance with claim 1 wherein said propellant capsule chamber has opposed longitudinally extending external wall portions, one of said wall portions having a longitudinally extending groove and the other wall portion having a tongue protuberance of a substantially similar configuration as said groove

6. The structure of claim 5 wherein said propellant chamber has a transverse section of substantially rectangular configuration.

7. The structure of claim 6 wherein said transverse section is of substantially square configuration.

8. The structure of claim 5 wherein said propellant chamber has a transverse section of substantially triangular configuration.

9. The structure of claim 5 wherein said groove extends to the rearward edge of said capsule.

10. The structue of claim 5 wherein said propellant chamber has a transverse section of polygonal configuration.

11. The structure of claim 5 wherein said propellant chamber has a transverse section of substantially square configuration, and said pressure chamber portion has means spaced from said primer for rearwardly exhausting operatively developed pressure gas therefrom.

12. The structure of claim 5 wherein said pressure chamber has means spaced from said primer for rearwardly exhausting operatively developed pressure gas therefrom.

13. The structure in accordance with claim 12 wherein said propellant chamber has a transverse section of polygonal configuration.

14. The structure in accordance with claim 5 wherein a tearward wall of said pressure chamber portion has a pre-formed weakened section for use with a recoil vent in breech means of a recoilless system.

15. The structure of claim 14 wherein said propellant chamber has a transverse section of polygonal configuration.

16. The structure of claim **15** wherein said transverse section is of substantially triangular configuration.

⁵⁵ 17. The structure in accordance with claim 15 wherein said transverse section is of substantially rectangular configuration.

18. The structure of claim **17** wherein said transverse section is of substantially square configuration.

19. The structure of claim 14 wherein said propellant chamber has a transverse section of polygonal configuration.

20. The structure in accordance with claim 1 wherein said propellant chamber has a transverse section of polygonal configuration.

21. The structure in accordance with claim 20 wherein a rearward wall of said pressure chamber por-

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tion has a pre-formed weakened section for use with a recoil vent in breech means of a recoilless system.

22. The structure of claim 1 wherein said propellant chamber has a transverse section of substantially square configuration, and said pressure chamber por- 5 tion has means spaced from said primer for rearwardly exhausting operatively developed pressure gas therefrom.

23. The structure of claim 1 wherein said metering orifice means is a necked down passage interconnect- 10 ing said capsule propellant chamber and said pressure chamber.

24. The structure of claim 23 wherein said propellant chamber has a transverse section of substantially square configuration.

25. The structure in accordance with claim 1 wherein a rearward wall of said pressure chamber portion has a pre-formed weakened section for use with a recoil vent in breech means of a recoilless system.

26. The structure of claim 25 wherein said propellant 20 ration chamber has a transverse section of polygonal configuration. 44.

27. The structure in accordance with claim 25 wherein said propellant chamber has a transverse section of substantially rectangular configuration.

28. The structure of claim 27 wherein said transverse section is of substantially square configuration.

29. The structure of claim 25 wherein said propellant chamber has a transverse section of substantially triangular configuration.

30. The structure in accordance with chaim 1 wherein wall portions of said clongated chamber and propellant capsule chamber are connected by a transversely extending wall, said chamber wall portions being transversely spaced from each other and having transversely 35 aligned operative port means defining said metering orifice means.

31. The structure of claim 30 wherein said propellant chamber has a transverse section of polygonal configuration.

32. The structure of claim 30 wherein said propellant chamber has a transverse section of substantially square configuration, and said pressure chamber poition has means spaced from said primer for rearwardly exhausting operatively developed pressure gas therefrom.

33. The structure of claim 30 wherein said propellant chamber less has longitudinally extending external wall portions including means for connecting an adjacent $\frac{50}{50}$

34. The structure of claim 30 wherein said pressure chamber portion has means spaced from said primer for rearwardly exhausting operatively developed pressure gas therefrom.

35. The structure in accordance with claim 34 wherein said propellant chamber has a transverse section of polygonal configuration.

36. The structure of claim 30 wherein said port means includes a gas vent in said characted chamber wall portion and a pre-formed we dense section in said propellant capsule chamber wall pertion

37. The structure of claim 36 wherein said propellant chamber has a transverse section of polygonal configuration.

38. The structure of claim **36** wherein said propellant chamber has a transverse section of substantially square configuration, and said pressure chamber por-

tion has means spaced from said primer for rearwardly exhausting operatively developed pressure gas therefrom.

39. The structure of claim 36 wherein said propellant chamber has longitudinally extending external wall portions including means for connecting and adjacent capsule.

40. The structure of claim 36 wherein said pressure chamber portion has means spaced from said primer for rearwardly exhausting operatively developed pressure gas therefrom.

41. The structure of claim 40 wherein said propellant chamber has a transverse section of polygonal configuration

42. The structure in accordance with claim 30 wherein an upper surface of said propellant chamber has a pre-formed weakened section.

43. The structure of claim 42 wherein said propellant chamber has a transverse section of polygonal configuration.

 44. The structure of claim 42 wherein said propellant chamber has a transverse section of substantially square configuration, and said pressure chamber portion has means spaced from said primer for rearwardly
 25 exhausting operatively developed pressure gas therefrom.

45. The structure in accordance with claim 42 wherein each propellant chamber has longitudinally extending external wall portions including means for con-30 necting an adjacent capsule.

46. The structure of claim 42 wherein said pressure chamber portion has means spaced from said primer for rearwardly exhausting operatively developed pressure gas therefrom.

47. The structure of claim 46 wherein said propellant chamber has a transverse section of polygonal configuration.

48. The structure in accordance with claim 42 wherein said propellant chamber has a transverse sec-40 tion of substantially rectangular configuration.

49. The structure of claim **48** wherein said transverse section is of substantially square configuration.

50. The structure of claim 42 wherein said propellant chamber has a transverse section of substantially triangular configuration.

51. The structure in accordance with claim 1 wherein said pressure chamber portion has an open rearward end.

52. The structure of claim 51 wherein said propellant chamber has a transverse section of polygonal configuration.

53. The structure in accordance with claim 51 wherein said propellant chamber has longitudinally extending external wall portions including means for connecting an adjacent capsule.

54. The structure in accordance with claim 51 wherein said propellant chamber has a transverse section of substantially rectangular configuration.

60 55. The structure of claim 54 wherein said transverse section is of substantially square configuration.

56. The structure of claim 51 wherein said propellant chamber has a transverse section of substantially triangular configuration.

57. The structure in accordance with claim 51 wherein said propellant capsule chamber has opposed longitudinally extending external wall portions, one of said wall portions having a longitudinally extending

groove and the other wall portion having a tongue protuberance of a substantially similar configuration as said groove

58. The structure of claim **57** wherein said propellant chamber has a transverse section of polygonal configure 5 ration.

59. The structure of claim **57** wherein said propellant chamber has a transverse section of substantially rectangular configuration.

60. The structure of claim 59 wherein said transverse 10 tion of substantially rectangular configuration. 81. The structure of claim 80 wherein said tra

61. The structure of claim **57** wherein said propellant chamber has a transverse section of substantially triangular configuration.

62. The structure in accordance with claim **51** 15 wherein an upper surface of said propellant chamber has a pre-formed weakened section.

63. The structure in accordance with claim **62** wherein said propellant chamber has a transverse section of substantially rectangular configuration

64. The structure of claim **63** wherein said transverse section is of substantially square configuration.

65. The structure of claim **62** wherein said propellant chamber has a transverse section of substantially triangular configuration.

66. The structure of claim 62 wherein said propellant chamber has a transverse section of polygonal configuration.

67. The structure of claim **62** wherein each propellant chamber has longitudinally extending external walf ³⁰ portions including means for connecting an adjacent capsule.

68. The structure of claim 1 wherein said propellant chamber has longitudinally extending external wall portions including means for connecting an adjacent ³⁵ capsule.

69. The structure in accordance with claim **68** wherein said connecting means is partially defined by a longitudinally extending groove in one of said external wall portions, said groove extending to the rearward 40 edge of said capsule.

70. The structure of claim **69** wherein a rearward wall of said pressure chamber portion has a pre-formed weakened section for use with a recoil vent in breech means of a recoilless system

71. The structure of claim 52 wherein inwardly protruding dimple means are located on the internal surface of said pressure chamber portion.

72. The structure of claim 71 wherein said propellant chamber has a transverse section of polygonal configuration

73. The structure in accordance with claim 71 wherein each propellant chamber has longitudinally extending external wall portions including means for connecting an adjacent capsule.

74. The structure of claim 73 wherein inwardly directed flange means are provided on the forward end of said barrel portion.

75. The structure of claim 74 wherein said propellant chamber has a transverse section of polygonal configuration

76. The structure of claim 74 wherein each propel lant chamber has longitudinally extending external wall portions including means for connecting an adjacent $_{65}$ capsule

77. The structure of claim 74 wherein said elongated chamber has a longitudinally slotted wall portion.

78. The structure of claim **77** wherein said propellant chamber has a transverse section of polygonal configuration

79. The structure of claim **77** wherein said propellant chamber has longitudinally extending external wall portions including means for connecting an adjacent capsule.

80. The structure in accordance with claim **77** wherein said propellant chamber has a transverse sec-

81. The structure of claim 80 wherein said transverse section is of substantially square configuration.

82. The structure of claim 77 wherein said propellant chamber has a transverse section of substantially triangular configuration.

83. The structure of claim 71 wherein said elongated chamber has a longitudinally slotted wall portion.

84. The structure of claim 83 wherein said propellant chamber has a transverse section of polygonal configu-20 ration

85. The structure of claim **83** wherein each propellant chamber has longitudinally extending external wall portions including means for connecting an adjacent 25 capsule.

86. The structure in accordance with claim 83 wherein said propellant capsule chamber has opposed longitudinally extending external wall portions, one of said wall portions having a longitudinally extending

) groove and the other wall portion having a tongue protuberance of a substantially similar configuration as said groove.

87. The structure in accordance with claim 86 wherein said propellant chamber has a transverse sec-5 tion of polygonal configuration.

88. The structure of claim **86** wherein and propellant chamber has a transverse section of substantially rectangular configuration

89. The structure of claim **88** wherein said transverse 0 section is of substantially square configuration.

90. The structure of claim **86** wherein said propellant chamber has a transverse section of substantially triangular configuration.

91. The structure in accordance with claim 8045 wherein an upper surface of said propellant chamber bas a pre-formed weakened section

92. The structure of claim **91** wherein said propellant chamber has a transverse section of polygonal configuration.

50 93. The structure in accordance with claim 91 wherein said propellant chamber has a transverse section of substantially rectangular configuration.

94. The structure of claim **93** wherein said transverse section is of substantially square configuration

55 95. The structure of claim 1 wherein said pressure chamber portion has means spaced from said primer for rearwardly exhausting operatively developed pressure gas therefrom

60 96. The structure in accordance with claim 83 wherein an upper surface of said propellant chamber has a pre-formed weakened section, and said propel lant chamber has a transverse section of polygonal configuration.

5 97. The structure of claim 96 wherein said propellant capsule chamber has opposed tongitudinally extending external wall portions, one of said wall portions having a longitudinally extending groove and the other wall portion having a tongue protuberance of a substantially similar configuration as said groove.

98. The structure of claim **96** wherein wall portions of said elongated chamber and propellant capsule chamber are connected by a transversely extending 5 wall, said chamber wall portions being transversely spaced from each other and having transversely aligned operative port means defining said metering orifice means.

99. The structure in accordance with claim 96 $_{10}$ figuration, wherein said propellant chamber has a transverse sec-

tion of substantially rectangular configuration.

100. The structure of claim 99 wherein said transverse section is of substantially square configuration.

101. The structure of claim 96 wherein said propellant chamber has a transverse section of substantially triangular configuration.

102. The structure of claim **95** wherein said propellant chamber has a transverse section of polygonal configuration.

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APPENDIX B

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Representative Firing Data

APPENDIX B

Representative Firing Data

| Rd <u>No</u> . | Weapon | Muzzle Velocity <u>(fps)</u> | Chamber Pressure (KPSI) | Bolt Cavity Pressure (KPSI) |
|-------------------|-------------|------------------------------------|--------------------------------|-----------------------------------|
| 391 | Belgian LAR | 3007 | _ | N |
| 392 | (Modified) | 2966 | 45.0 | 11 |
| 393 | | 3226 | 44.5 | *1 |
| 394 | | 3262 | 50.0 | 11 |
| 395 | | 3134 | 43.0 | 11 |
| 396 | | 3028 | 39.5 | 11 |
| 3 9 7 | | 3222 | 43.5 | F1 |
| | | $\bar{v} = 3121$ | $\overline{\mathbf{P}}$ = 44.2 | |
| 3 9 8 | Belgian LAR | 29 72 | 44.5 | N |
| 399 | (Modified) | 2874 | 48.0 | 11 |
| 400 | | 3040 | 41.5 | !! |
| 401 | | 3228 | 41.5 | 12 |
| 402 | | 30 37 | 44.5 | 11 |
| 403 | | 3166 | 44.5 | ** |
| 404 | | 3046 | 44.5 | 11 |
| 405 | | 3105 | 43.0 | 11 |
| | | v = 3059 | $\overline{P} = 44.0$ | |
| 406 | M-16 Al | 3218 | N | 16.5 |
| 407 | (Modified) | - | 1 | 15.0 |
| 408 | | 3169 | | 11.0 |
| 409 | | 3249 | 11 | 11.0 |
| 410 | | 3211 | ŦT | 19.5 |
| 411 | | 3323 | 11 | 19.5 |
| 412 | | 3174 | 11 | 20.0 |
| 413 | | 3271 | ti | 19.5 |
| 414 | | 3213 | Ħ | 19.5 |
| | | v = 3228 | | P = 16.8 |

N - Not Measured

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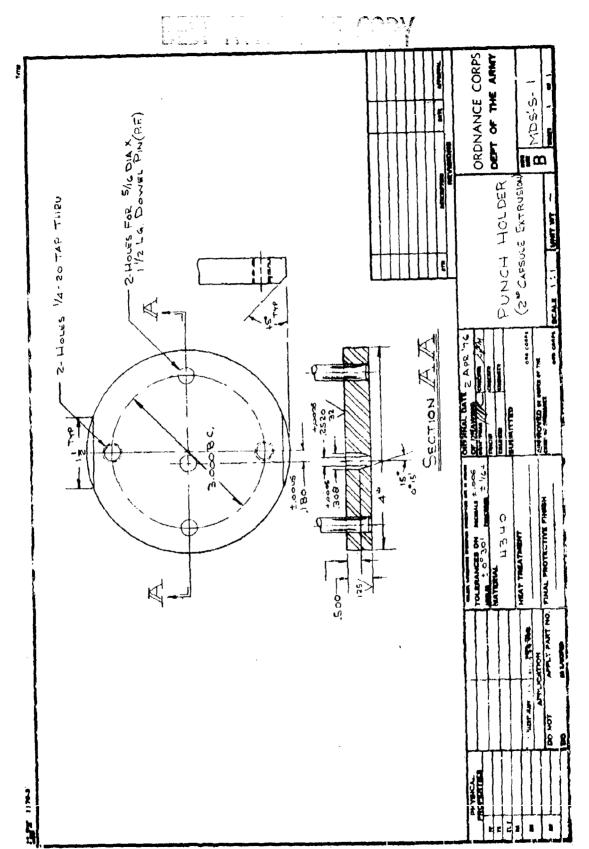
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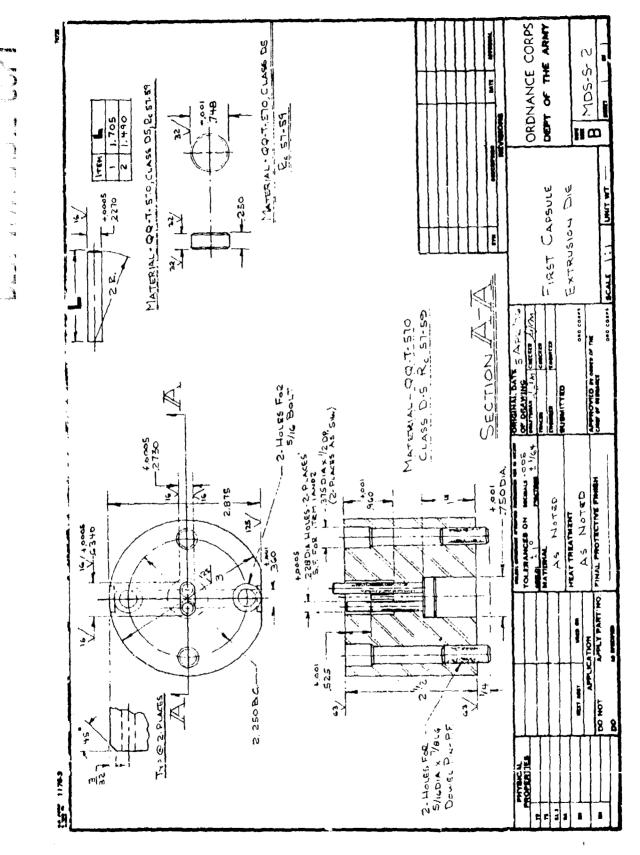
APPENDIX C

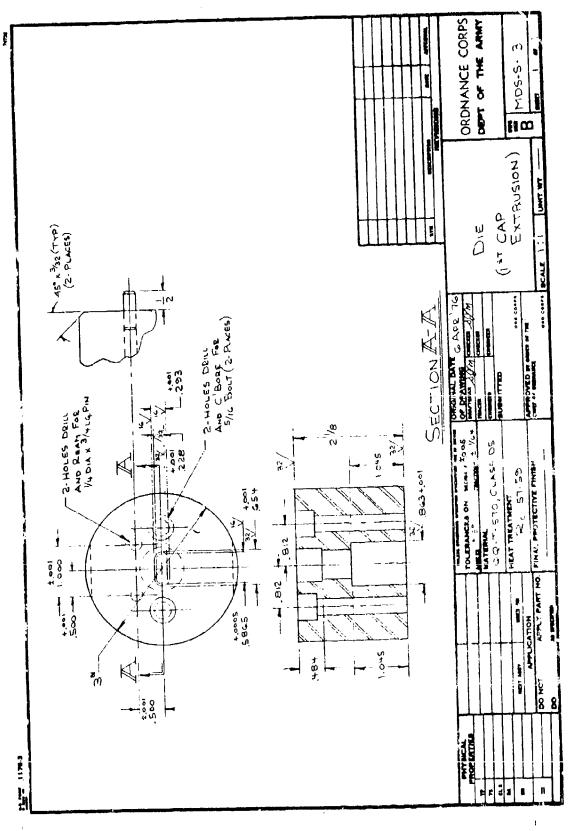
Tool Drawings for Case Extrusion Process

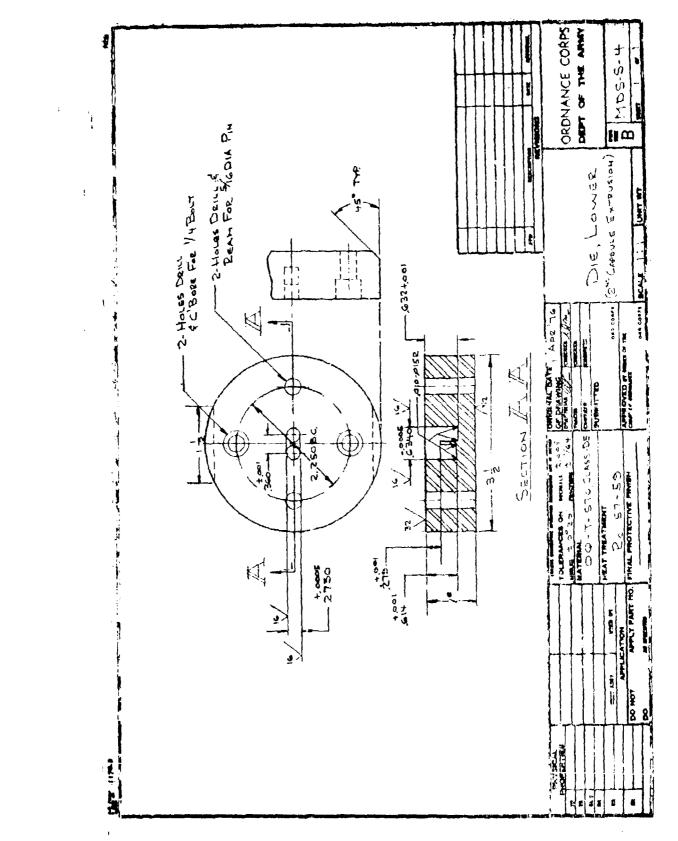


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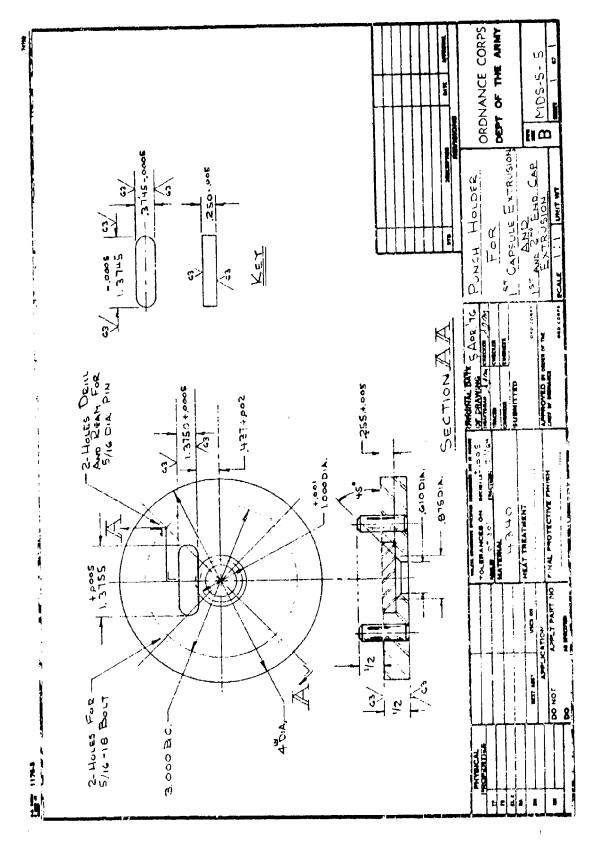


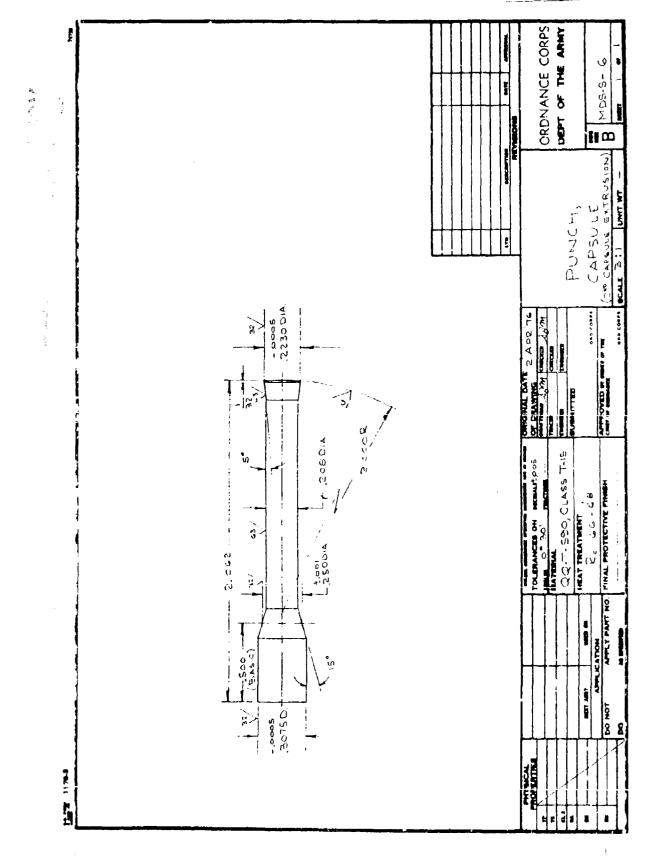




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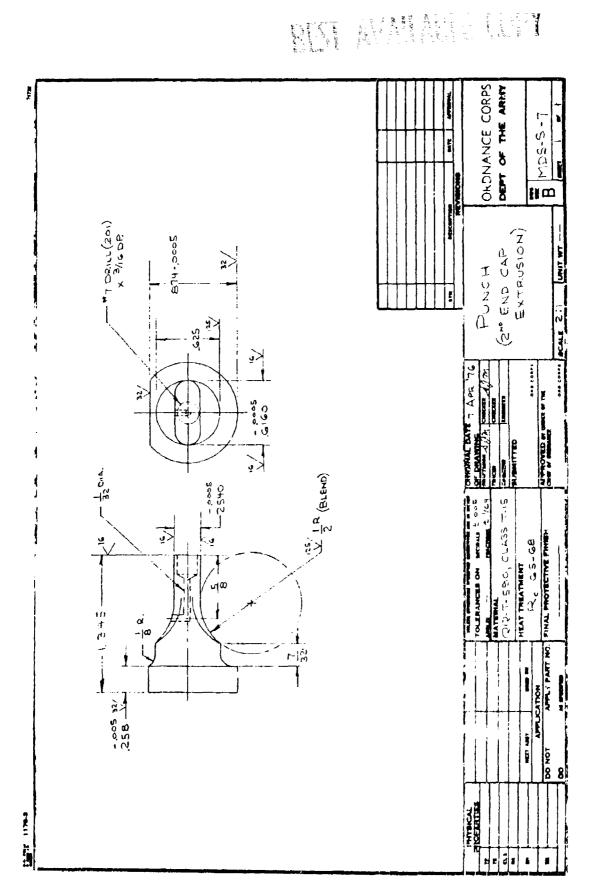


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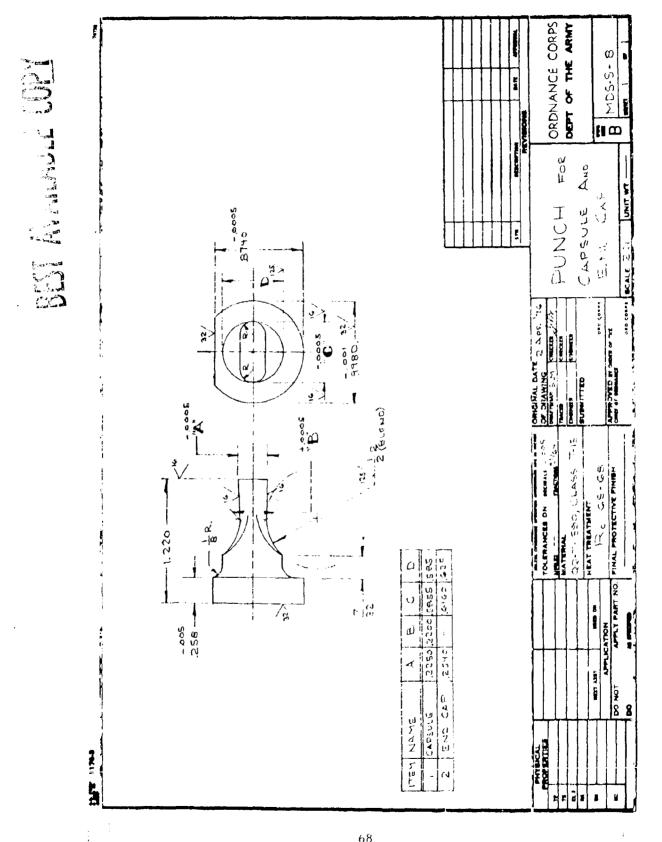
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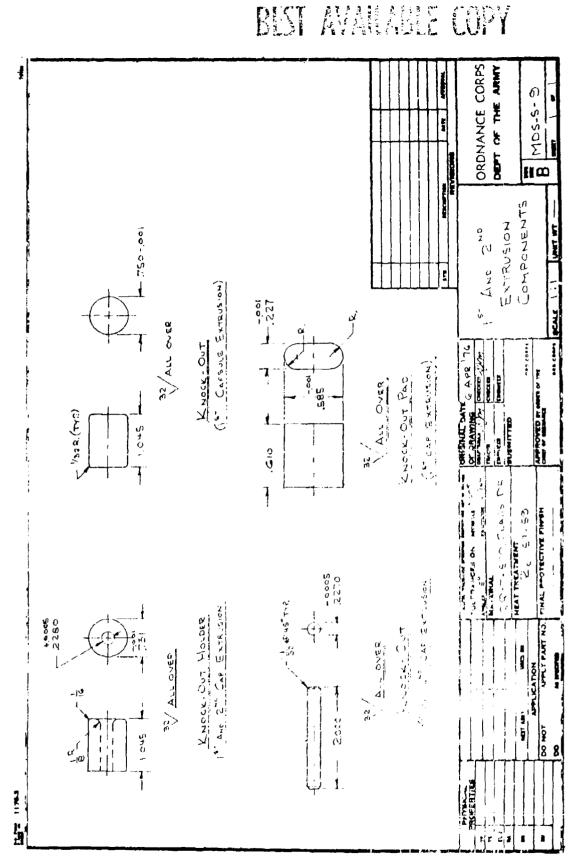
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APPENDIX D

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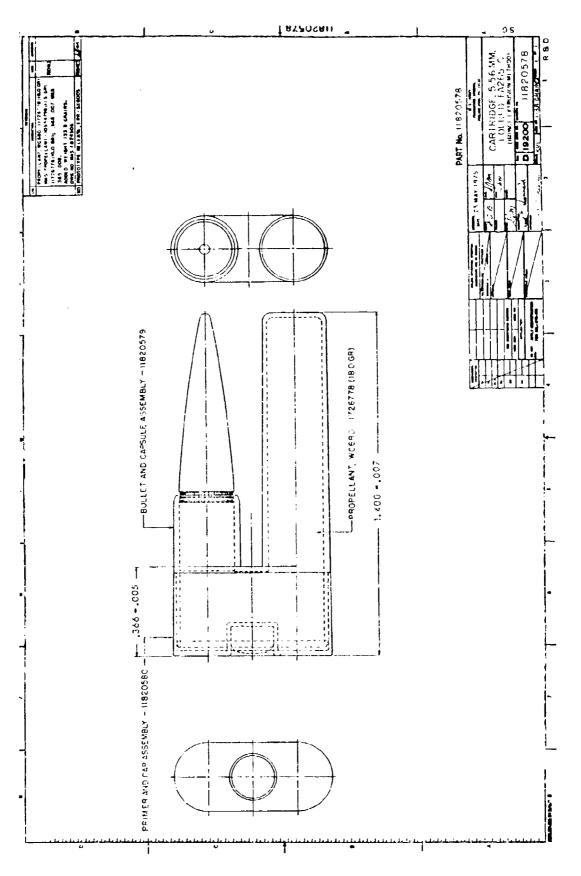
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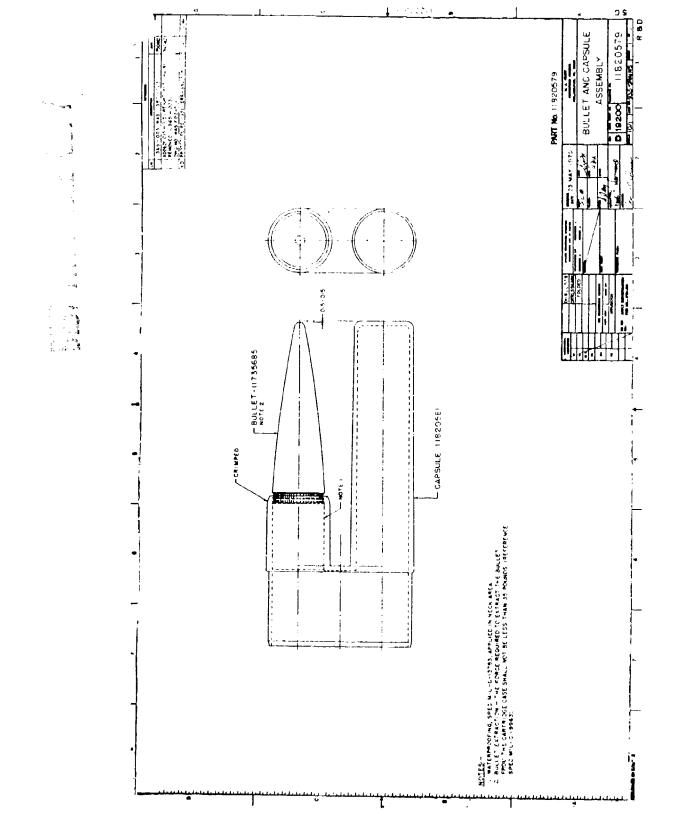
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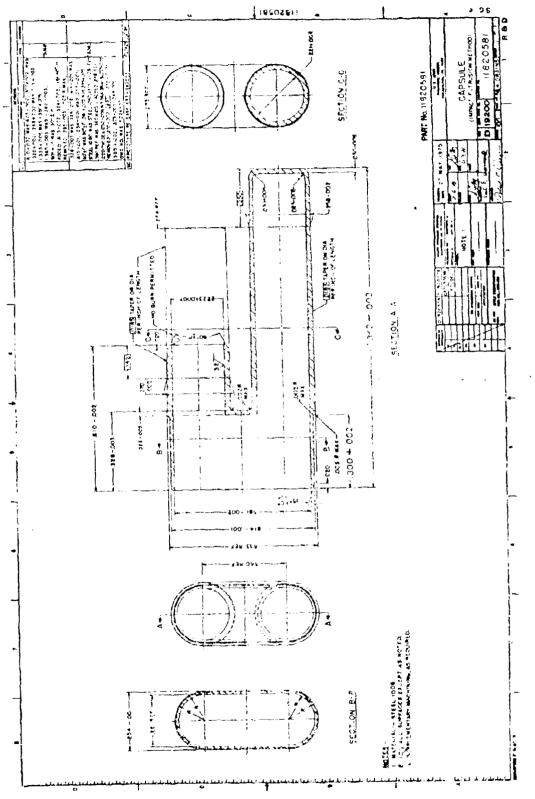




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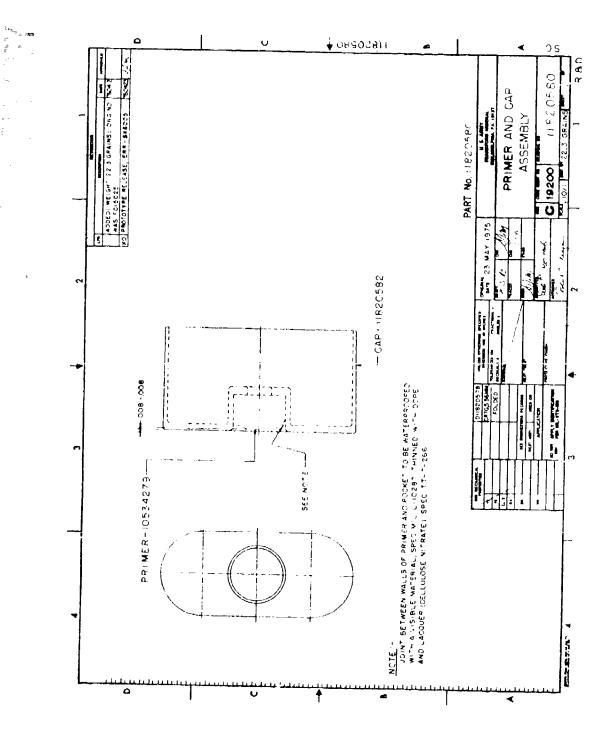
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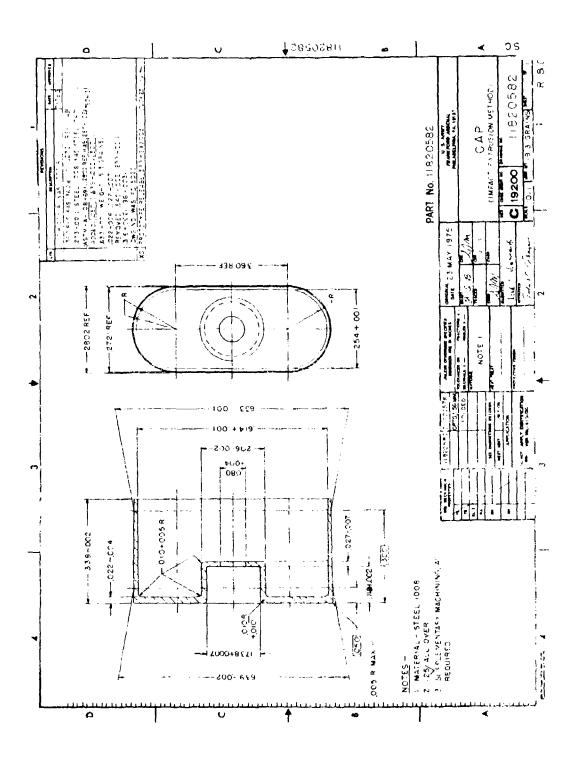
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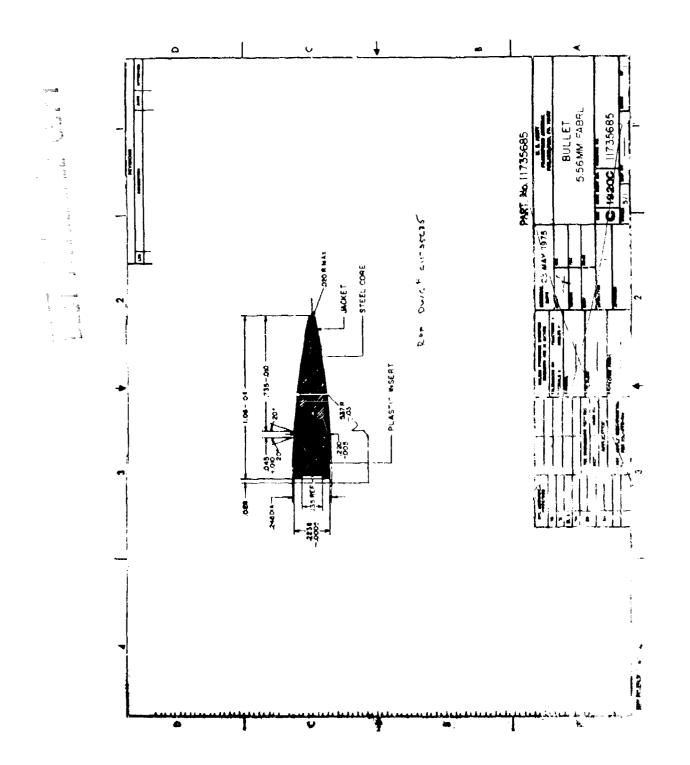


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APPENDIX E

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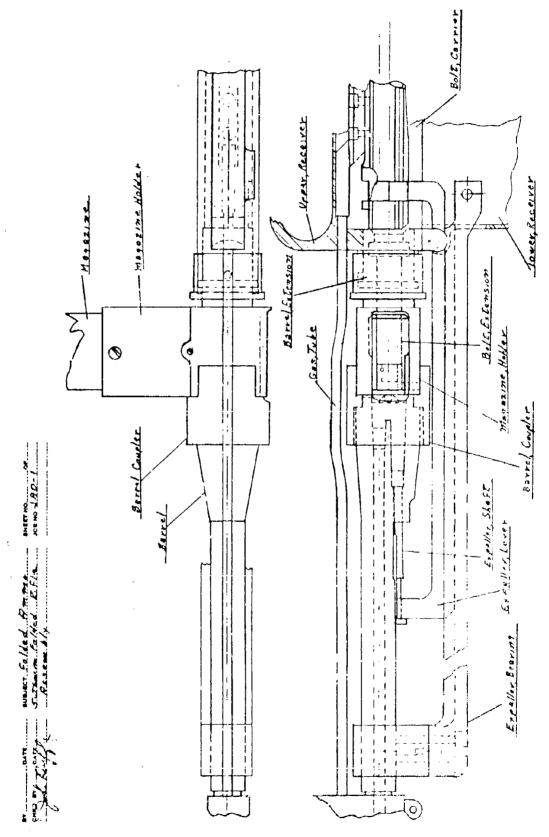
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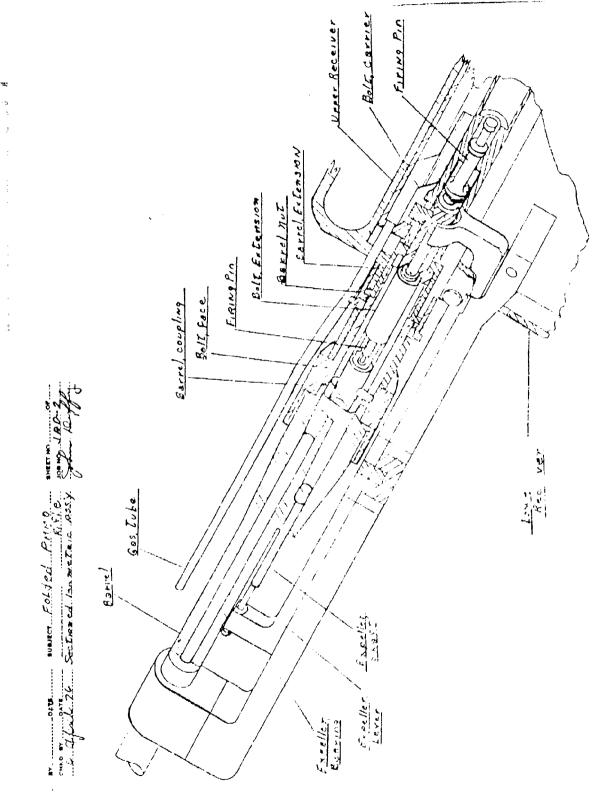
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SKETCH LIST

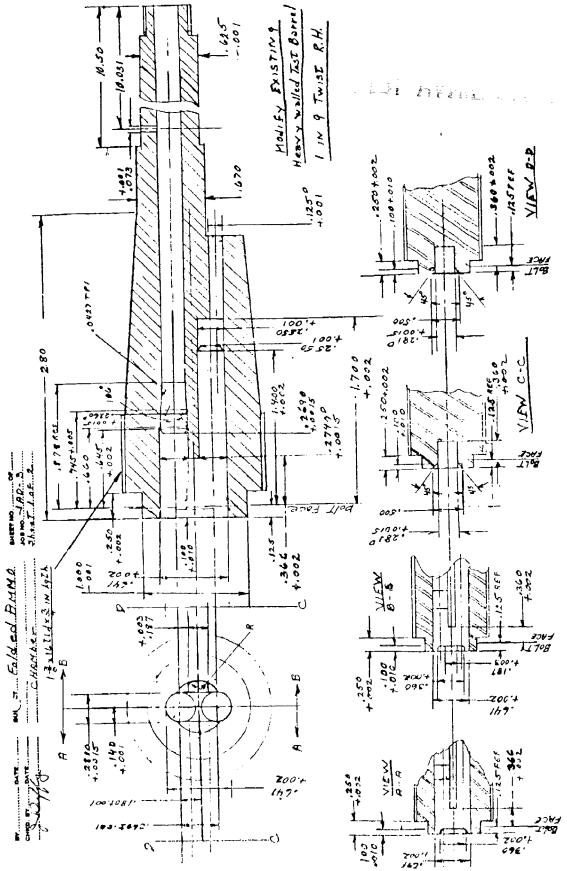
| NO | NOMENCLATURE | SKETCH NO |
|----|--|-----------|
| | | JAD-1 |
| 1 | 5.56mm Folded Rifle, Assembly | JAD-2 |
| 2 | 5.56mm Folded Rifle, Sectioned Isometric Ass'y | JAD-3 |
| 3 | Barrel (2 Sheets) | JAD-4 |
| 4 | Firing Pin | JAD-5 |
| 5 | Bolt Face | JAD-6 |
| 6 | Bolt Extension | JAD-7 |
| 7 | Bolt Carrier Modification | JAD-8 |
| 8 | Upper Receiver Housing Modification | JAD9 |
| 9 | Barrel Coupling | JAD-10 |
| 10 | Buffer | JAD-11 |
| 11 | Expeller Lever | 370-12 |
| 12 | Expeller Shaft | JA')-13 |
| 13 | Expeller Bearing | JAD-14 |
| 14 | Magazine Housing | JAD-15 |
| 15 | Lower Housing Modification | JAD-16 |
| 16 | Sear, Automatic Modification | 71N 17 |
| 17 | Pins, Lower Housing Assembly | - Au - 0 |
| 18 | Magazine, Assembly | 10 |
| 19 | Magazine, Body | JAD-20 |
| 20 | Magazine, Spring | JAD-21 |
| 21 | Magazine, Base | JAD-22 |
| 22 | Magazine, Follower | 1,1-23 |
| 23 | Chamber, Reamers | |

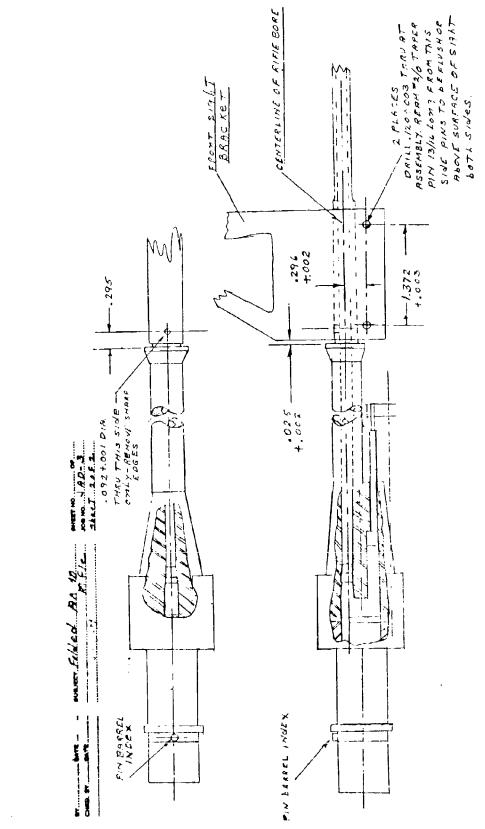
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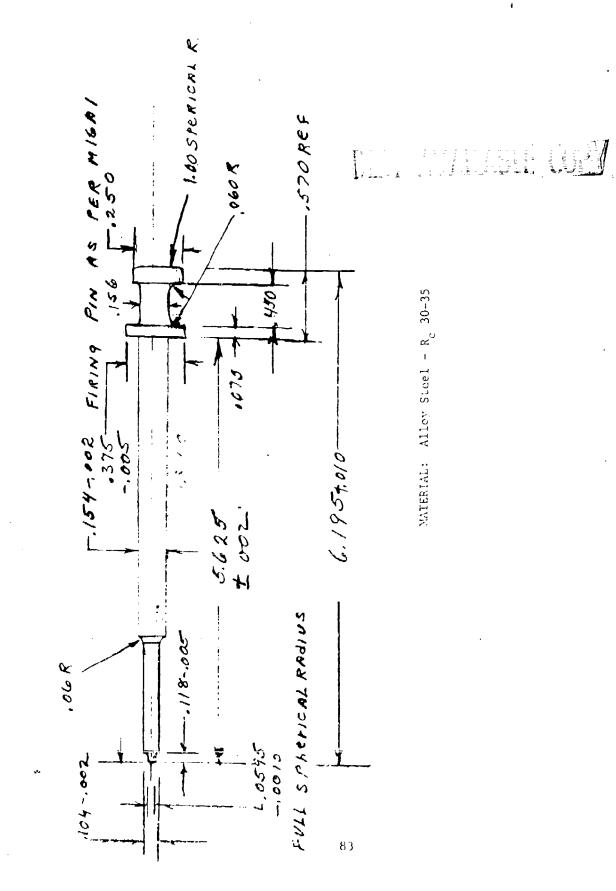
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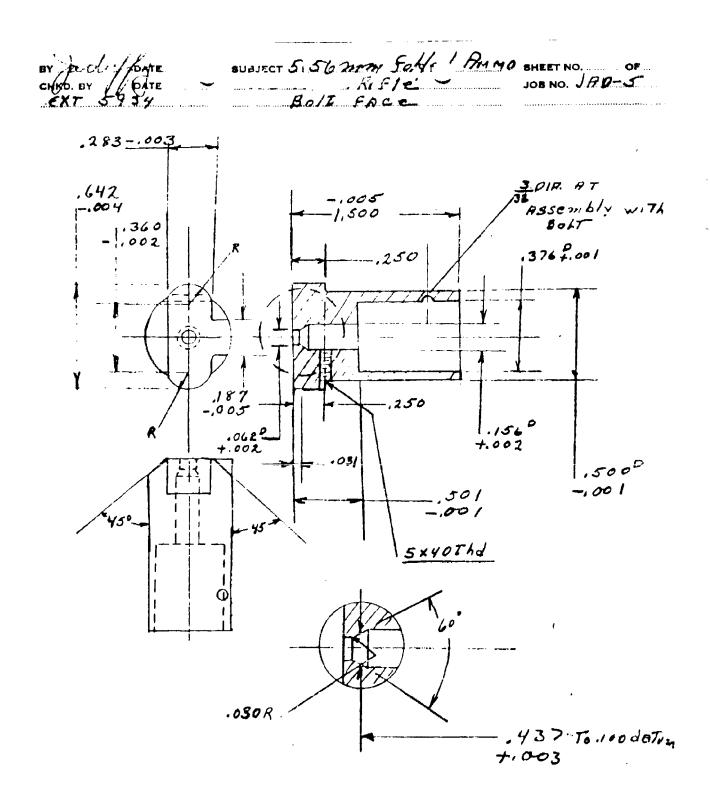
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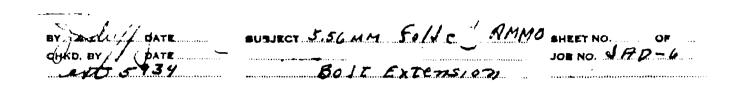
FLRING PIN

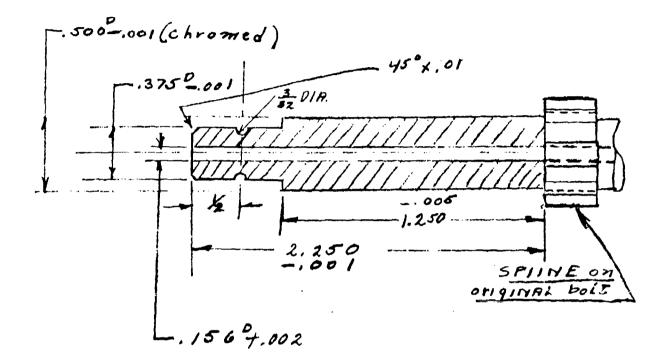
JAD-4





MATERIAL: Alloy Steel, $R_c = 38-42$



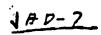


MATERIAL: Alloy Steel, $R_c = 38-42$

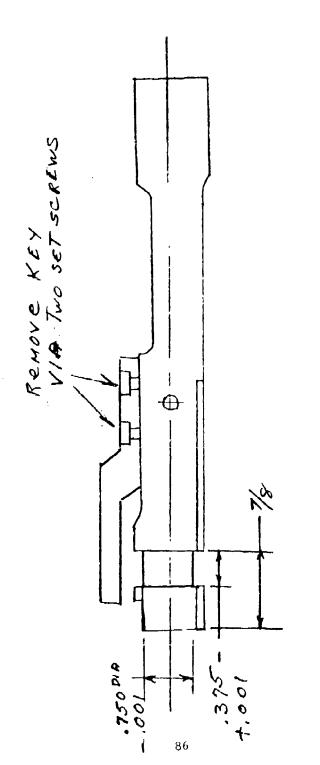
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Folded AMMO BOLT Carrier



Modification - Grind a .375 inch wide groove having an .075 inch diameter.



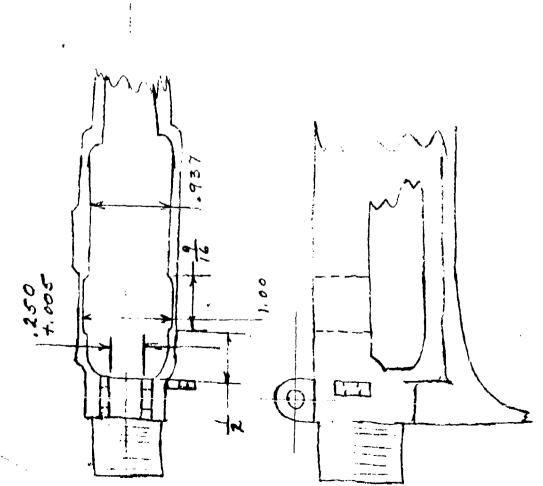


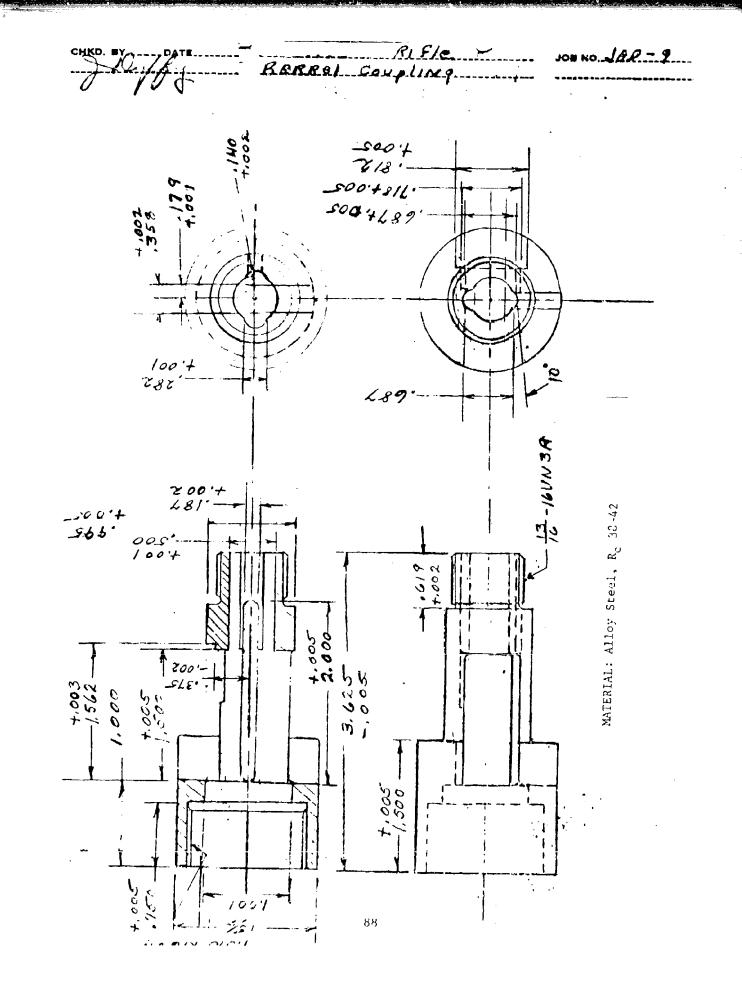
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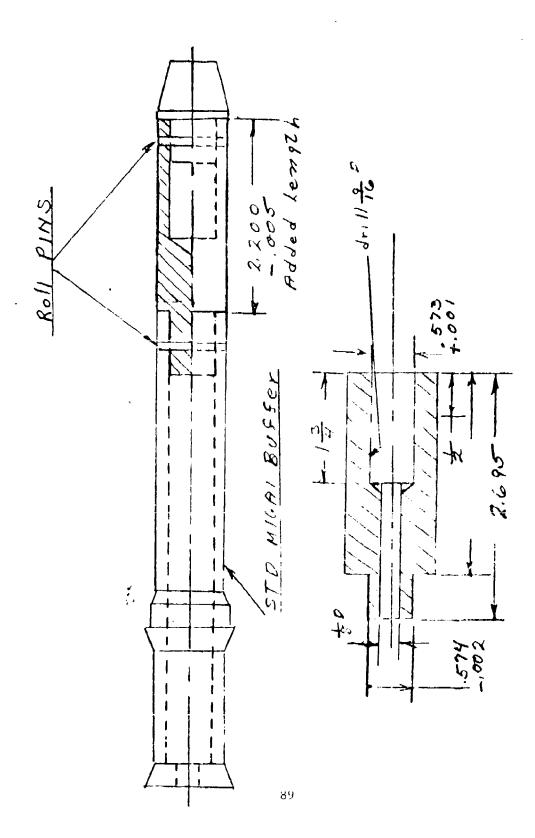
Folded AMMO xiFle UPPER RECEIVER HOUSING

JA0-8



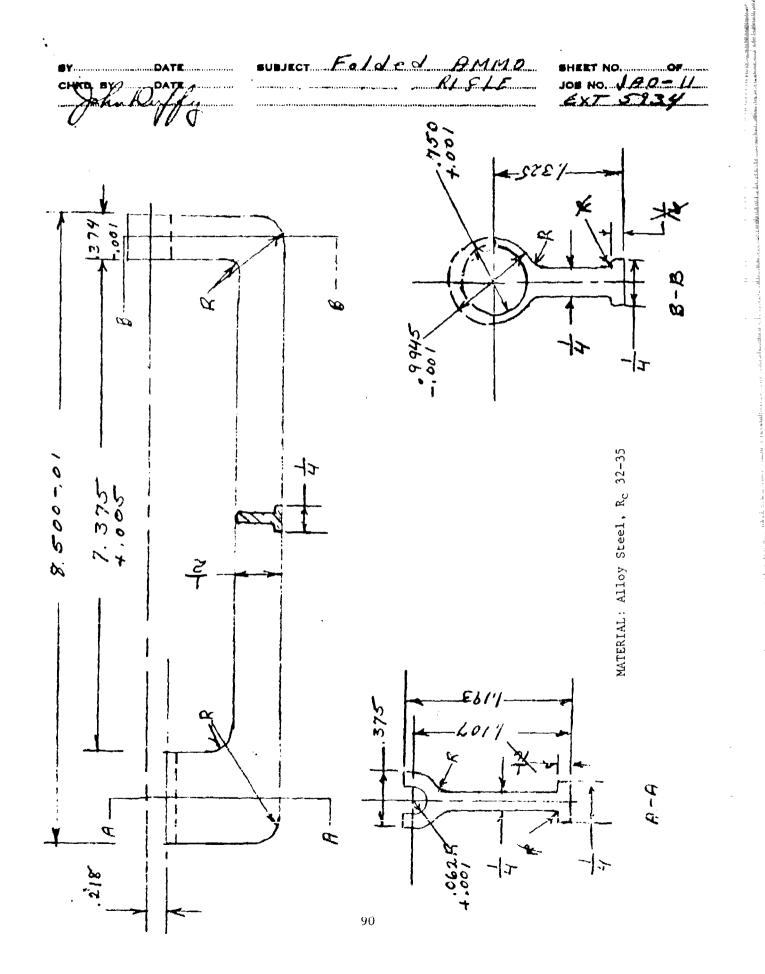


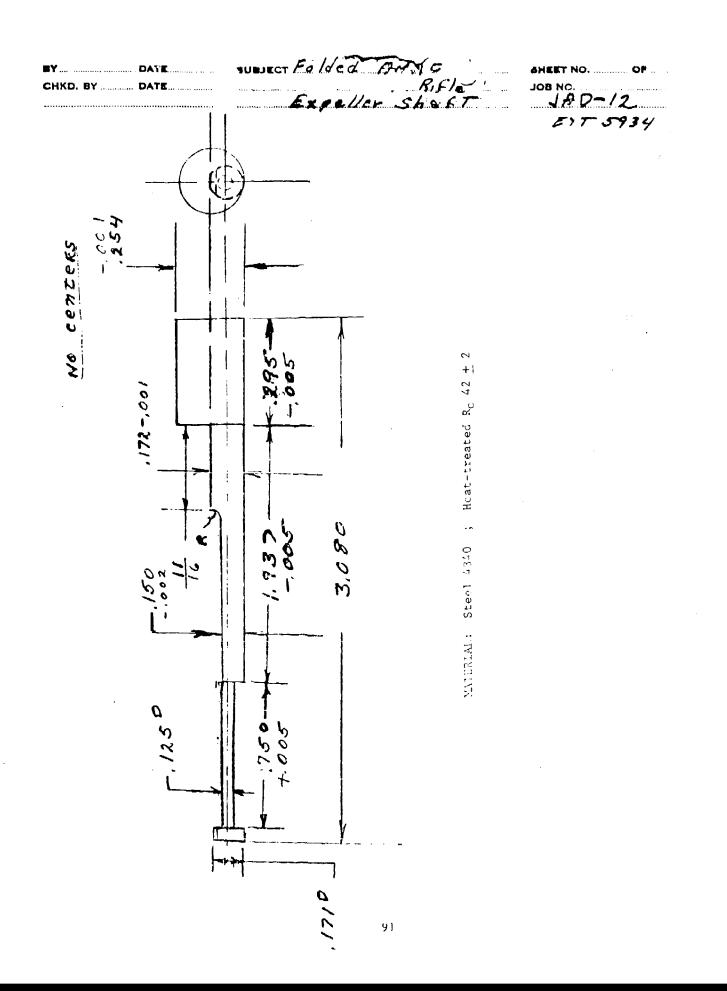
BUFFER SHEET NO. ..**OF**..... BY. JOS NO. VAR-10 СНКО DATE 13 5934 15 10. An

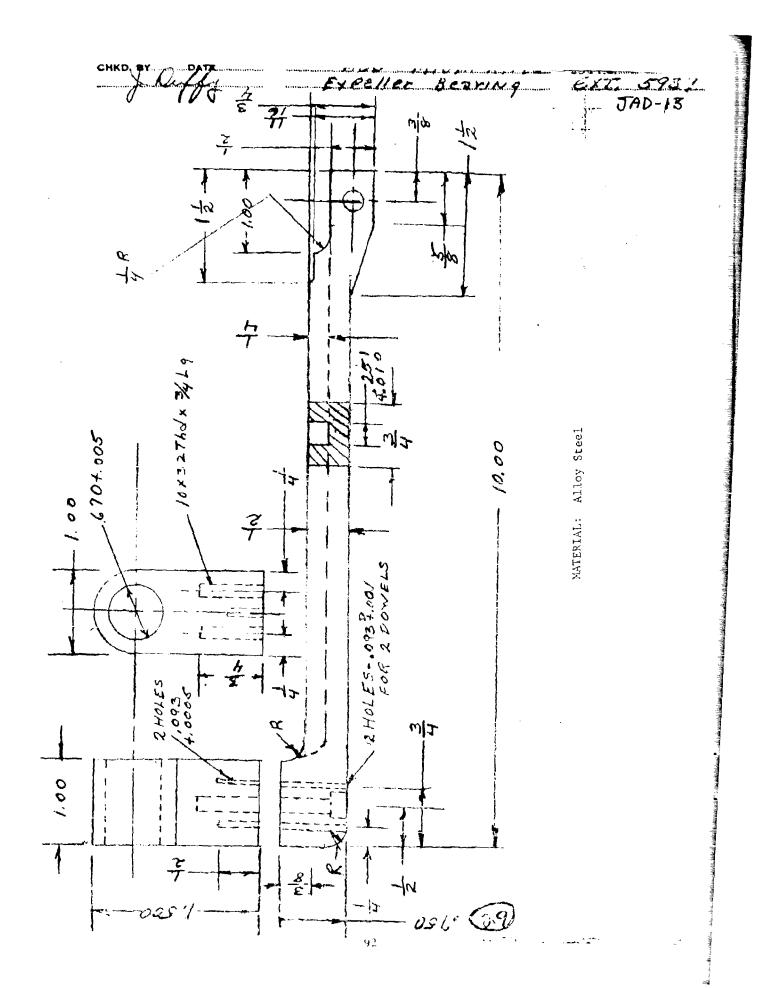


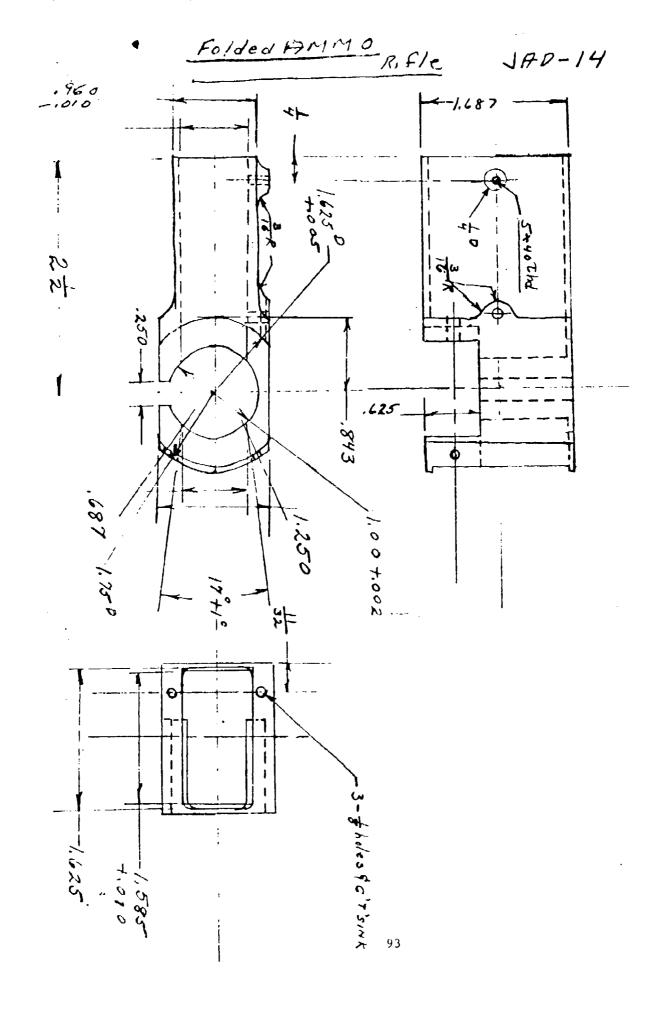
MATERIAL: Steel

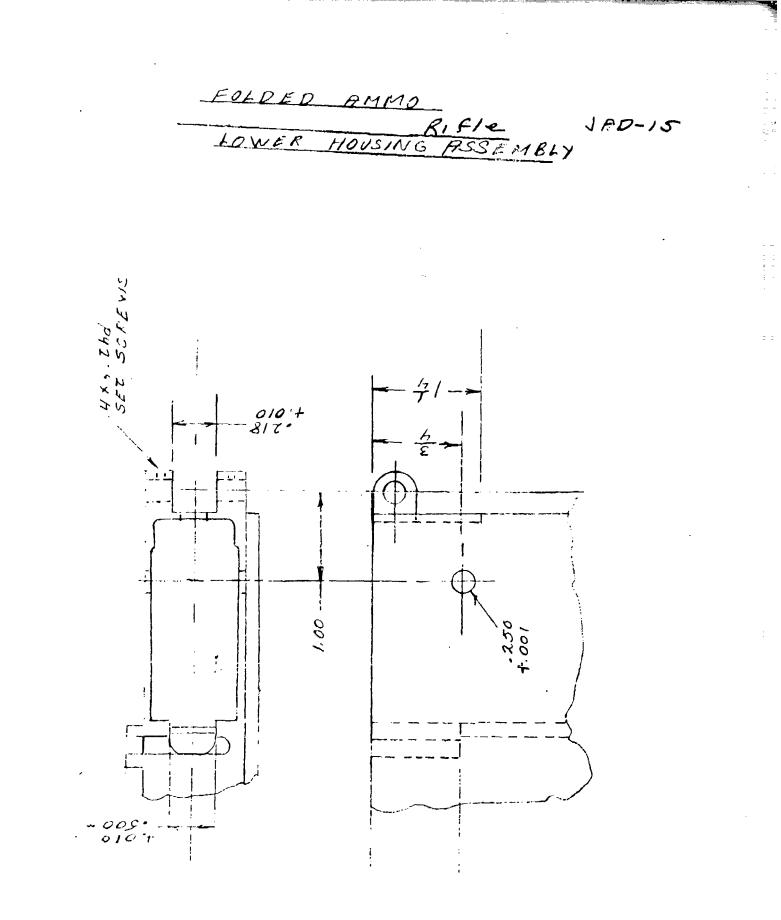
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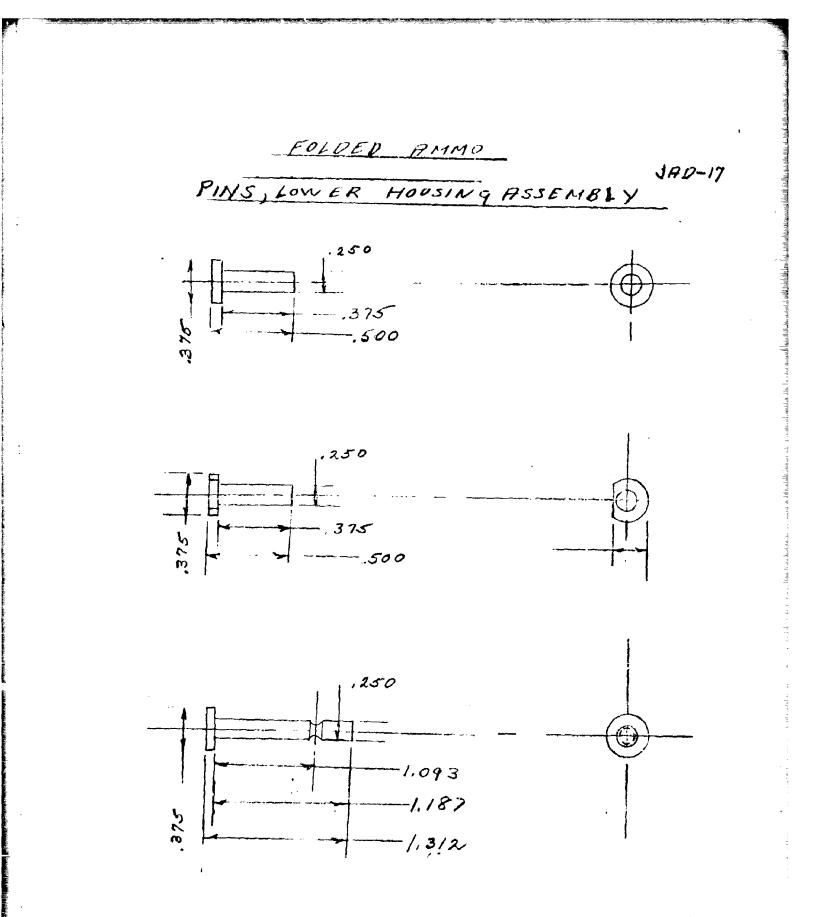








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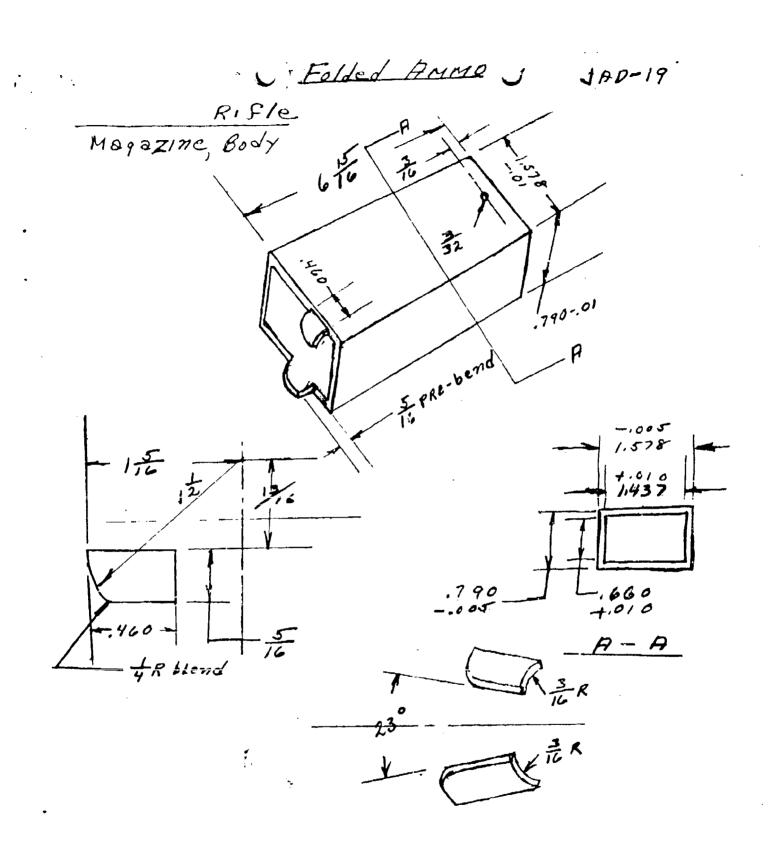
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JAD-21

5. Roll Cin



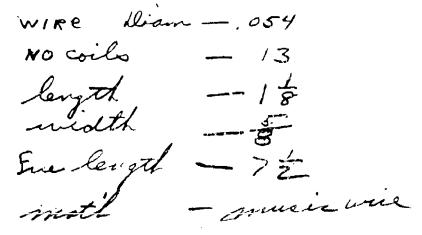
MATERIAL: Steel/brass

John Reitfry

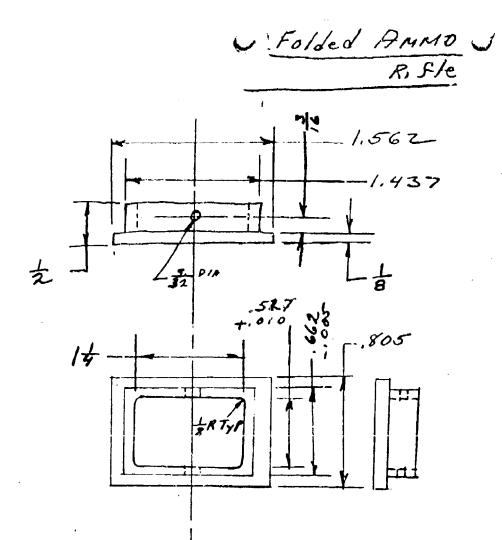
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JAD-20





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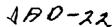


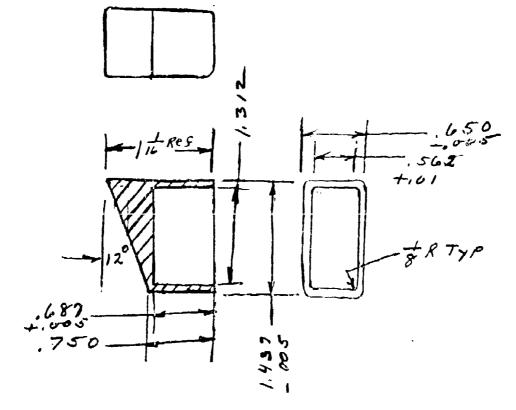
MATERIAL: Aluminum

John Duffy

BASE, MAG221. JAD-21

N. Folded Ammo J. <u>Rifle</u> Follower, Magazime



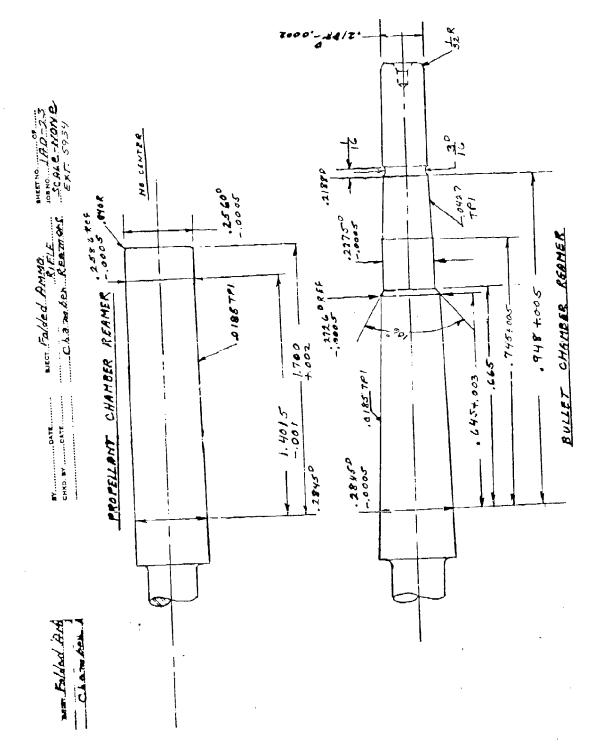


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John Ruff.



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APPENDIX F

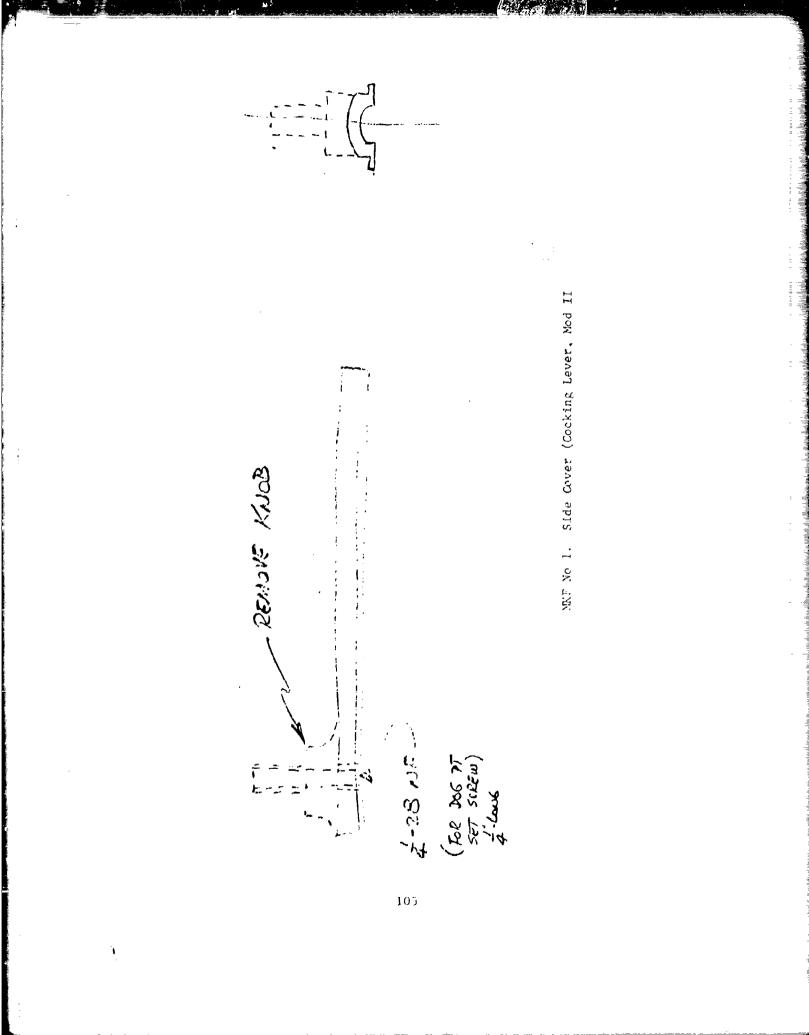
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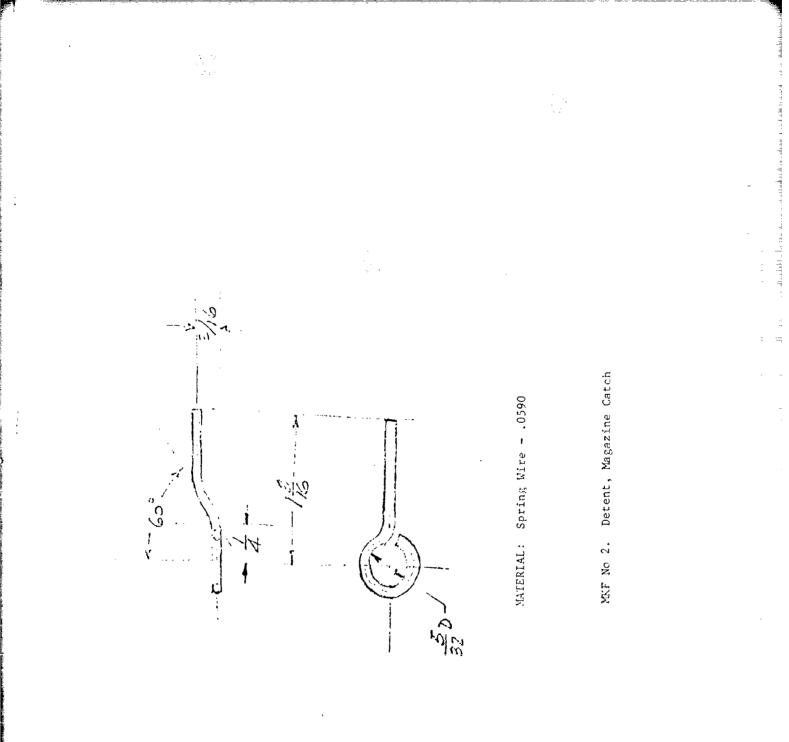
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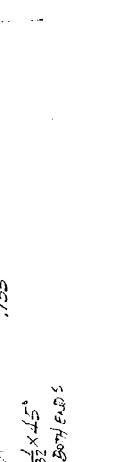
Modified Belgian LAR Details

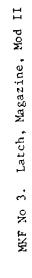
SKETCH LIST

MKF Sketch No. Side Cover (Cocking Lever), Mod II 1 2 Detent, Magazine Catch 3 Latch, Magazine, Mod Il 4 Sub-Assembly, Magazine, Adaptor 5 Back Magazine, Fln-Folded 6 Body Magazine, Flu-Folded 7 End, Magazine 8 Follower, Nod 11 9 Spring, Magazine, Mod Il 10 Tube, Magazine Sub-Assembly, Mod Il 11 Rod, Piston Adapter, Gas Cylinder, Mod 11 12 13 Piston 14 Detail of Groove Cylinder, Gas, Mod II 15 Cylinder, Gas, Mod II Sleeve, Mod JI 16 17 Socket, Gas Tube, Mod 11 18 Spring, Expeller Return 19 Extension, Expelier, Mod 11 20 Expeller, Mod 11 21 Spring, Pin, Bolt 22 Spring, Return Firing Pin 23 Pin, Bolt Support, Mod 11 24 Carrier, Bolt, Mod II 2.5 Firing Pin, Mod J1 26 Front Sight Sub-Assembly, Modified L.A.R. 27 Kicker Handle, Mod 11 28 29 Barrel (Detail), Mod 11 30 Barrel, Mod II 31 Receiver, Kicker Slot Detail, Mod II 32 Pawl, Mod 11 33 Bolt, Mod 11 34 Magazine Adaptor, Fln 5.56 mm Folded 35 Sub-Assembly, Cas System 36 Bolt, Sub-Assembly 37 Buliet, Chamber, Reamer 38 Fore Arm, Mod 11 39 Chamber, Mod 11



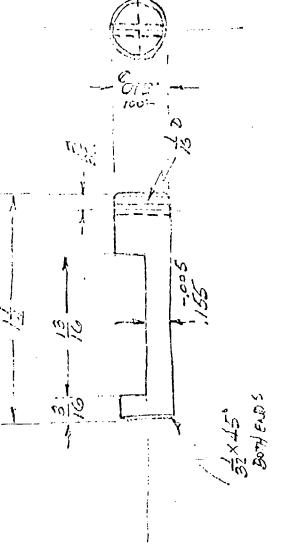






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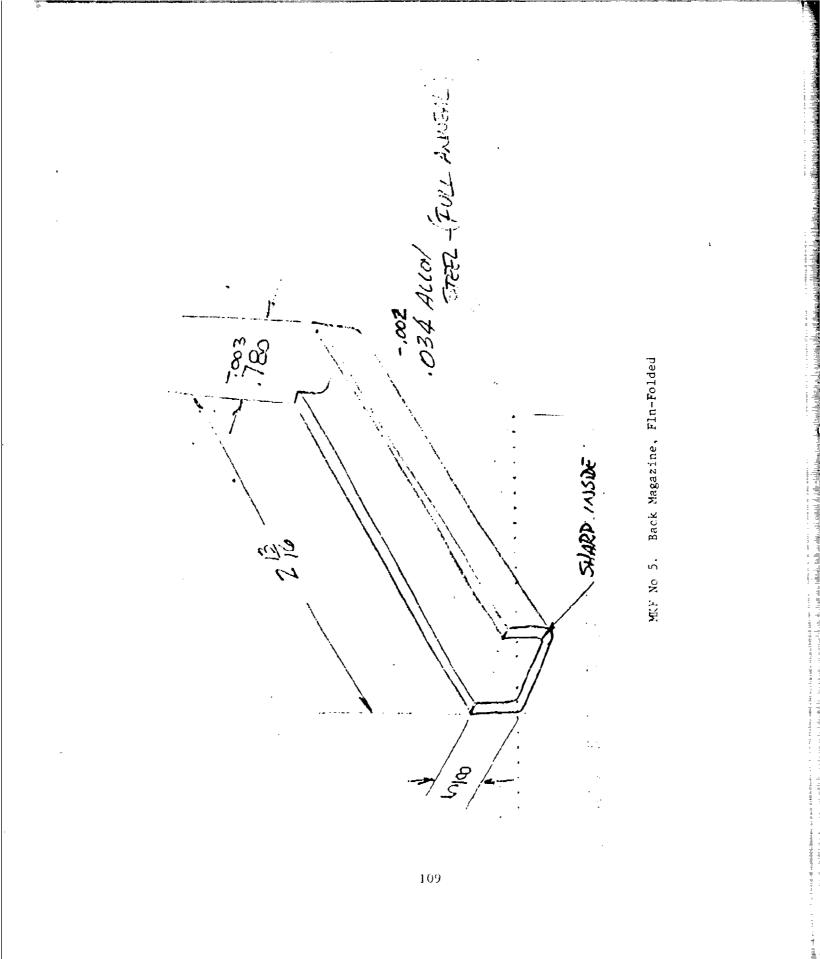
MATEPIAL: Alloy Steel $(R_C 48-50)$

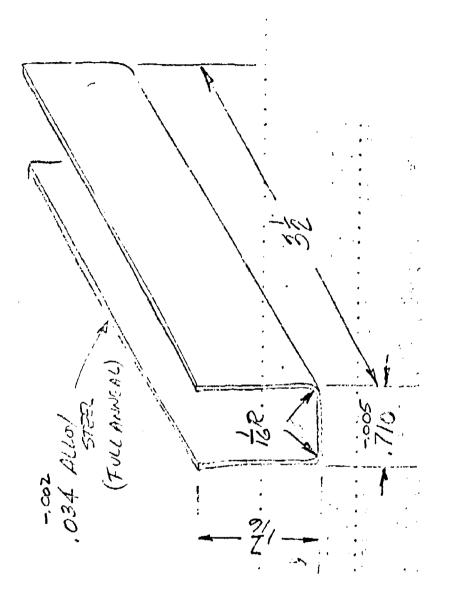


JETENT MAGAZINE LATOH SCREW- 6-52 X 2 LONG ADAPTOR, MAGGZWJE 24104 MAGDENE

MKF No 4. Sub-Assembly, Magazine, Adaptor

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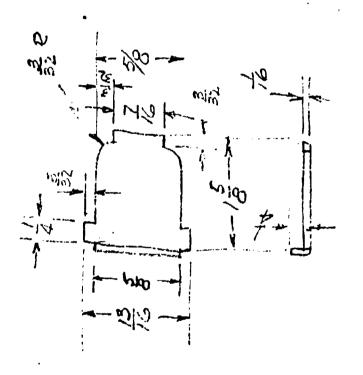
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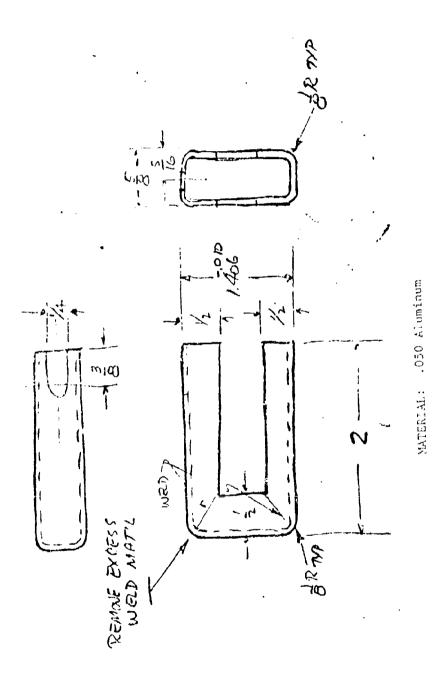
MKF No 6. Body Magazine, Fln-Folded

J 10



MATERIAL: Steel - 1/32" thick

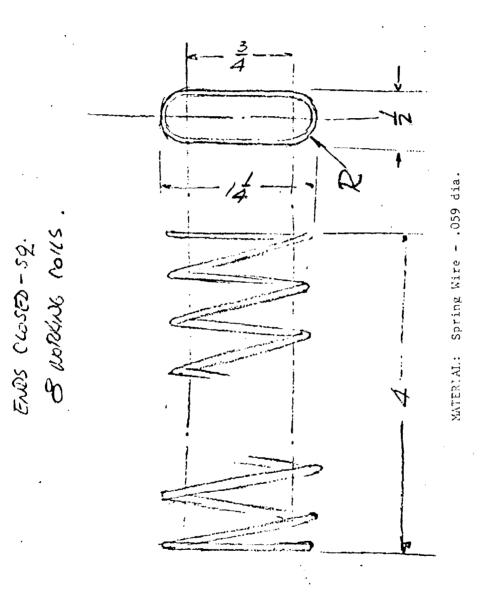
MKF No 7. End, Magazine



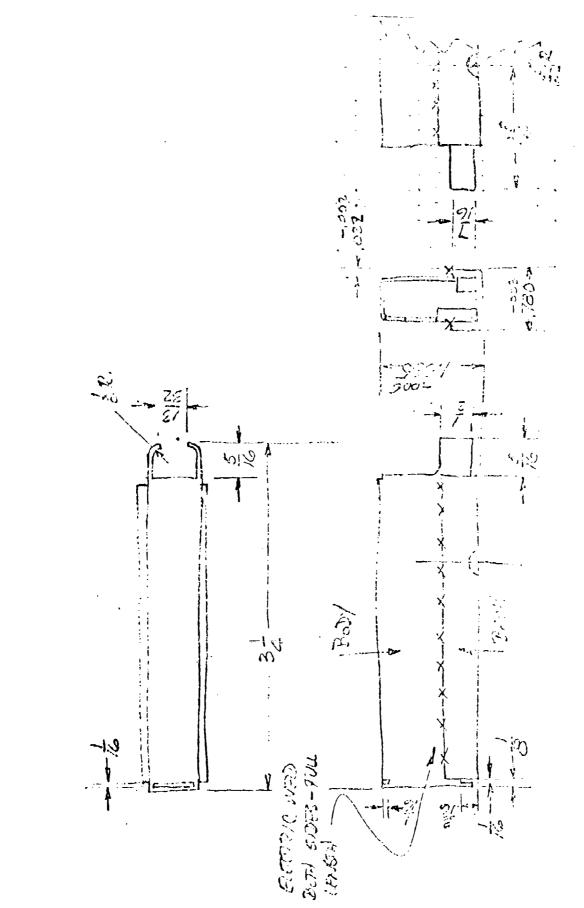
MKF No 8. Follower. Mod II

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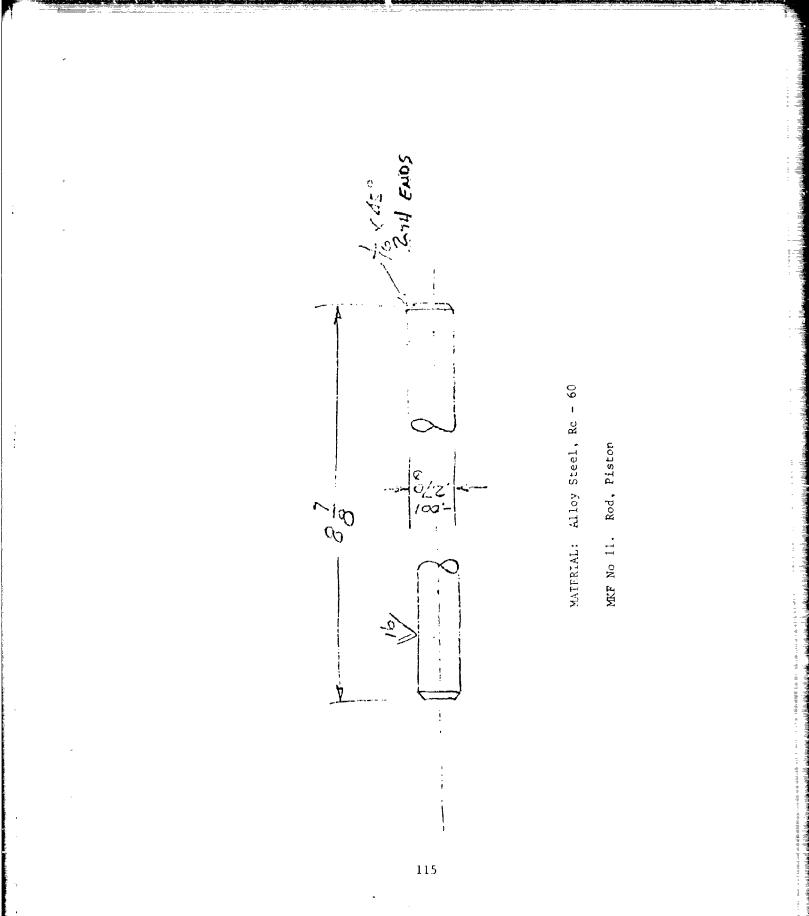
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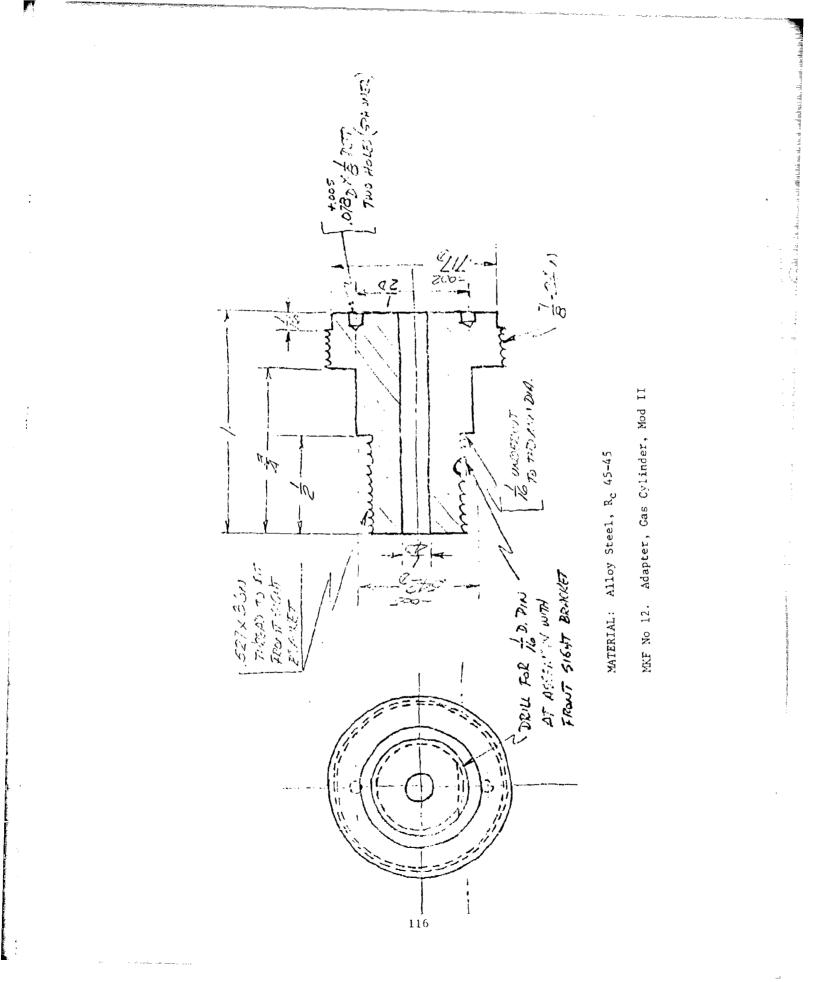


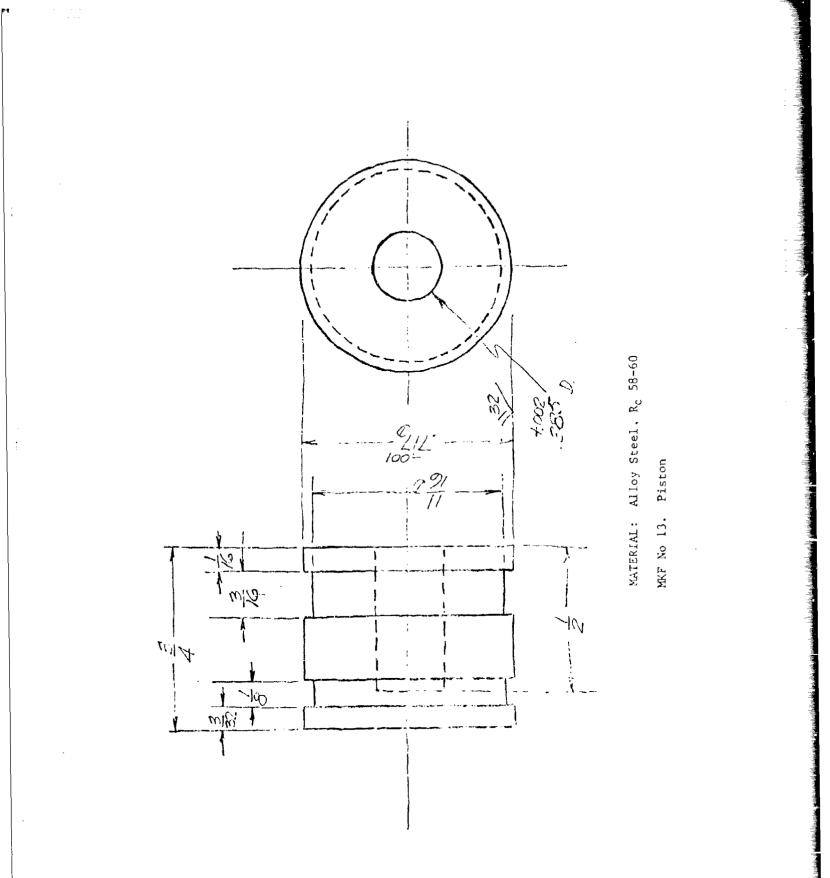
MKF No 9. Spring, Magazine, Mod II

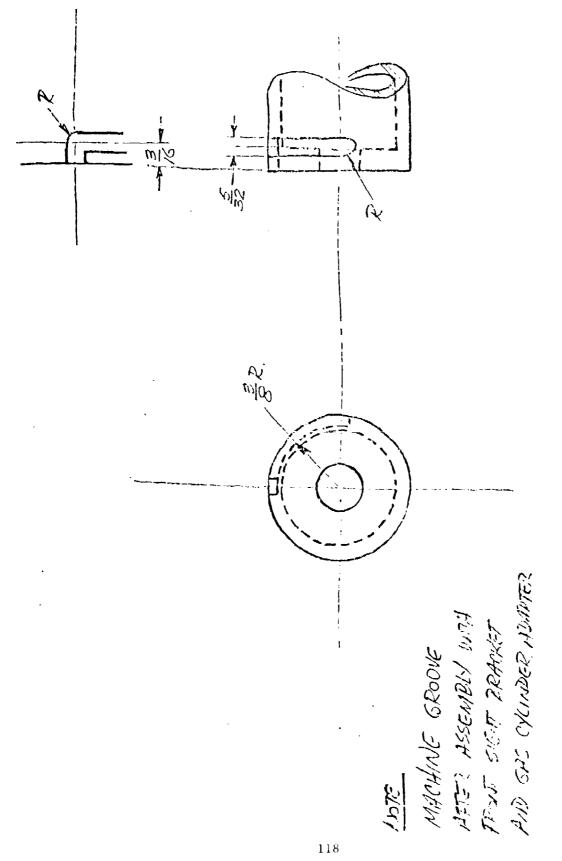


MKF No 10. Tube, Magazine Sub-Assembly, Mod II





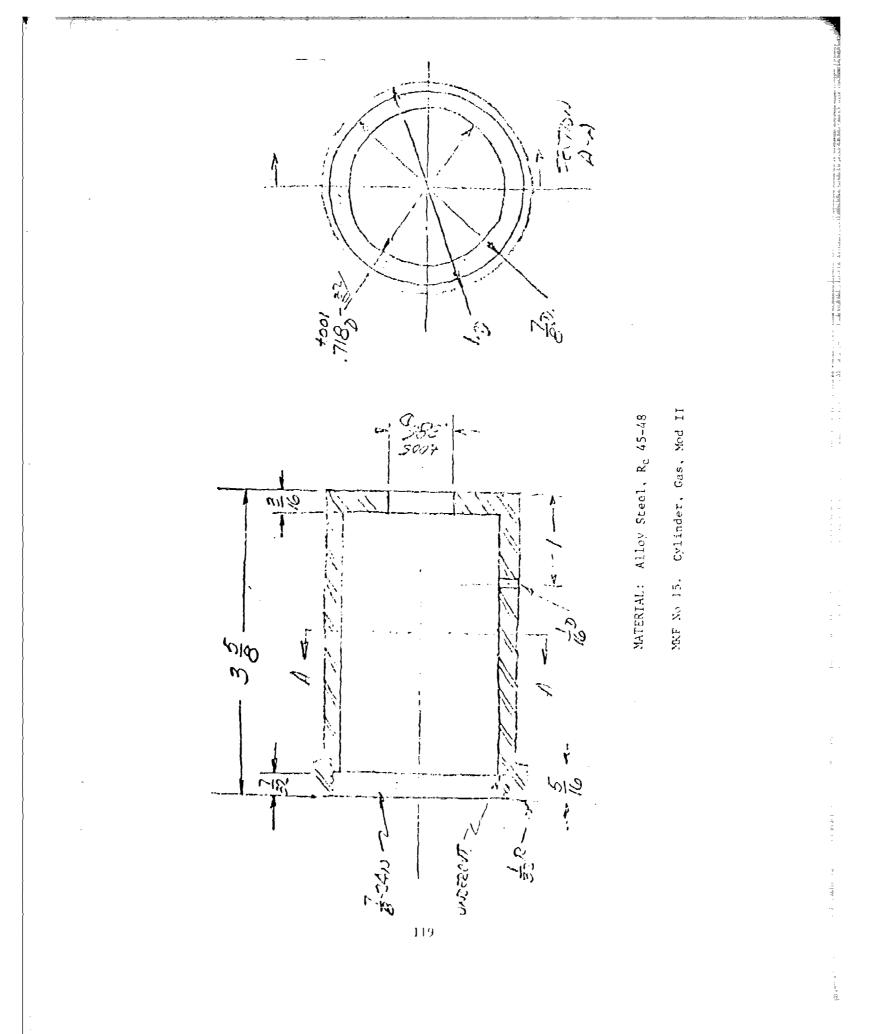


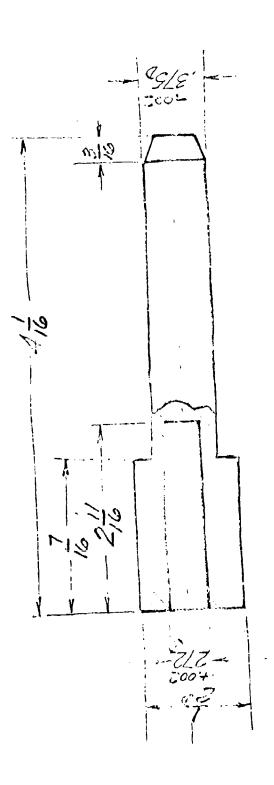


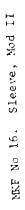


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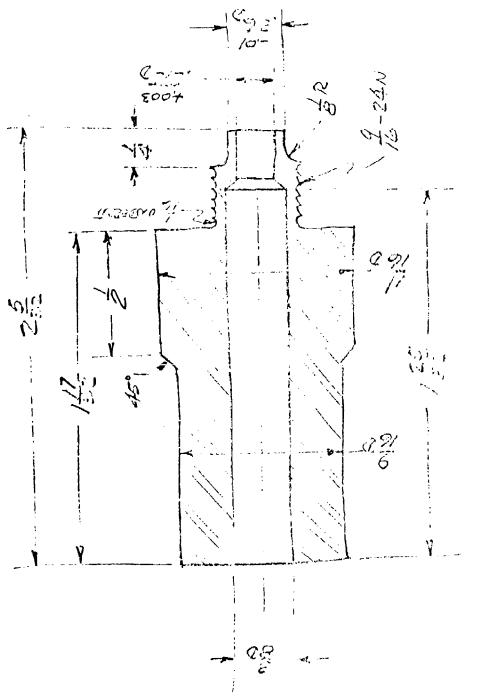
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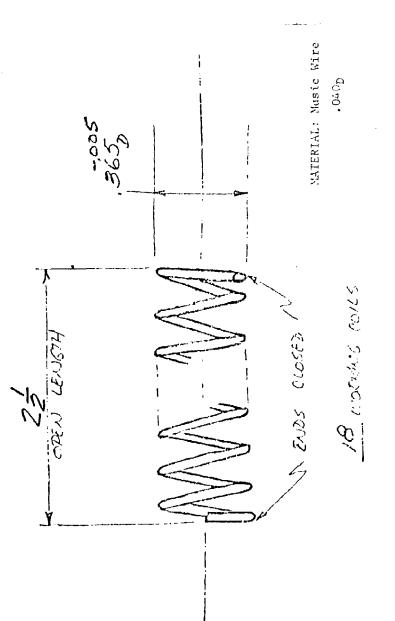




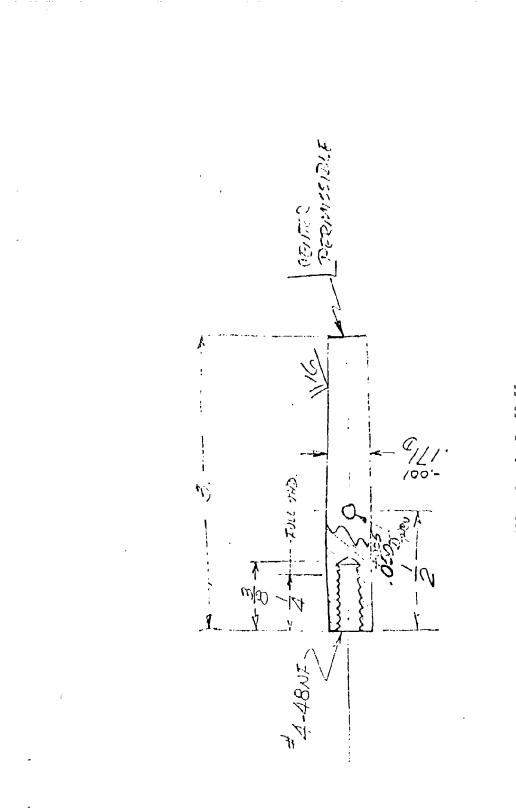
MATERIAL: Alloy Steel, R_c 55-60







MKF No 18. Spring, Expeller Return

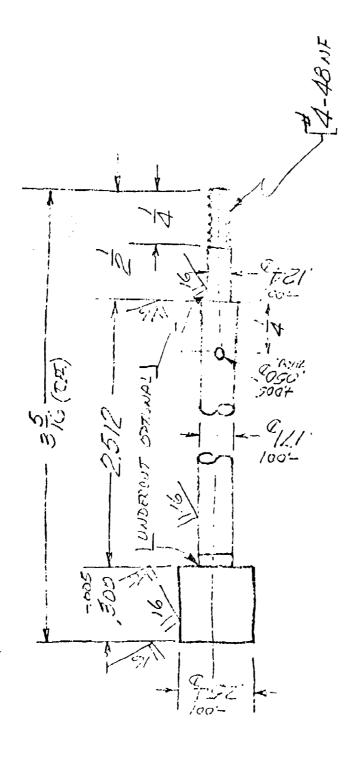


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MATERIAL: Alloy Steel, R_c 53-55

MKF No 19. Extension, Expeller, Mod II



MATERIAL: Alloy Steel, R_c 53-55

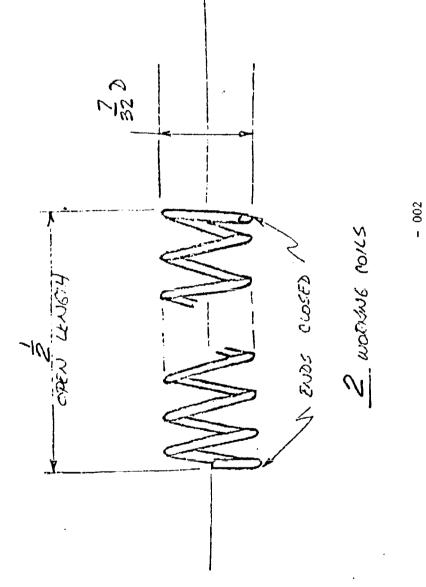
MKF No 20. Expeller, Mod II

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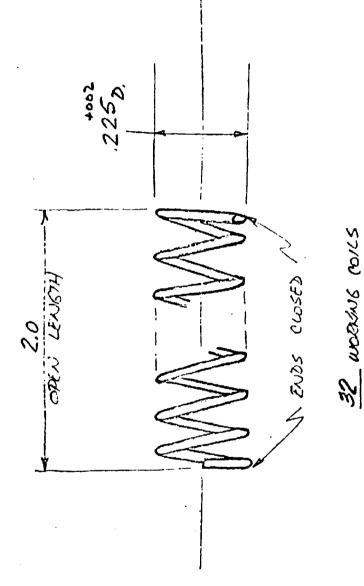
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MKF No 21. Spring, Pin, Bolt

MATERIAL: Music Wire, $.015_{\mathrm{D}}$

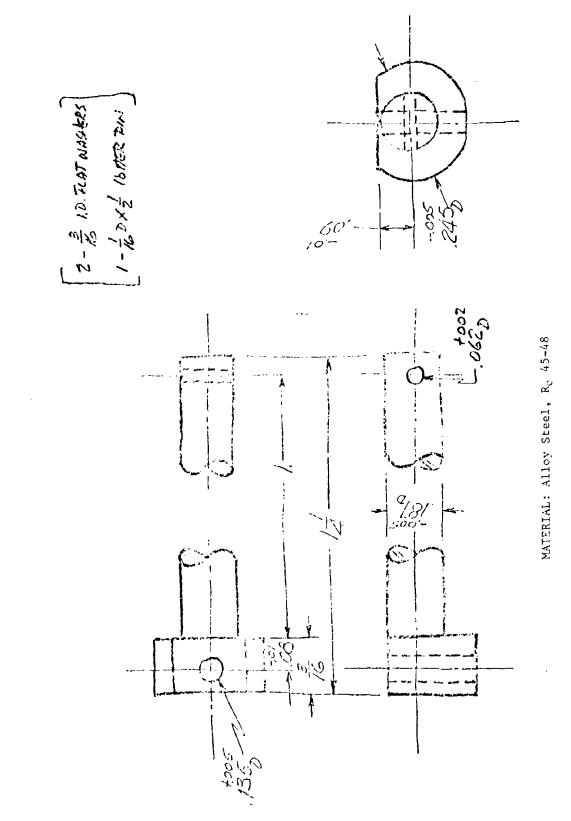


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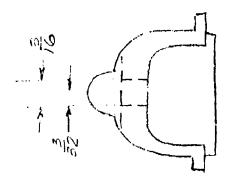
MATERIAL: Music Wire, .032_D

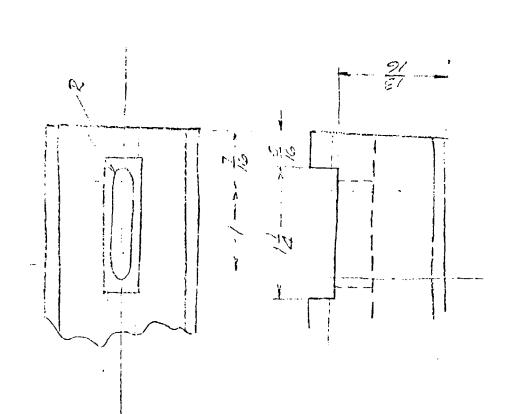
MKF No 22. Spring, Return, Firing Pin

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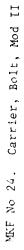
MKF No 23. Pin, Bolt Support, Mod II





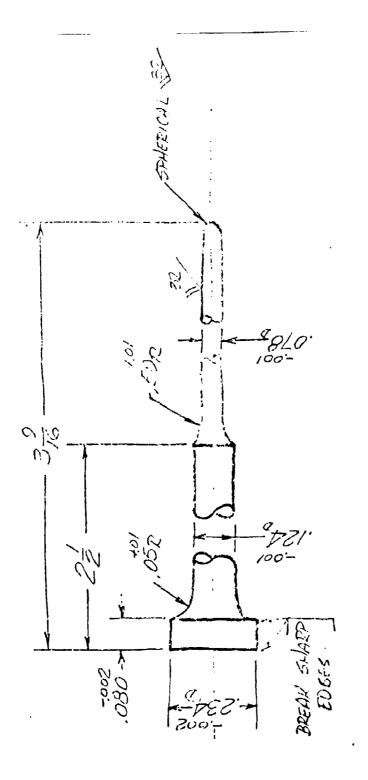
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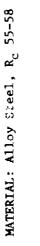
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MKF No 25. Firing Pin, Mod II

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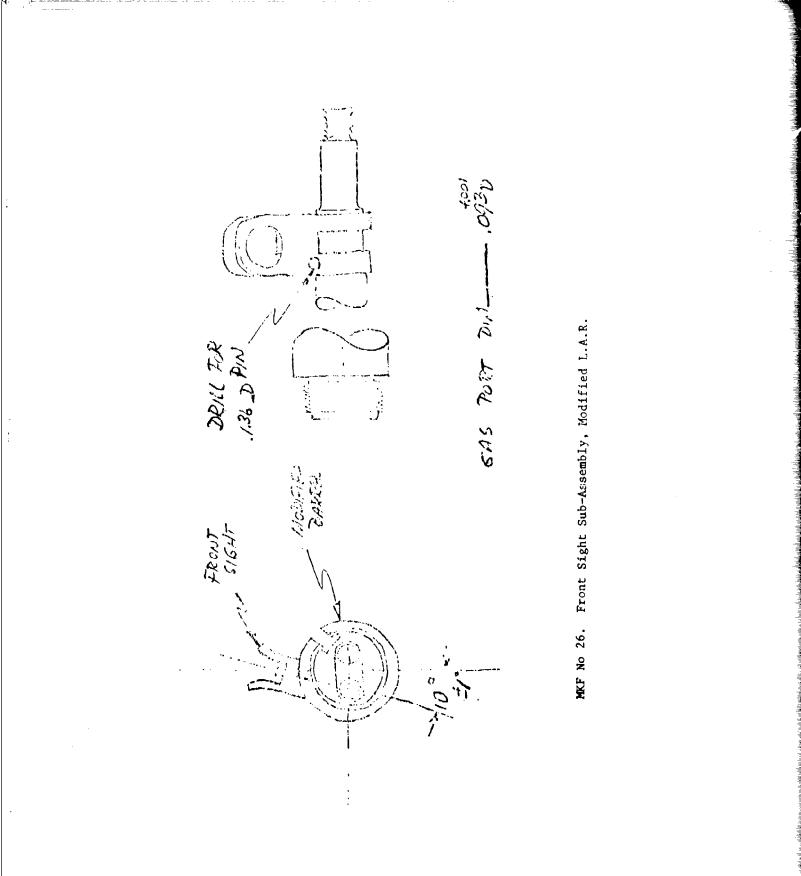
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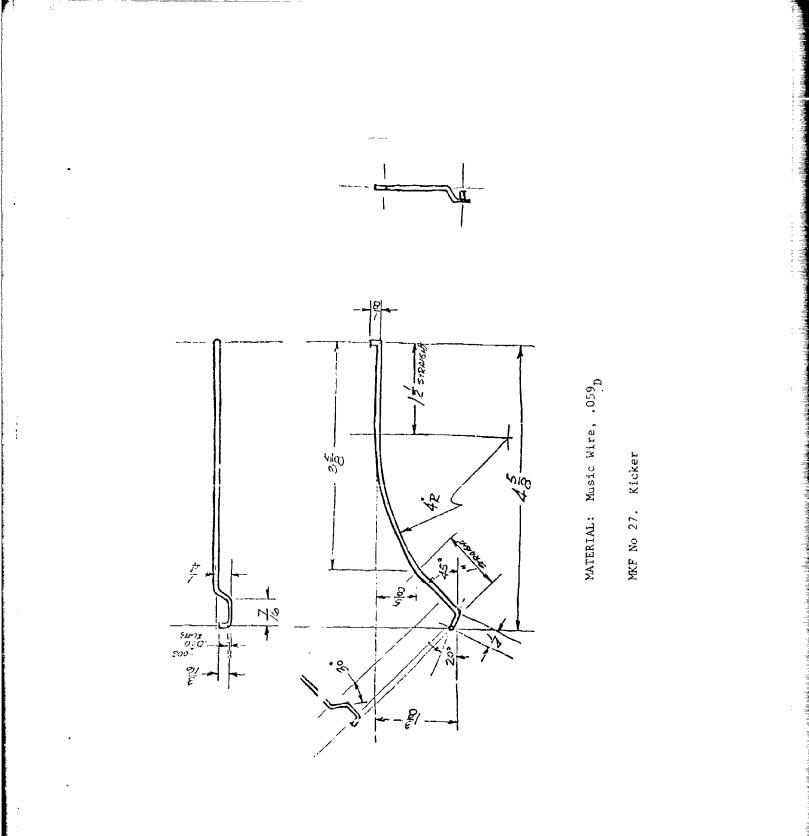
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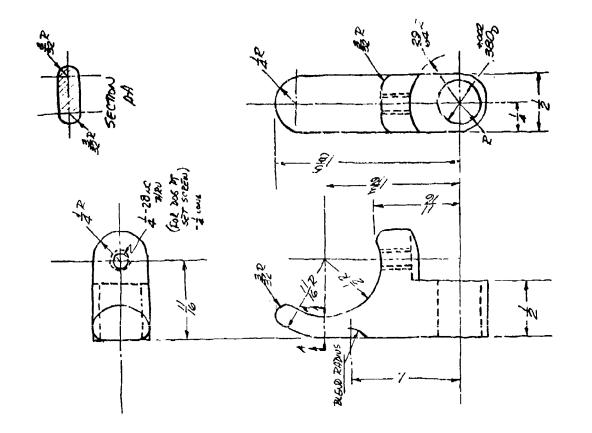
data taddala

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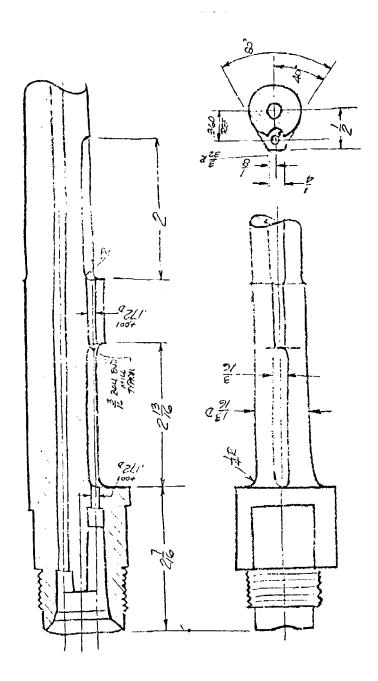




MATERIAL: Aluminum



MKF No 28. Handle, Mod II



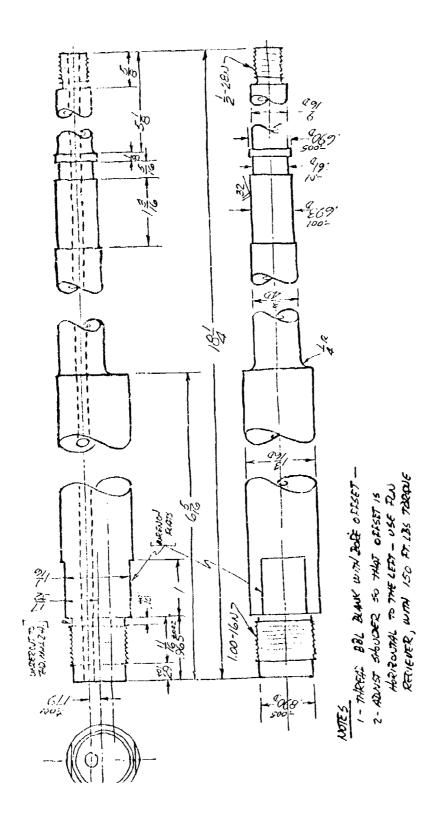
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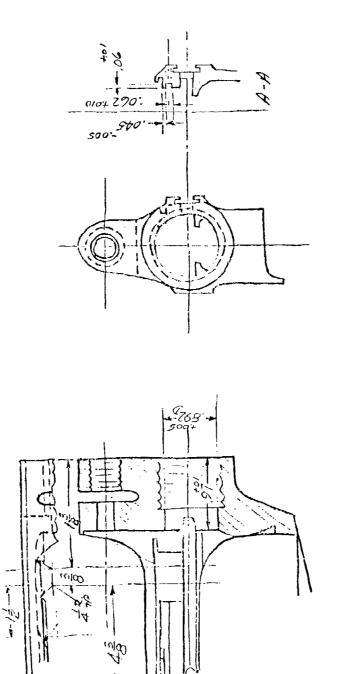




MKF No 30. Barrel, Mod II

5.56 mm - 1/9 Twist

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MKF No 31. Receiver, Kicker Slot Detail, Mod 11

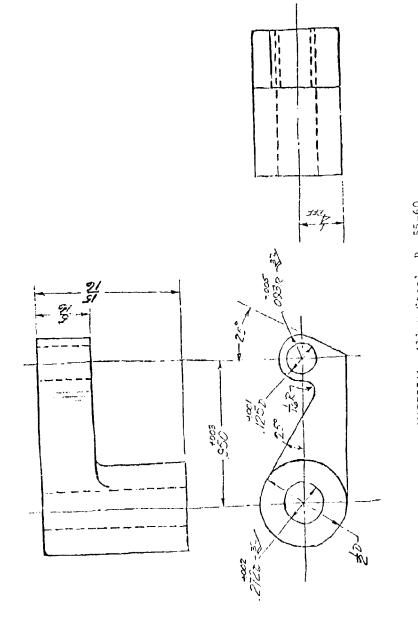
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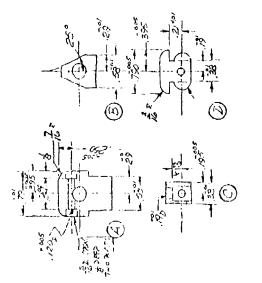
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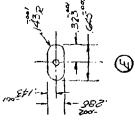
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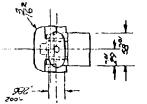
TYPE No. 32. Pawl, Mod II

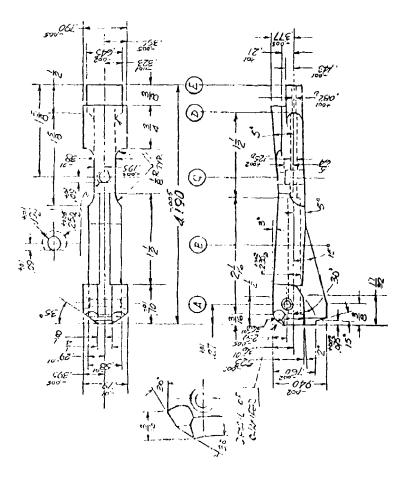
MATERIAL: Alloy Steel, R. 55-60



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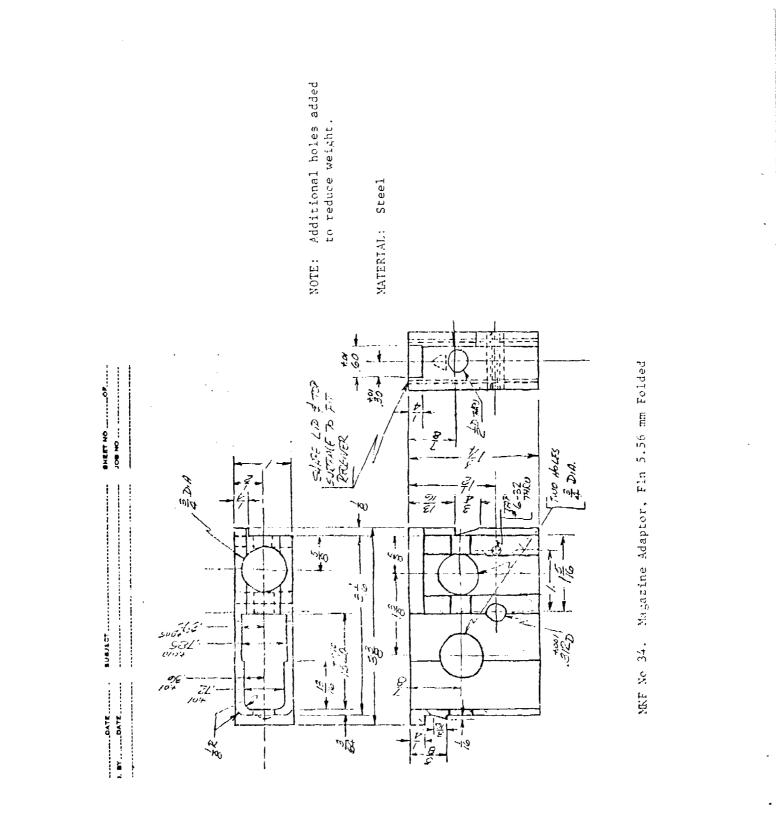


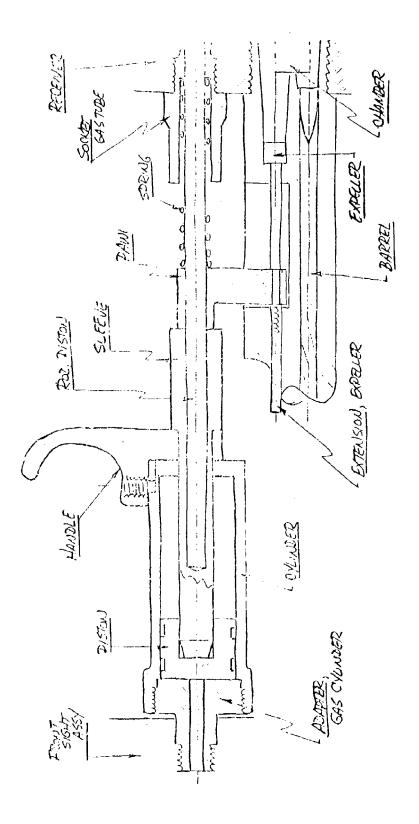




MKF No 33. Bolt, Mod II

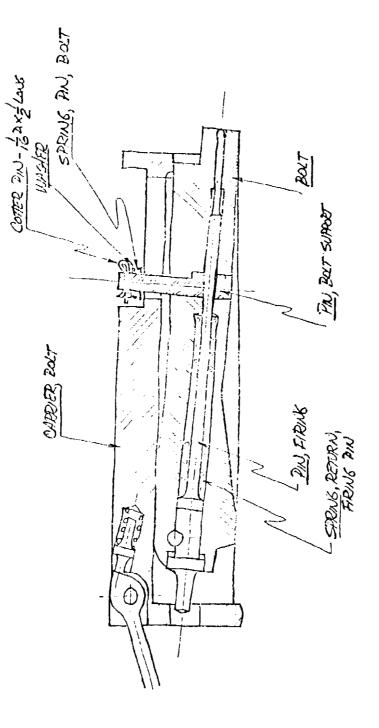
MATERIAL: Alloy Steel, R_c 58-60



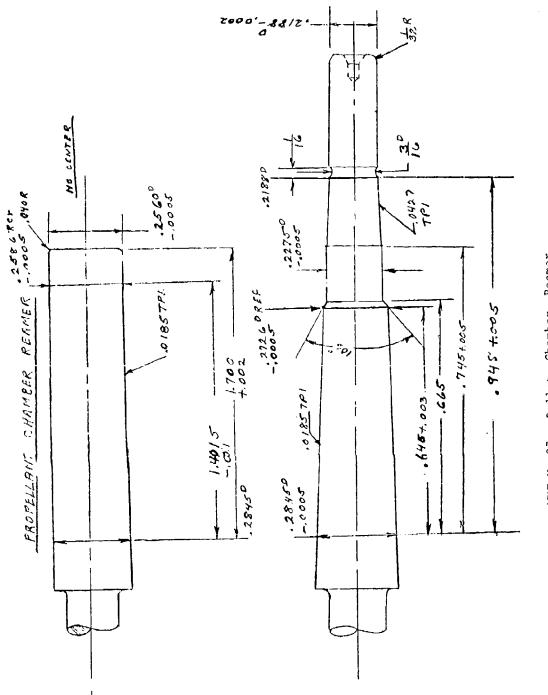


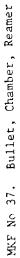
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MKF No 35. Sub-Assembly, Gas System











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MKF No 38. Fore Arm, Mod II

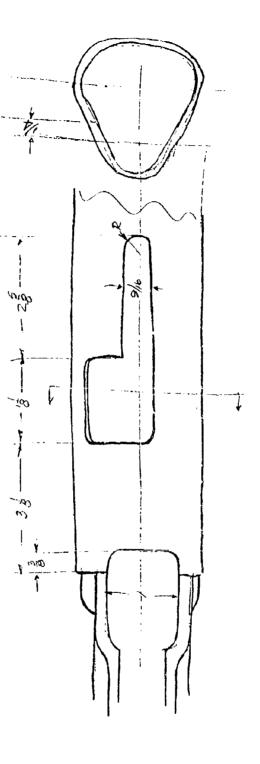
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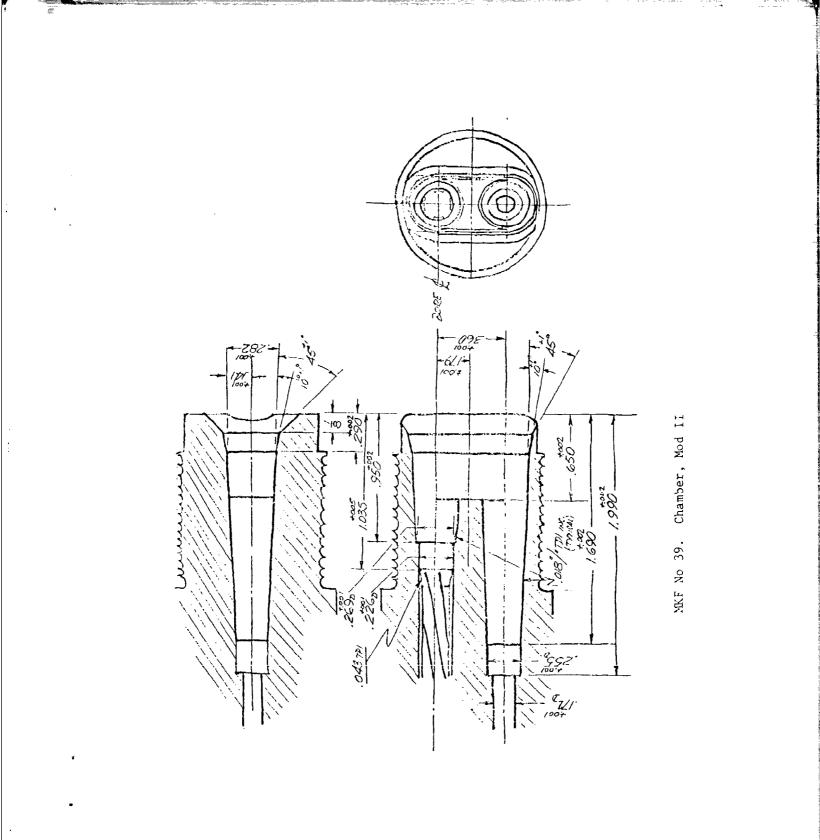
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